

Design for Additive Manufacturing: TOffee

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Imperial College of London
UQLab
UK



Smith *institute*
for industrial mathematics and system engineering

UQLab

People: 5 Post Docs, 5 PhD students, 1 Academic

Sponsors-Collaborations: Rolls-Royce (UK), General Electric (UK-Italy), Criepti (Japan), Airbus (UK-Fr-DE), EPSRC, NASA Langley (US), etc

Major Areas: Uncertainty Quantification and Additive Manufacturing

Prizes of the Lab: Lloyd's Prize runner up for Science of Risk, John Frances Prize (best Imperial PhD student), Elaine Austin Centenary Memorial Prize, UK Parliament invitation (STEM for Britain), Reynolds prize poster finalist etc

Spinouts: **MonolithAI**



TOffee

TOffee



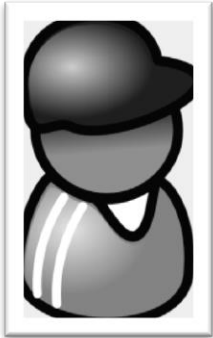
Rolls-Royce®



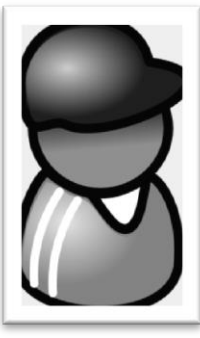
AIRBUS
AN EADS COMPANY



UQLab



Dr V Badalassi



N Pepper



A Cassinelli



A Gaymann



N Casari



H Gauch



Dr M. Pietropaoli



Dr Hui Xu



Dr G Castiglioni



Dr R. Ahlfeld



Prof F Montomoli



Arianna

Recent Prizes

- **Audrey:** Amelia Earhart Fellowship, worldwide prize, one of the best 32 females worldwide in aviation
- **Marco:** EPSRC Doctoral Prize, STEM for Britain selected at UK Parliament as one of best UK researches, Take AIM second place, CDT Prize
- **Richard:** EPSRC Fellowship, RAEng fellowship, Francis Prize as best PhD student of Imperial College
- **MonolithAI** named one of the best 7 Deep Science Startups in the World for industry 4.0
- **TOffee:** Amazon AWS programmable 2018 winner

Programm/able



Research Areas



MONOLITH

Uncertainty Quantification

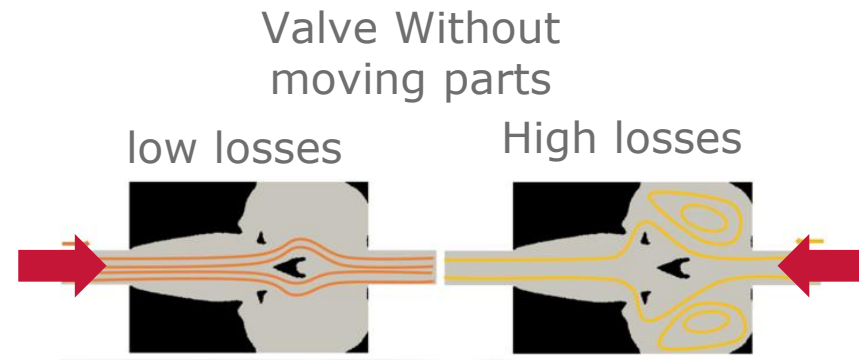
Data Analytics AI

TOffee

***Design for AM Under Uncertainty
(Robust Optimization)***

Toffee optimizes Under Uncertainty

- Toffee is an in house optimization code, fluid-structure:
- Conjugate Heat Transfer and Heat Exchangers
- Bi-directional flows (valves without moving parts)
- Low pressure losses
- Robustness against variations
- Applied to real cases
- And much more



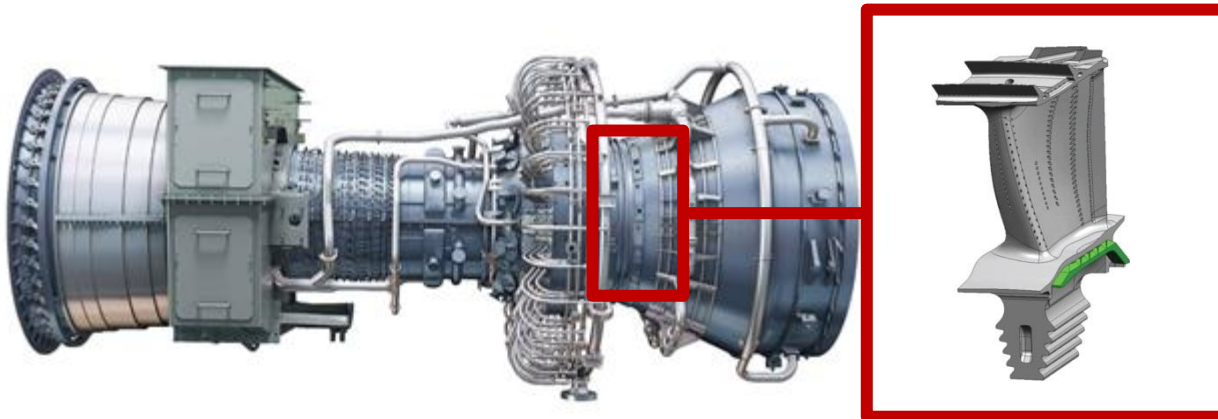
Gas Turbine Cooling: our vision

Increase of efficiency and reliability of gas turbines

- *Higher TET ~2200K in the last generation engines*
- *Variation of ~30K can reduce by half the life of the engine*



