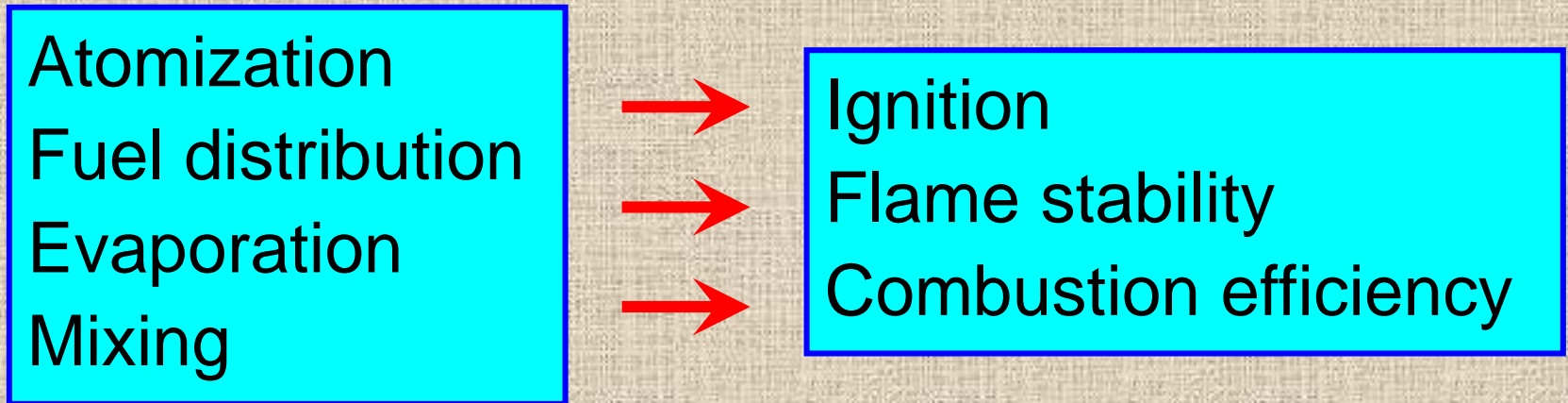


Atomization vs. Vaporization of Fuel in Micro Gas Turbines

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Impact of fuel delivery system



Difficulties in small-scale jet engine:

- ➔ Low ignition energy
- ➔ Small combustor volume
- ➔ Simplicity of the atomizer/vaporizer

Existing models for combustion of liquid fuel

- Semi-empirical droplet sizing models for a single atomizer
 - Evaporation models for droplets
- Need for integration of processes and models
- CFD involving two-phase flow and evaporation, combustion chemistry, heat transfer
- Heavy numerical calculations



Objectives

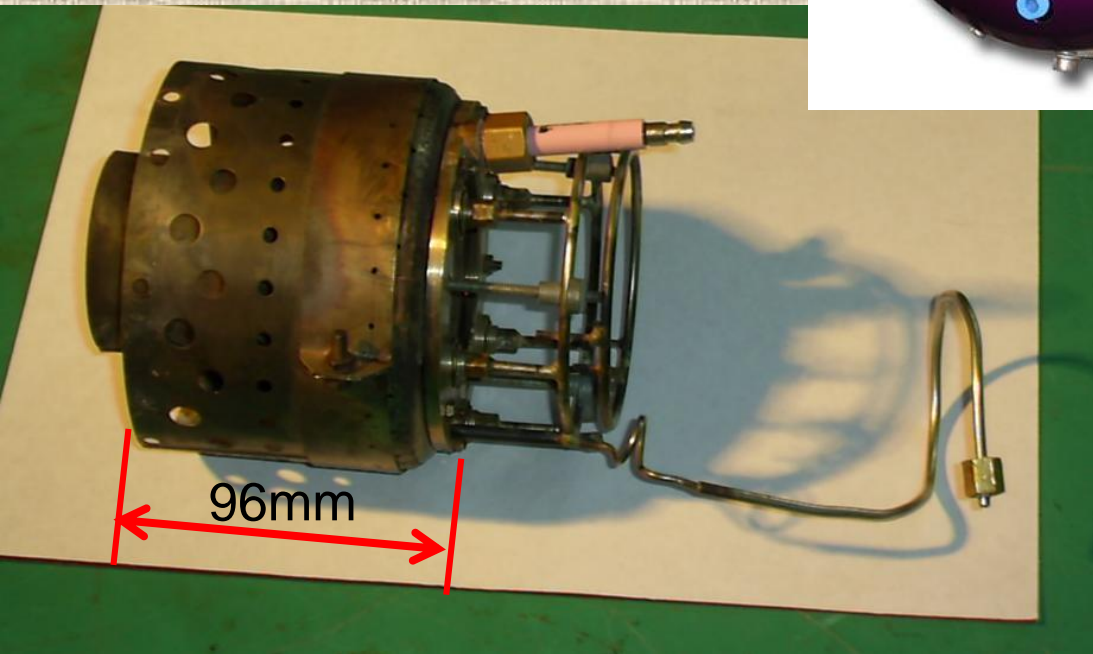
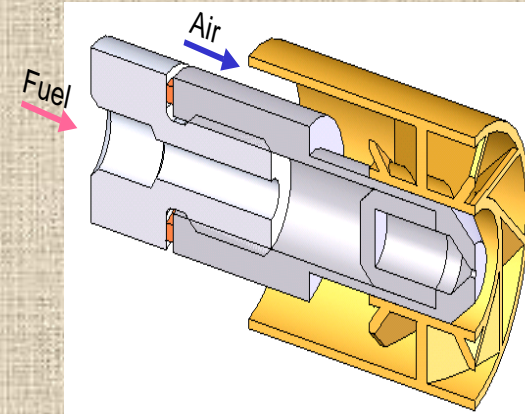
Comparison: Atomizers vs. Vaporizers for Micro Gas Turbines

- Ignition system
- Stable operational envelope
- Combustion Efficiencies



Research method

- Small-scale combustor
- Comparison of fuel supply only, not a combustor upgrade



Research method (cont.)

Measurements:

- Air & fuel flows
- Combustor outlet temperature at 4 points
- Combustion gas concentrations at 4 points

Evaluations:

- Stable operational envelope
- Combustion Efficiency
- Chemical efficiency



Research method (cont.)

Combustion efficiencies – definitions:

$$\text{Combustion Efficiency, } \eta(T) = \frac{\text{Enthalpy rise through combustor}}{\text{Chemical energy of liquid fuel}}$$

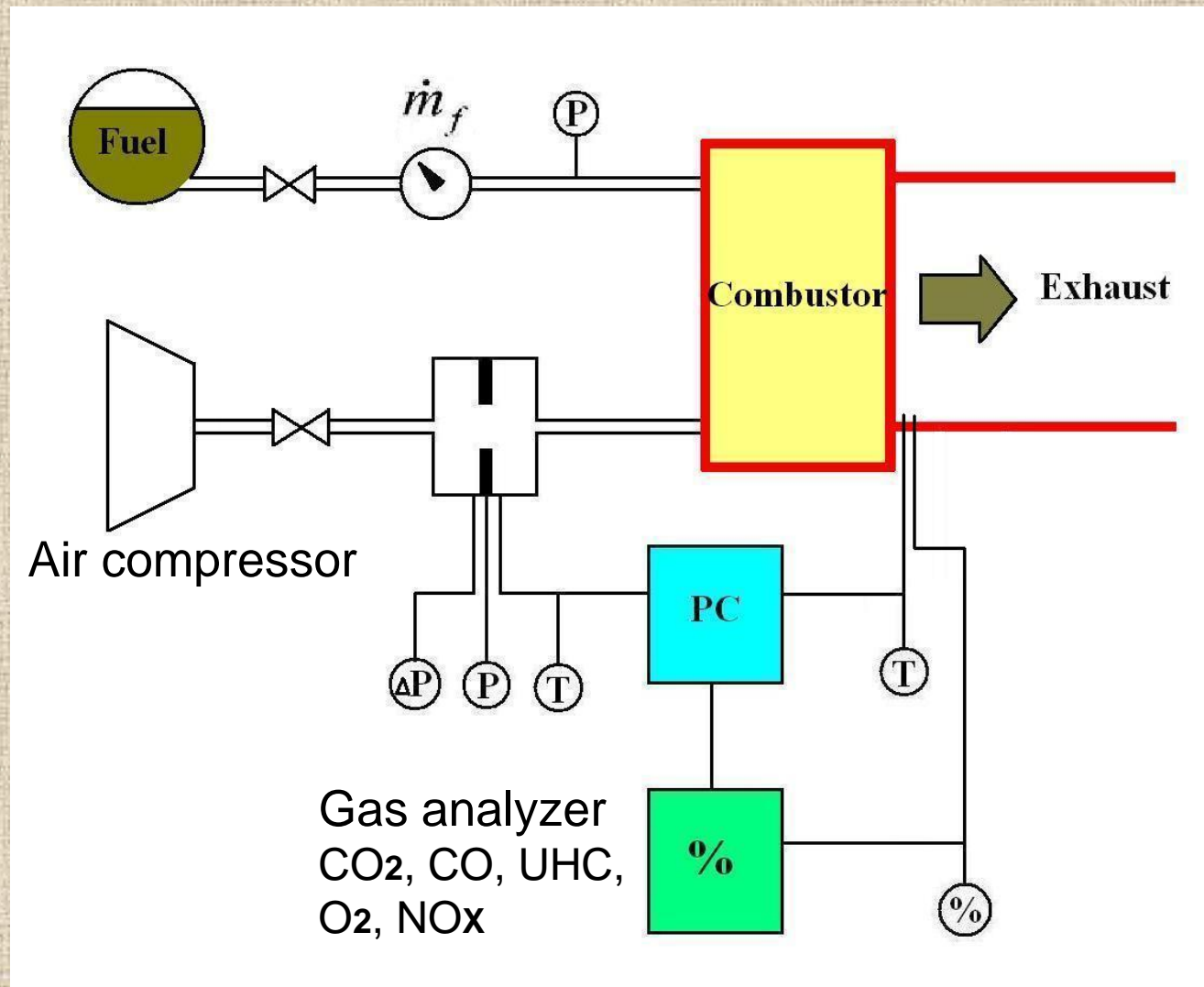
$$\text{Chemical efficiency, } \eta(\text{Ch}) = 1 - \frac{\text{Chemical energy of fuel remain in exhaust gases}}{\text{Chemical energy of vaporized liquid fuel}}$$

$$\eta_{b,T} = \frac{(1+f)T_{04}C_{Pg} - T_{03}C_{Pin}}{Q_R f}$$
$$\eta_{b,ch} = 1 - \frac{0.5[CO] + 1.3[CH_4]}{[CO_2] + [CO] + [CH_4]}$$