More complex and efficient coolant systems



Bio-Inspired coolant design

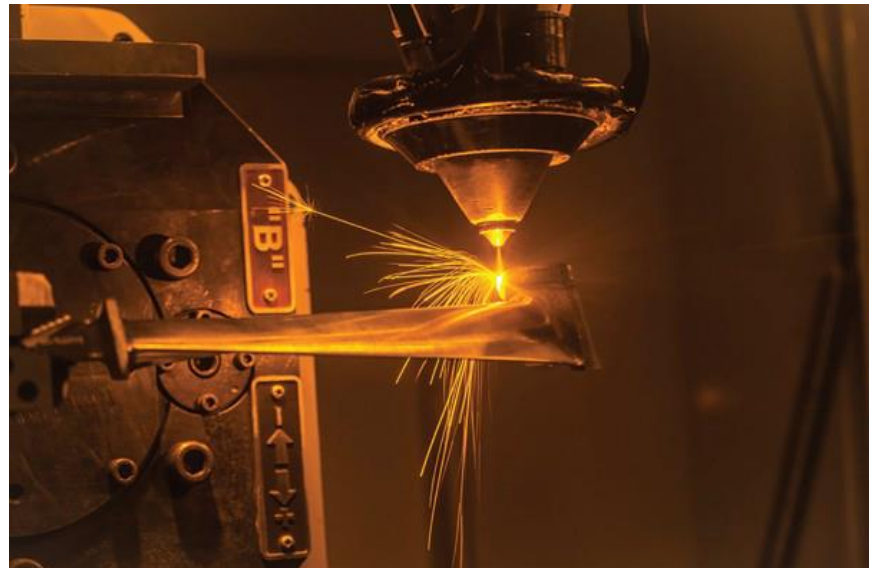
The design process must take into account several aspects

- *Pressure drop of the coolant flow*
- *Temperature of the mechanical parts*
- *Temperature gradients across the whole blade*
- *Reliability against mechanical stress*
- ***Manufacturing constraints***

.. . and it must be automatic!!!

How to build it: Additive Manufacturing (AM)

- *Production of complex mechanical parts, avoiding standard manufacturing operation (drilling, milling...)*
- *Today is used in the **wrong way**: same part design.....*
- *It is a common problem when you have a new manufacturing technology*

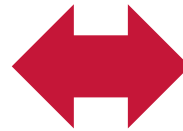


New Manufacturing... usually same design

Same design, different manufacturing process



Wooden ship



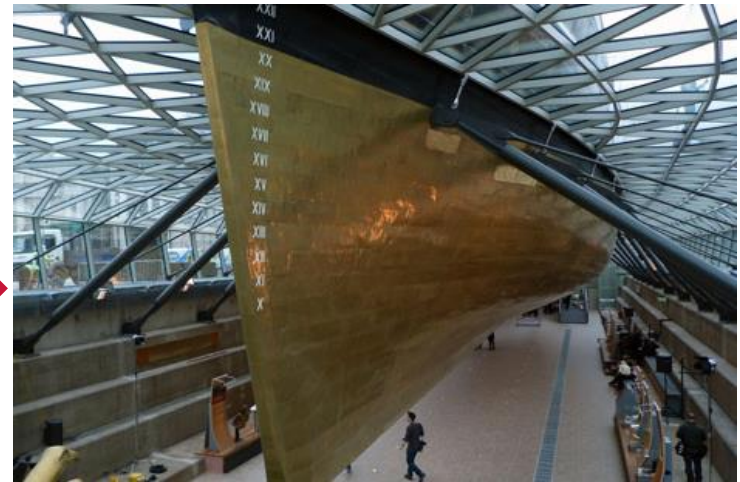
Metal hull, Cutty Sark, London

New Manufacturing... usually same design

Same design, different manufacturing process



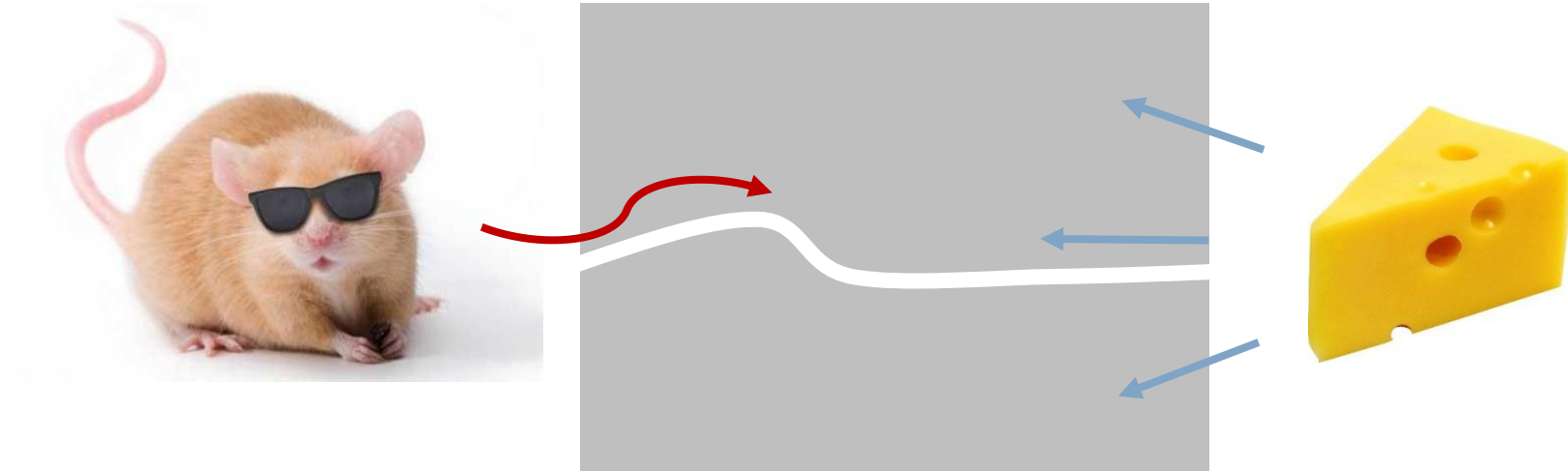
Wooden ship



Metal hull, Cutty Sark, London



Adjoint Algorithm



p
 v
 T

Primal Variables

q
 u
 τ

Adjoint Variables

Theoretical Model

Lagrangian optimisation approach

$$L = F - \int_{\Omega} \xi_i R_i d\Omega$$

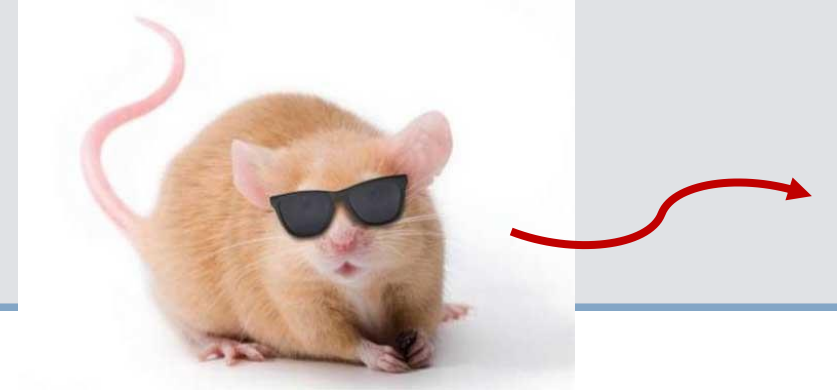
F Objective Function

R_i Constraints – Fluid governing equations for incompressible flow

ξ_i Lagrangian multipliers – Adjoint variables

The domain is a porous medium with variable impermeability α

Theoretical Model



Lagrangian optimisation approach

$$L = F - \int_{\Omega} \xi_i R_i d\Omega$$

Continuity

$$\nabla \cdot v = 0$$

Momentum

$$v \cdot \nabla v = -\frac{\nabla p}{\rho} + \nabla \cdot (\nu \nabla v) - \alpha v$$

Energy

$$v \cdot \nabla T = \frac{1}{\rho c} \nabla \cdot (k \nabla T)$$

The solution must verify $\delta_{\alpha} L = 0$

Adjoint Optimisation



After a long computation, the lagrangian variation $\delta_{\alpha} L$ is found

- *A set of adjoint equations and adjoint boundary conditions is derived to evaluate the adjoint variables*

$$\nabla \cdot u = 0$$

$$-v \cdot (\nabla u + \nabla^t u) = -\frac{\nabla q}{\rho} + \nabla \cdot (\nu \nabla u) - \alpha u - c\tau \nabla T$$