UHC → CH₄



Experiment setup



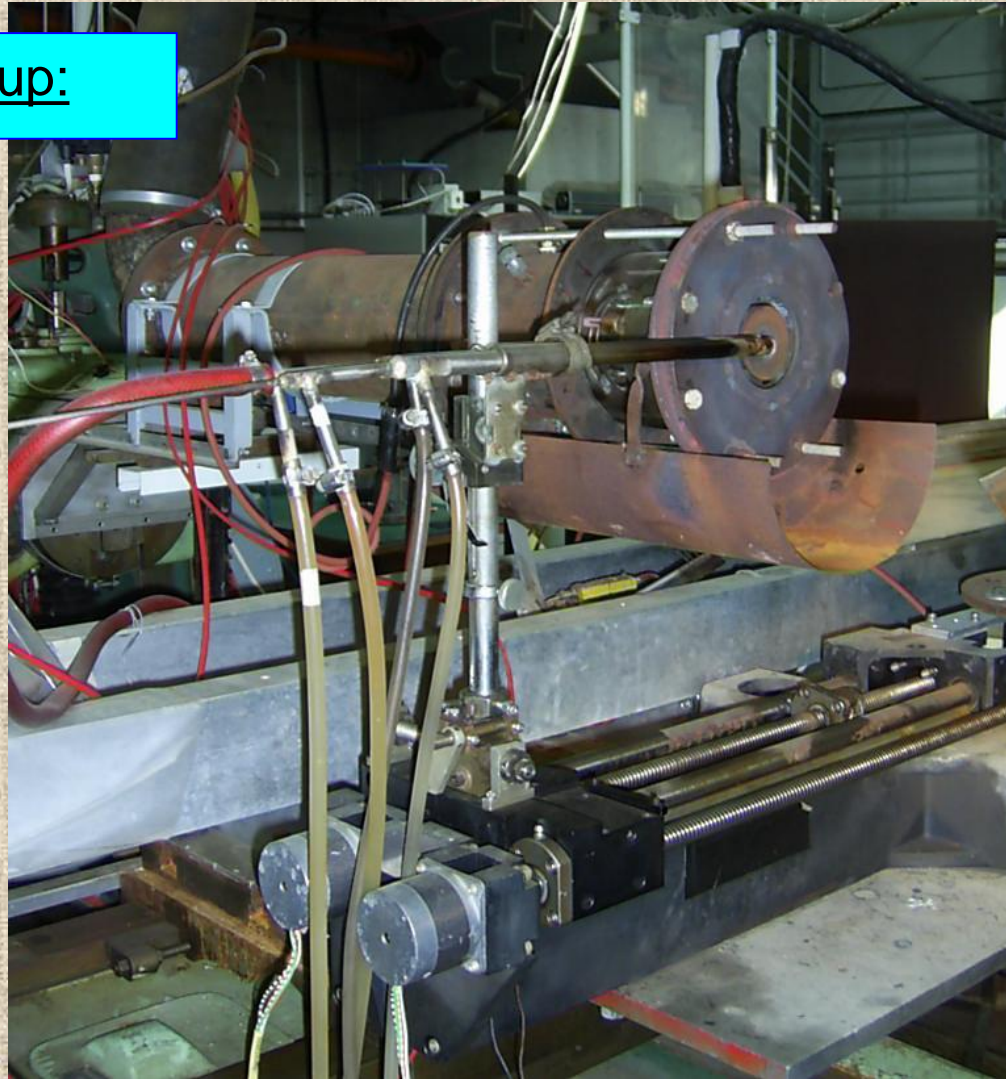
Experiment setup (cont.)

Gas analyzer:
CO₂, CO, UHC, NO_x, O₂

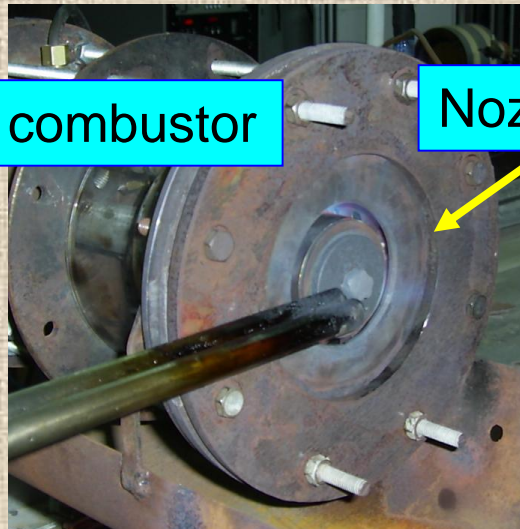


Experiment setup (cont.)

Combustor setup:

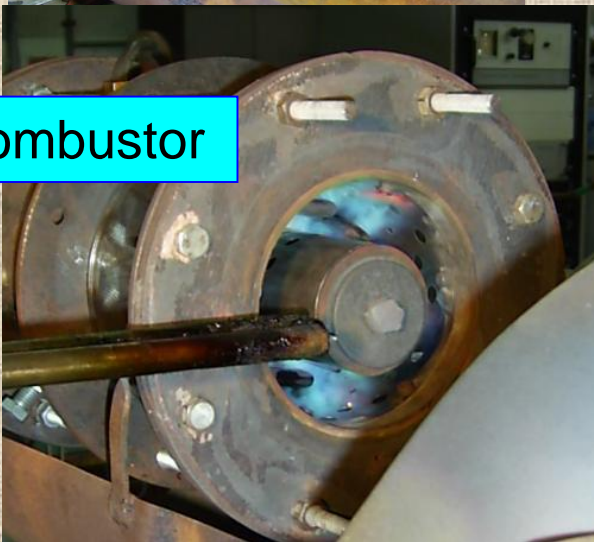
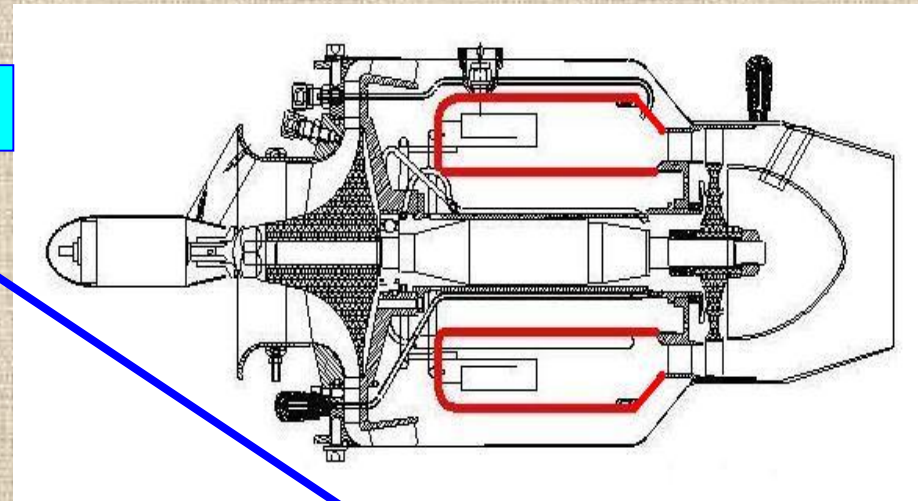


Experiment setup (cont.)

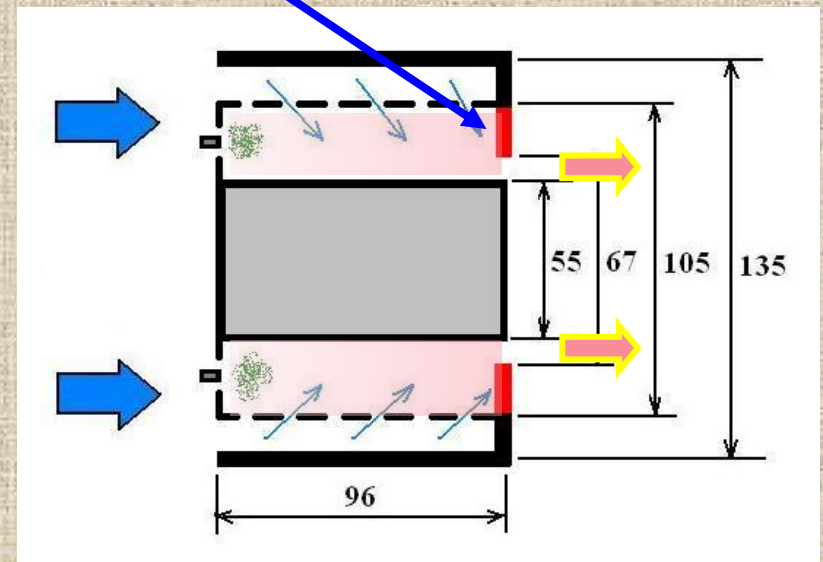


Blocked combustor

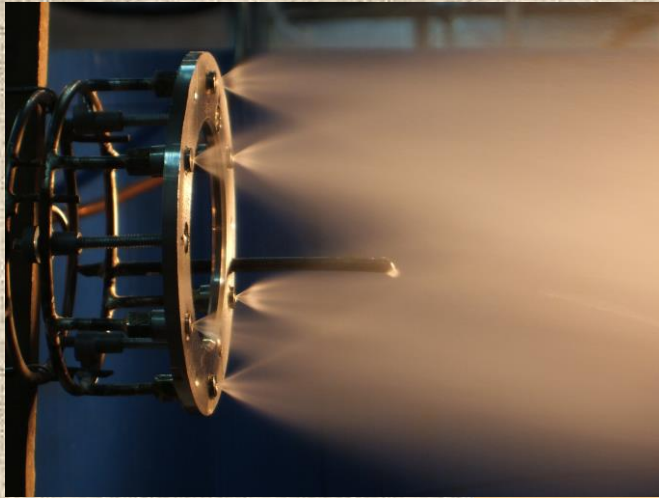
Nozzle ring



Open combustor



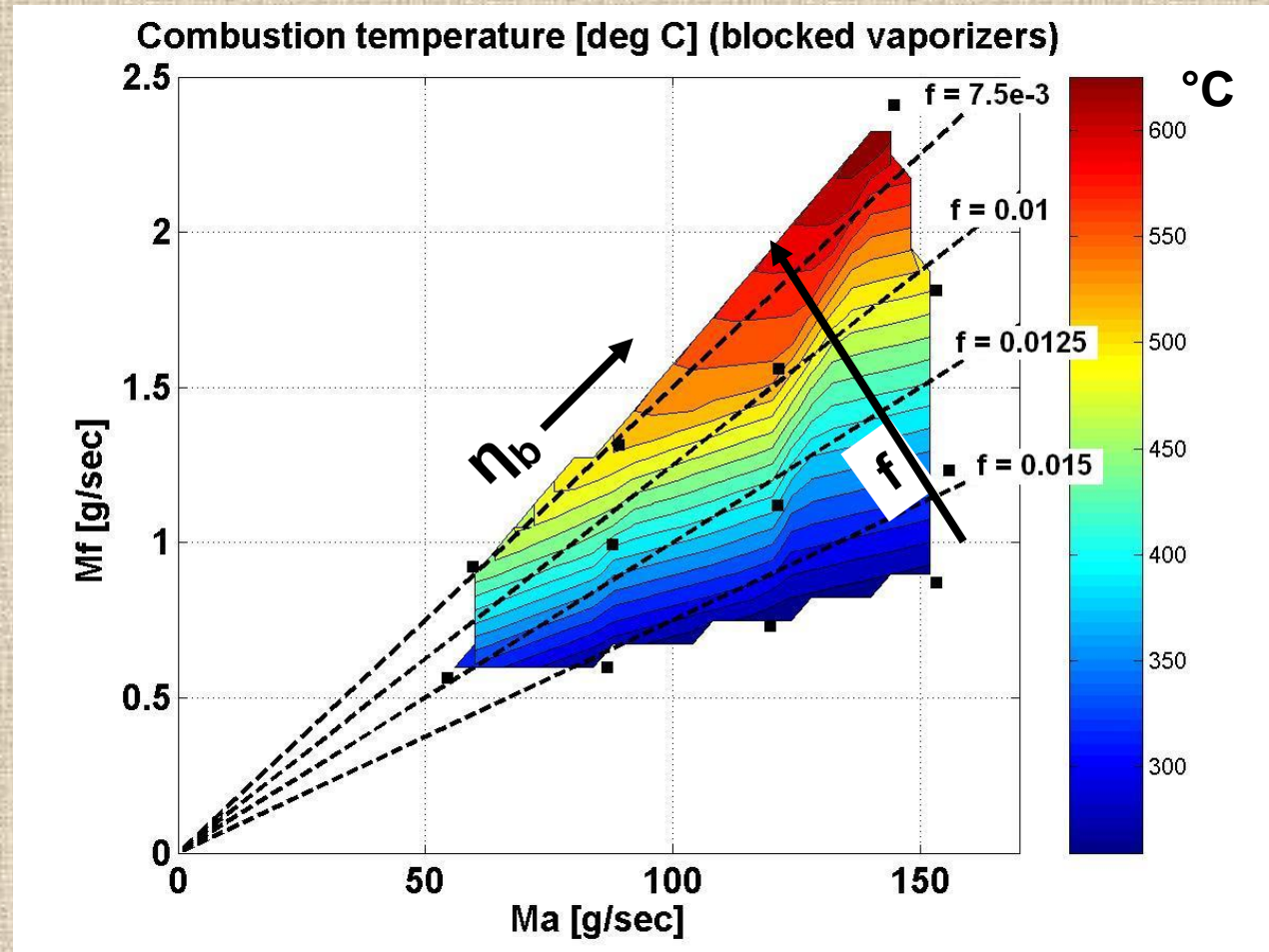
Experiment setup (cont.)