$$-v \cdot \nabla \tau = \frac{1}{\rho c} \nabla \cdot (k \nabla \tau)$$

Objective Functions

Stagnation pressure dissipation and heat transfer must be optimised

$$F = \omega_1 f_1 + \omega_2 f_2$$

Pressure drop to be minimised

$$f_1 = \int_{\Sigma} \left(p + \frac{1}{2} \rho |v|^2 \right) v_n \, d\Sigma$$

Temperature gain to be maximised

$$f_2 = \int_{\Sigma} \rho c \, T v_n \, d\Sigma$$

Results - U-Bend case

Test case for pressure drop optimisation

- *Comparison are made with the standard case*



TO Domain

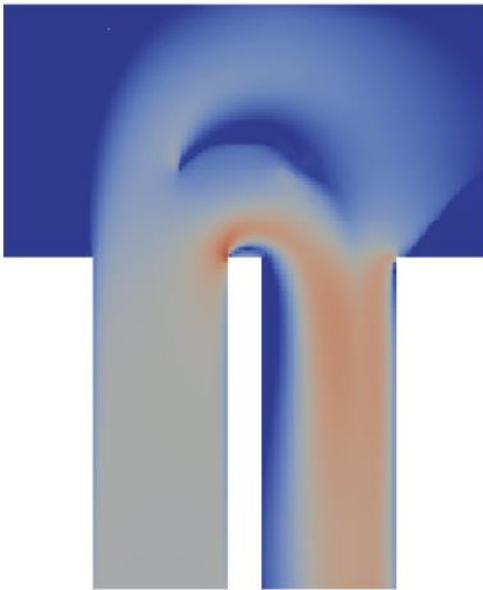


Standard case

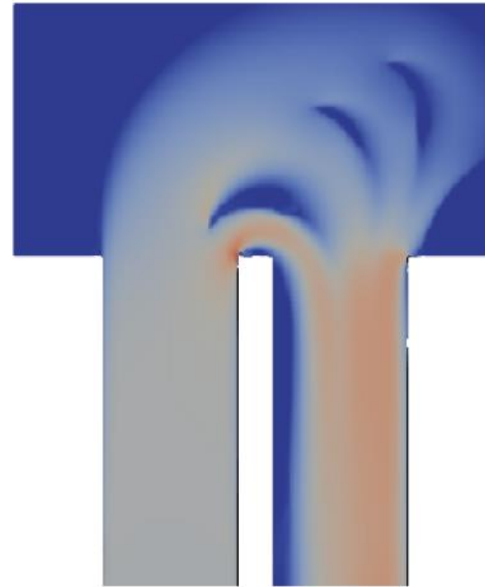
Results - Different Inlet Velocity

Inlet velocity :

6 m/s



17.5 m/s



<i>Inlet Velocity</i>	<i>Pressure Drop</i>	<i>Improvement</i>
<i>6 m/s</i>	<i>~ 47%</i>	<i>~ 50%</i>
<i>17.5 m/s</i>	<i>~ 39%</i>	<i>~ 54%</i>

Results - Different Inlet Velocity

Inlet velocity :

6 m/s



17.5 m/s

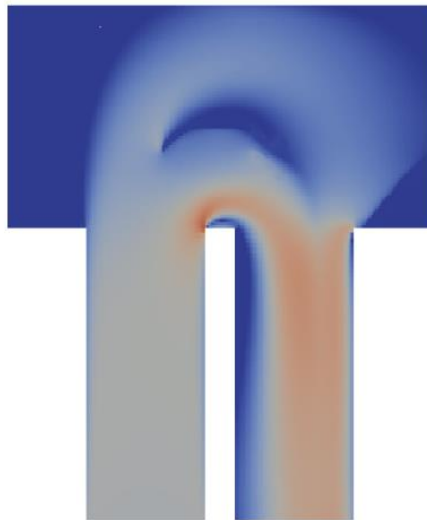


Filtered Geometry: the black region indicates the fluid region, i.e. the portion where the impermeability is low

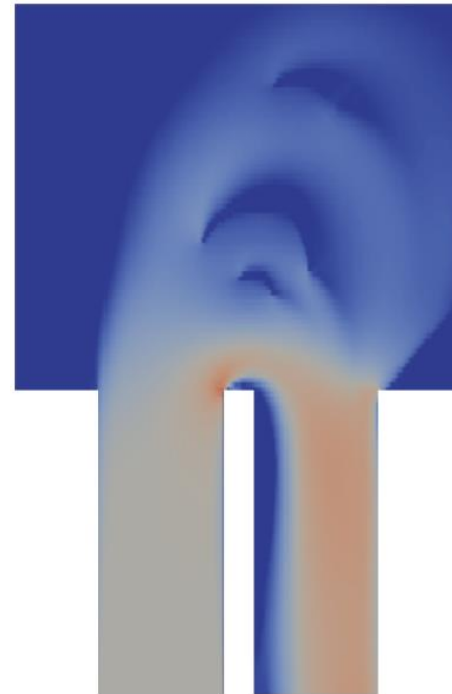
Results - Different Aspect Ratio

Aspect ratio (inlet vel. 17.5 m/s):

2:1



2:2



<i>Aspect ratio</i>	<i>Pressure Drop</i>	<i>Improvement</i>
2:1	~ 39%	~ 54%
2:2	~ 33%	~ 60%

Results - Different Aspect Ratio

Aspect ratio (inlet vel. 17.5 m/s):

2:1



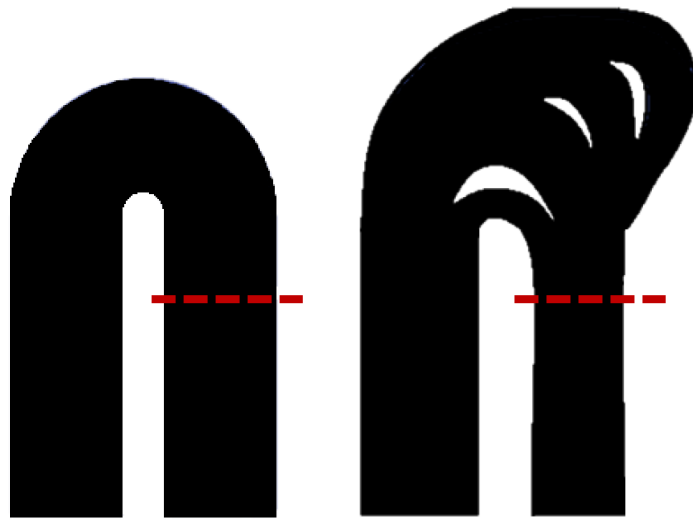
2:2



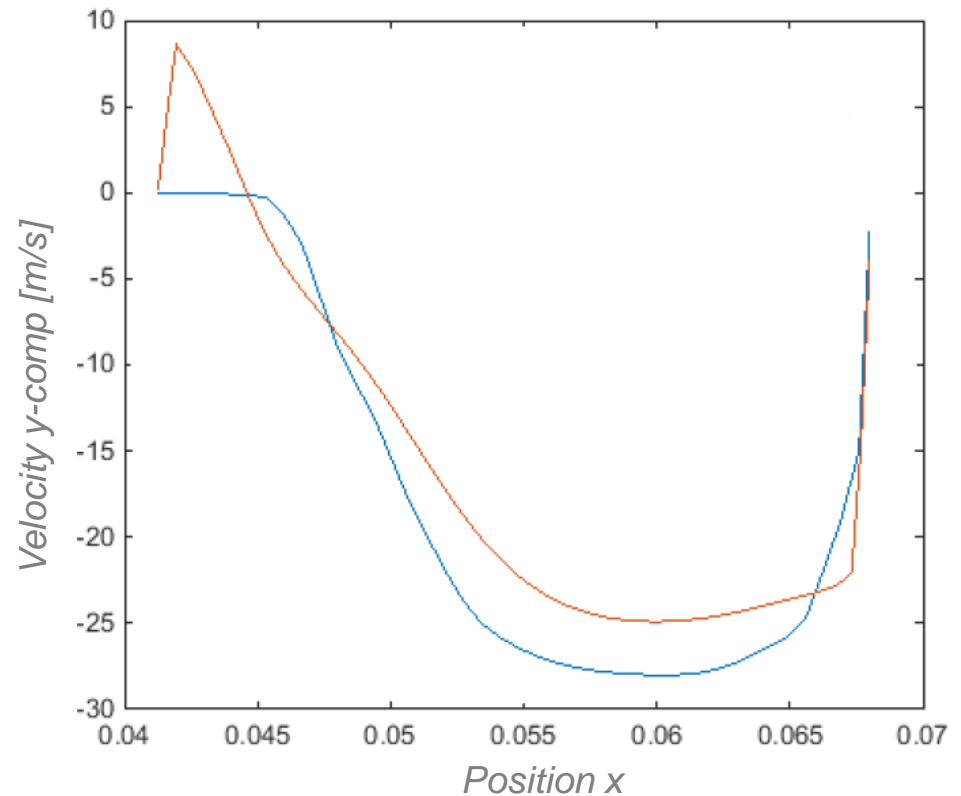
Filtered Geometry: the black region indicates the fluid region, i.e. the portion where the impermeability is low