Results

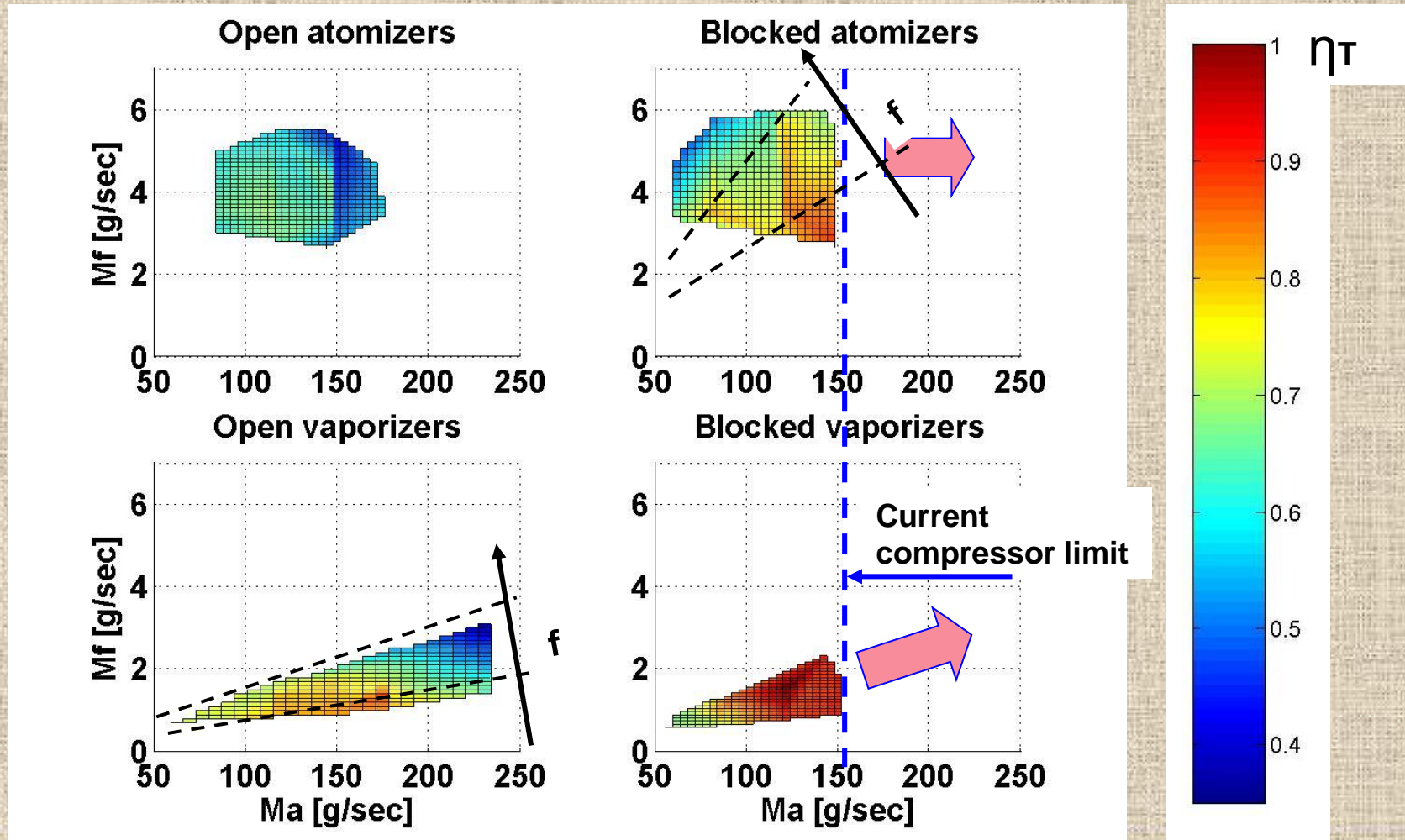
Exit temperature for different air and fuel mass flow rates