Results – Velocity Profile

Velocity profile across the cutting red line for inlet velocity 17.5 m/s

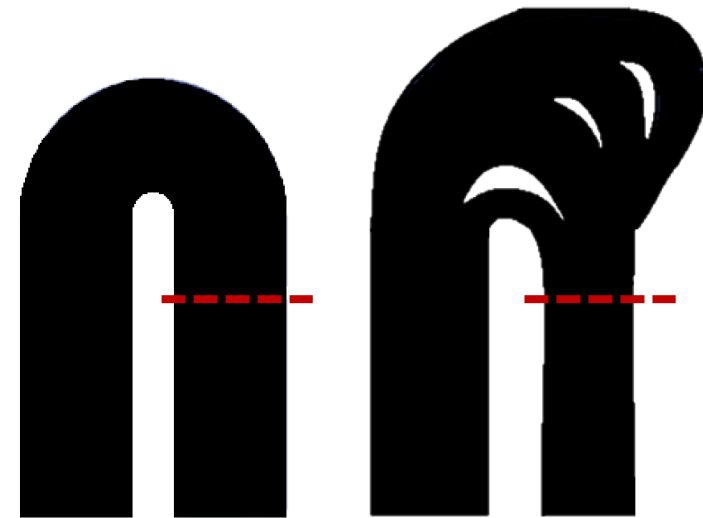


— Standard geometry
— Optimised geometry

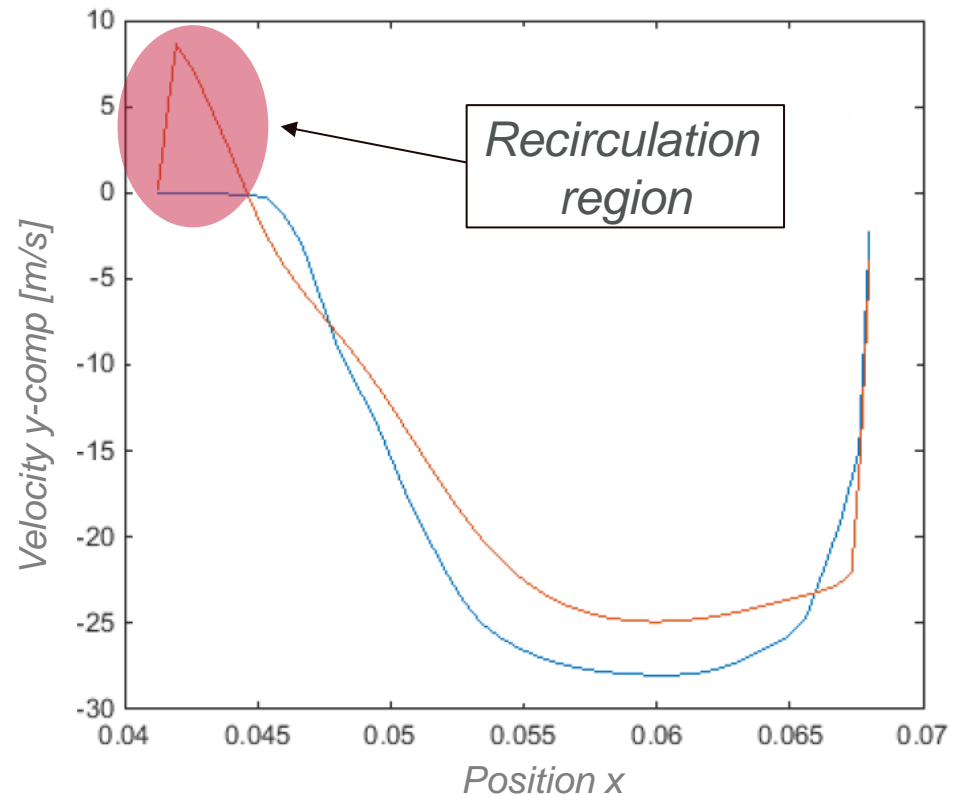


Results – Velocity Profile

Velocity profile across the cutting red line for inlet velocity 17.5 m/s

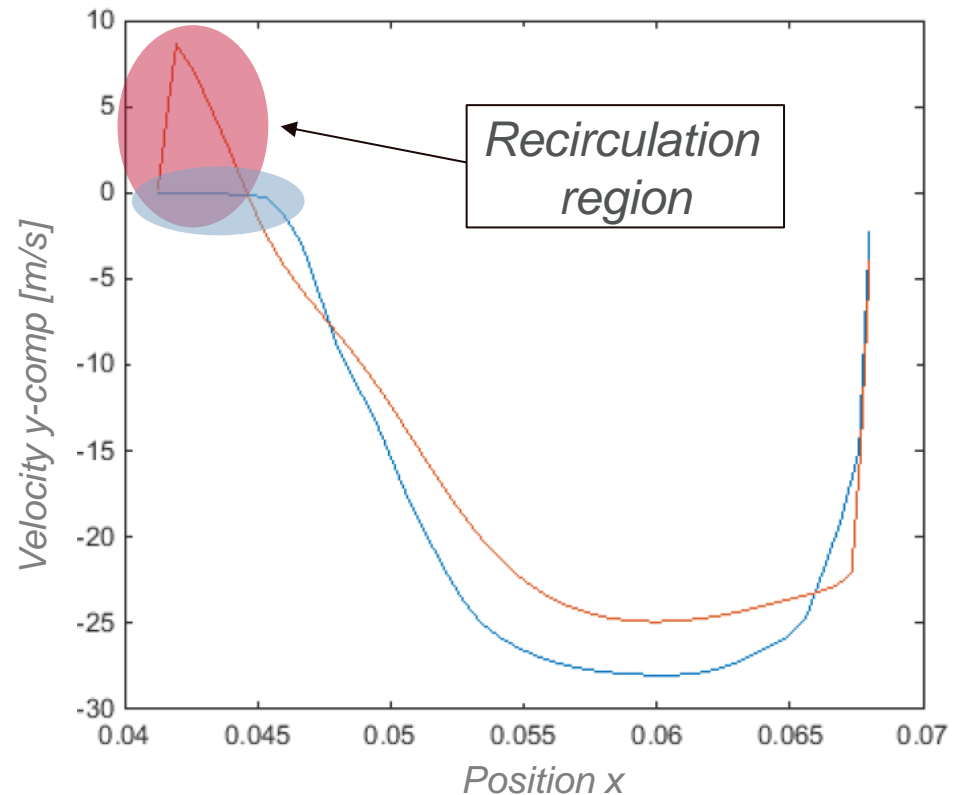
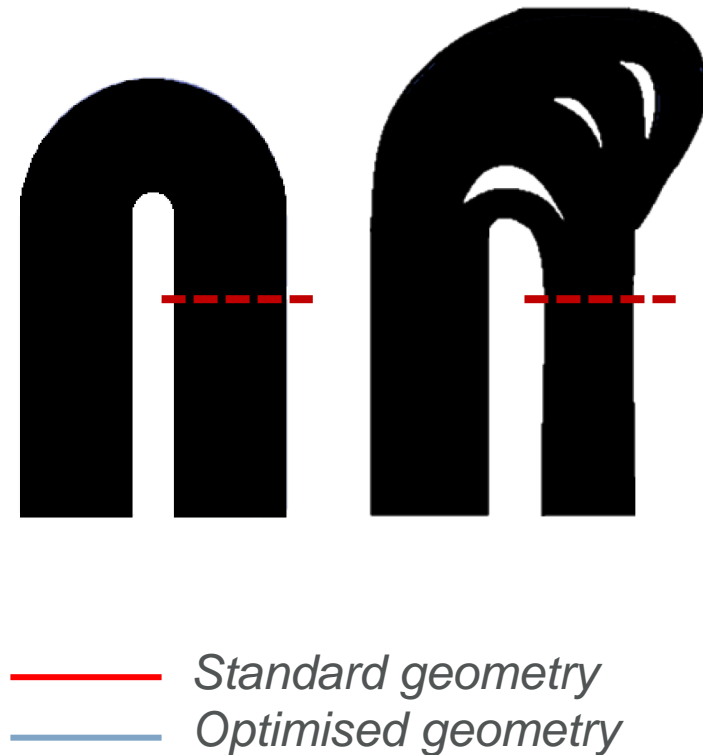