$$T_{04} = \frac{Q_R f + C_{P_{in}} T_{03}}{C_{P_g}(1 + f)}$$



Results (cont.):

combustion efficiencies η_T for different configurations



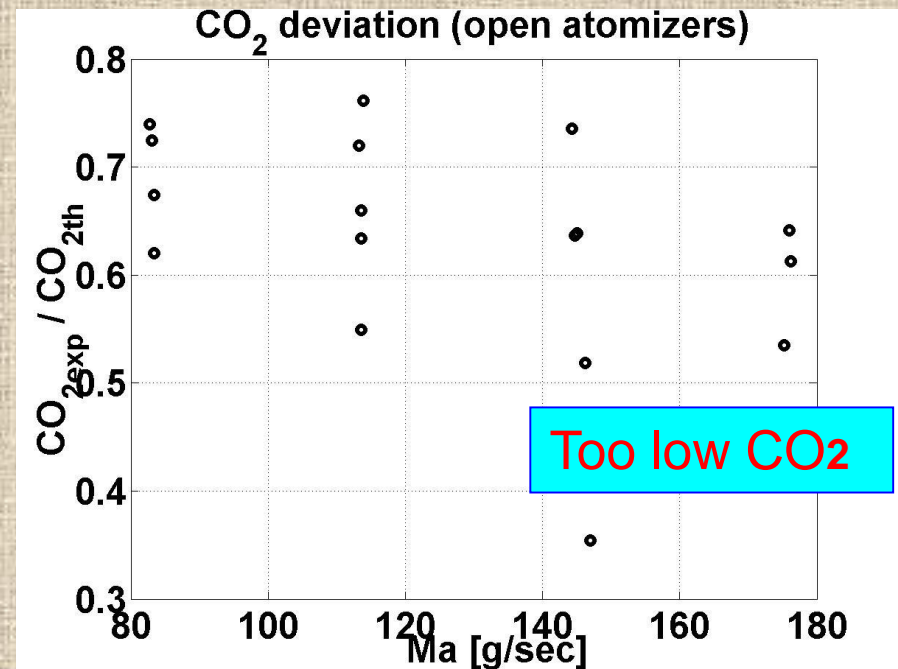
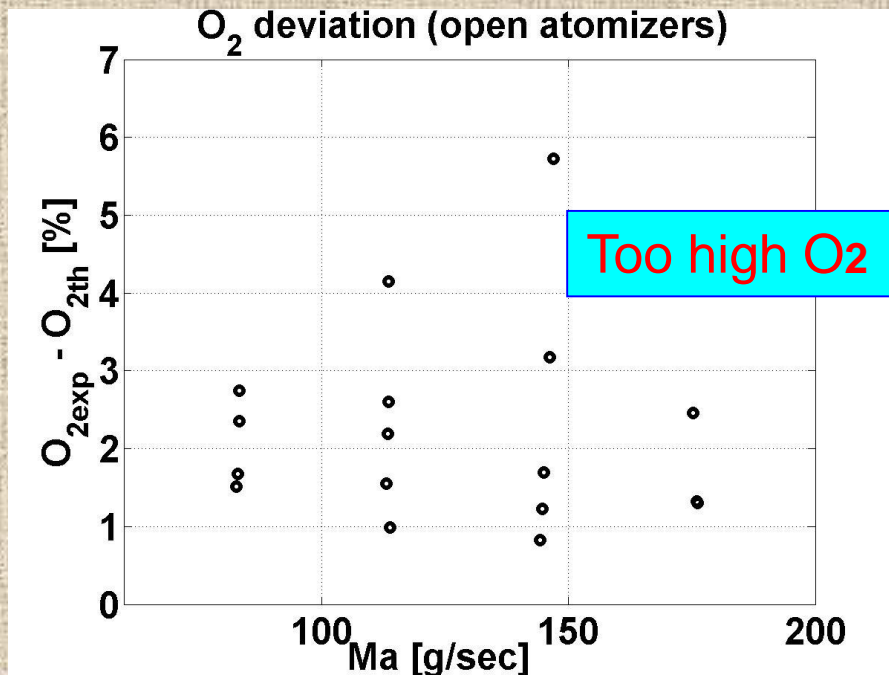
Results (cont.)

Low efficiency:

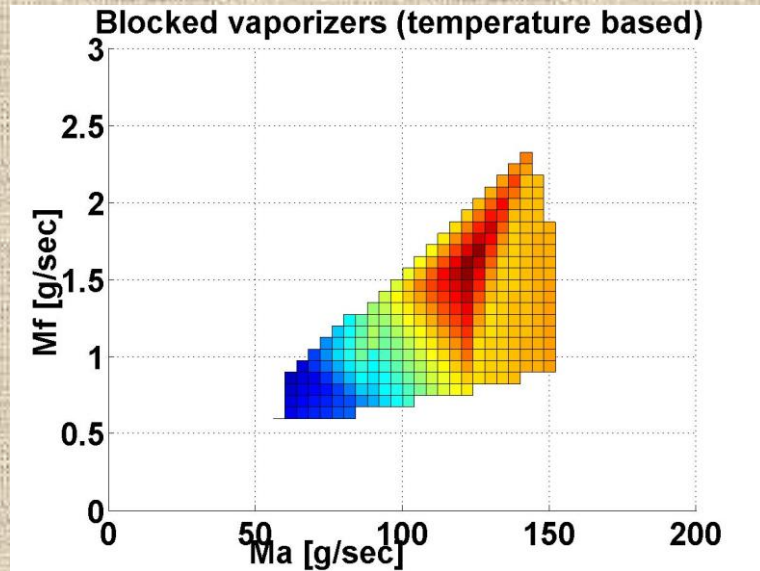
- low inlet temperature
- low air pressure
- ➔ Incomplete vaporization
- ➔ Insufficient residence time
- ➔ Insufficient mixing