— Standard geometry
— Optimised geometry



Results – Velocity Profile

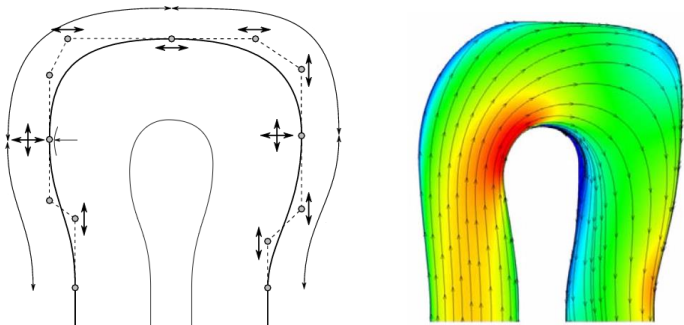
Velocity profile across the cutting red line for inlet velocity 17.5 m/s



TO and other Optimisation Methods

TO shows better performances compared to other optimisation methods

[T. Verstraete et al. GT2011 – 46541]



[Pietropaoli et al ASME IGTI 2017]

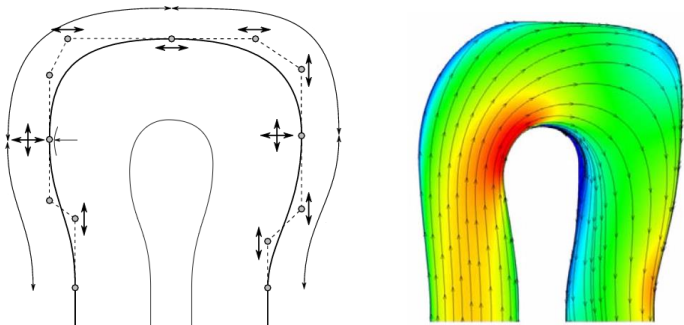


<i>Method</i>	<i>Improvement</i>
<i>Genetic Alg. + Bezier parameter.</i>	<i>~ 37%</i>
<i>Adjoint Opt. + Boundaries Disp.</i>	<i>~ 37%</i>
<i>Adjoint Opt. + Bezier parameter.</i>	<i>~ 47%</i>
<i>Adjoint Opt. + TO (aspect ratio (2:1))</i>	<i>~ 54%</i>
<i>Adjoint Opt. + TO (aspect ratio (2:2))</i>	<i>~ 60%</i>

TO and other Optimisation Methods

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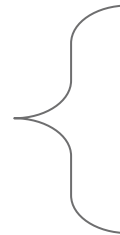
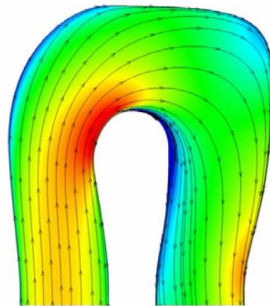
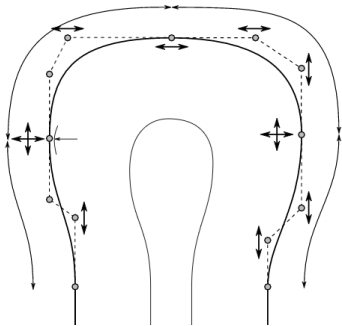


Method	Improvement
Genetic Alg. + Bezier parameter.	~ 37%
Adjoint Opt. + Boundaries Disp.	~ 37%
Adjoint Opt. + Bezier parameter.	~ 47%
Adjoint Opt. + TO (aspect ratio (2:1))	~ 54%
Adjoint Opt. + TO (aspect ratio (2:2))	~ 60%