$$\eta = F(t_R)$$

$$t_R = \frac{\rho V}{\dot{m}} \approx 3ms$$

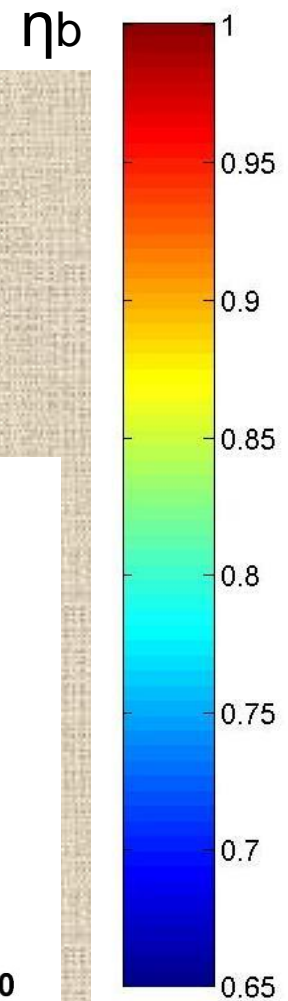
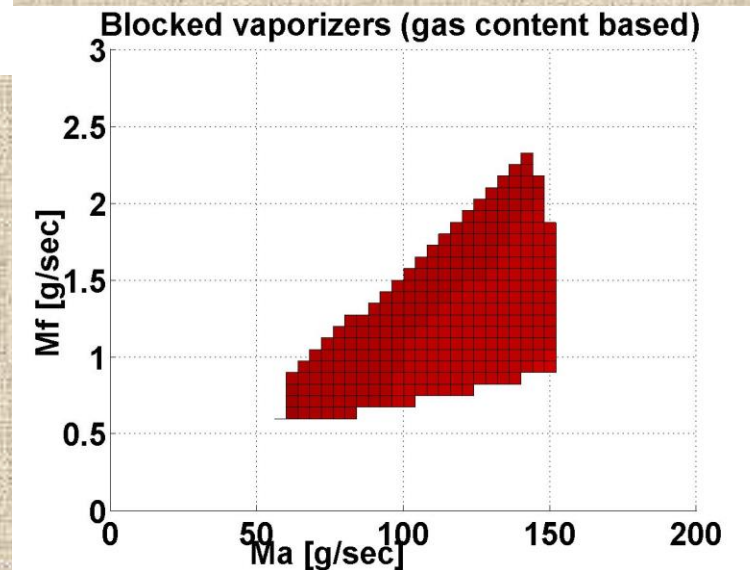


Results (cont.)

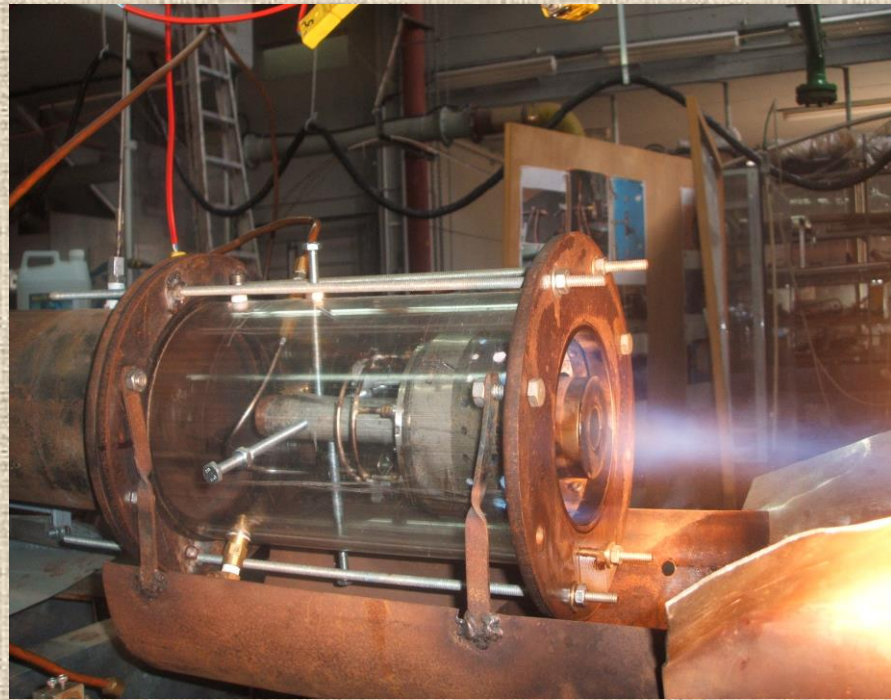


$$\eta_{b,T} < \eta_{b,ch}$$

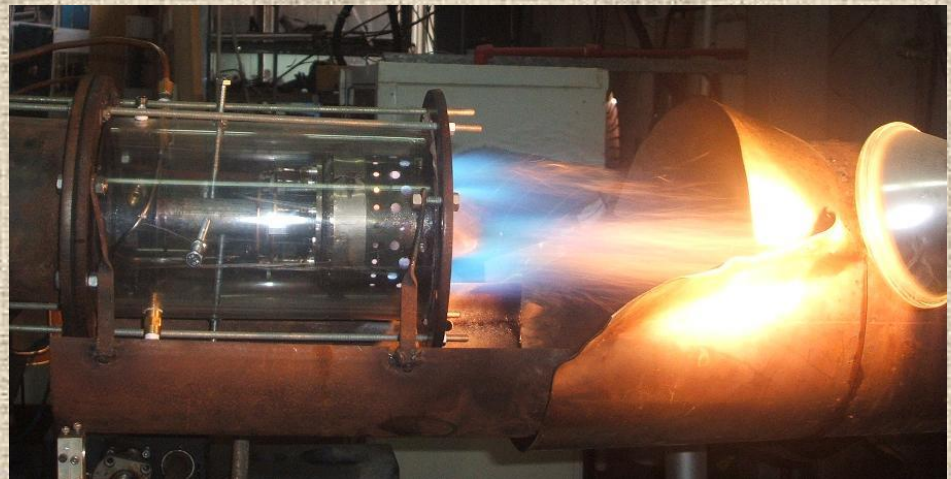
Differences mainly due to
unvaporized liquid fuel



Low equivalence ratio:
almost complete
combustion



High equivalence ratio:
incomplete combustion



Conclusions

Atomizers

- ✓ Wider stable operational range
(wider fuel supply ranges: 3 – 6g/sec vs. 0.5 – 2g/sec)
- ✓ Higher fuel flow rates & temperatures at the max efficiency points
(4g/sec vs. 1g/sec & 1300K vs. 800K)
- ✓ Simple ignition (electric plug)
- ✓ Faster transient response
- X Larger combustor volume required (for evaporation & mixing)
- X Higher fuel pressure requirement

Vaporizers

- ✓ Higher efficiency due to longer vaporized fuel flow path
($\eta=0.9$ vs. 0.75) (higher combustion pressure would give higher efficiencies)
- X Complicated ignition (additional gas ignition system required)
- X Slower transient response (thermal inertia & fuel evaporation time)



Conclusions (Cont.)

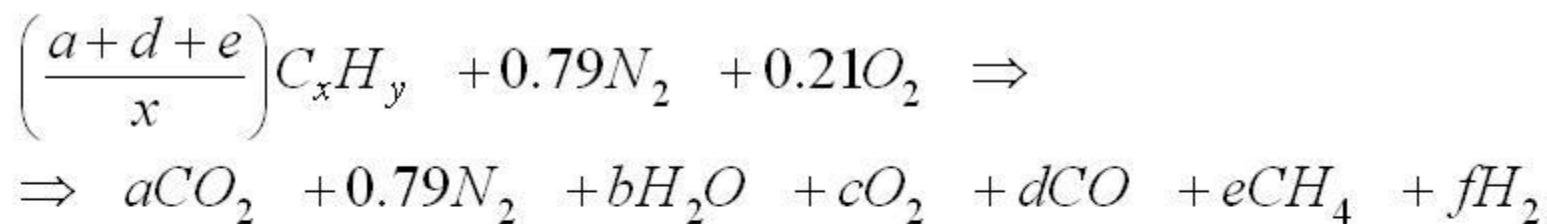
General

- Combustion efficiency increases with pressure
- Unvaporized fuel affect efficiency values, hence should be accounted for ($\eta(\text{Ch})$ vs. $\eta(\text{T})$)
- Direct retrofitting of atomizers instead of vaporizes widen operational envelope but reduces combustion efficiencies
- New designs that can account for the larger volume required by atomizer may benefit from its advantages



The background of the slide is a close-up, high-resolution image of flames. The fire is intense, with a color palette ranging from deep reds and oranges to bright yellows and whites at the tips of the tongues of fire. The flames are dynamic and swirling, creating a sense of heat and movement.

Questions?



$$\eta_b = 1 - \frac{dQ_{R,CO} + eQ_{R,CH_4} + fQ_{R,H_2}}{\left(\frac{a+d+e}{x}\right)Q_R}$$

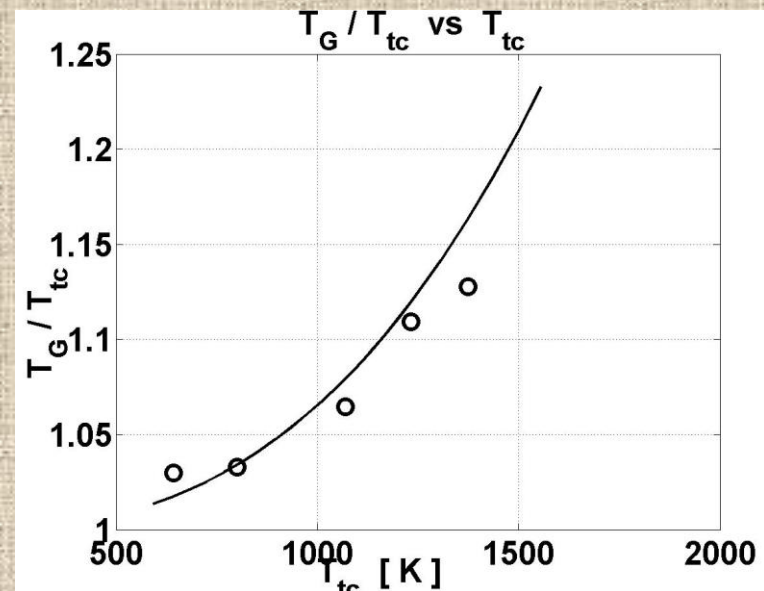


Experiment setup (cont.)

Compensation of the radiative heat losses from TC:

$$T_g = T_{TC} + \frac{\varepsilon \sigma F (2T_{TC}^4 - T_\infty^4 - T_t^4)}{h}$$
$$h = \frac{k_g}{D} Nu_D \quad V = \sqrt{2C_{Pg} T_{app} (1 - P_a / P_C)^{(\gamma-1)/\gamma}}$$

The model was validated by comparison of shielded and unshielded thermocouples



Results (cont.)

Low efficiency: high UHC values