TO and other Optimisation Methods

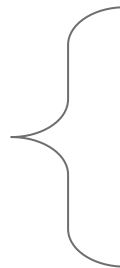
TO shows better performances compared to other optimisation methods

[T. Verstraete et al. GT2011 – 46541]



Shape Opt.
- 26 degree of freedom,
- ~100 CFD

[Pietropaoli et al ASME IGTI 2017]



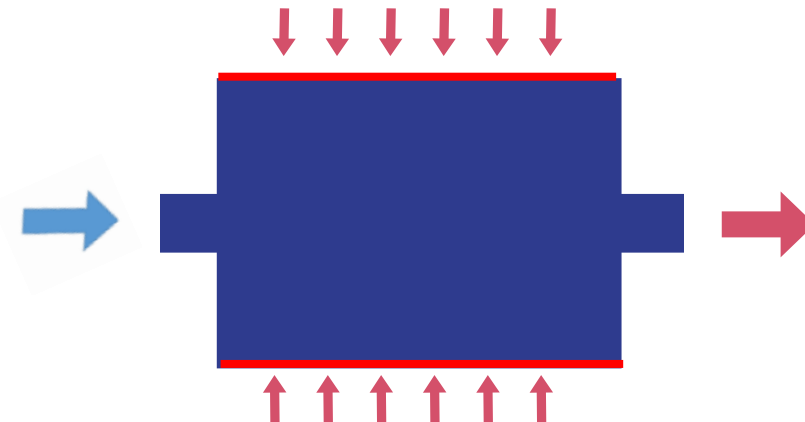
Topology Opt.
- ~ 1 million degree of freedom,
- ~5x CFD

[T. Verstraete et al. GT2011 – 46541]

Can we add heat transfer?

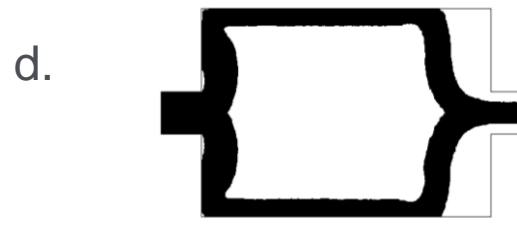
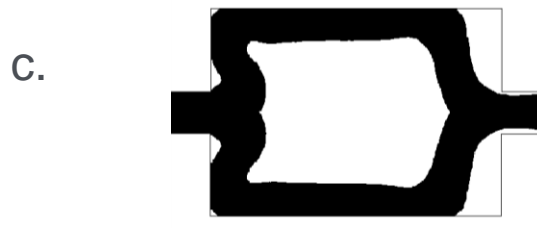
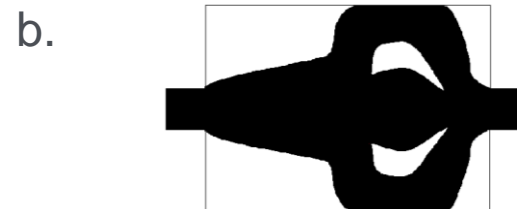
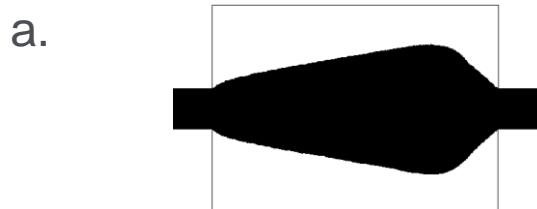
Heat Transfer and TOffee

- Energy equation for incompressible flow $v \cdot \nabla T = \frac{1}{\rho c} \nabla \cdot (k \nabla T)$
- Objective function: temperature gain of the flc $\int_{\Sigma} \rho c T v_n d\Sigma$



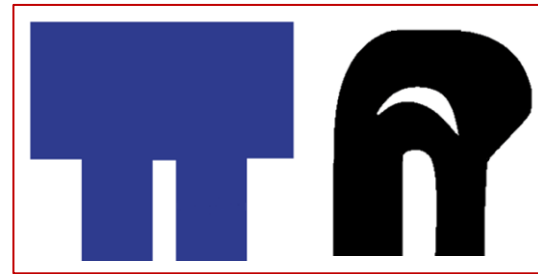
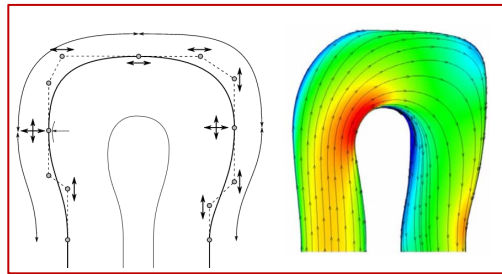
2D results reducing losses and increasing HT

	Weights	Pressure drop	Temperature gain
a.	$\hat{w}_1 = 1, \hat{w}_2 = 0$	73.7%	8.4%
b.	$\hat{w}_1 = 0.995, \hat{w}_2 = 0.005$	88.5%	12.7%
c.	$\hat{w}_1 = 0.99, \hat{w}_2 = 0.01$	94.6%	41.0%
d.	$\hat{w}_1 = 0.9, \hat{w}_2 = 0.1$	96.1%	49.7%

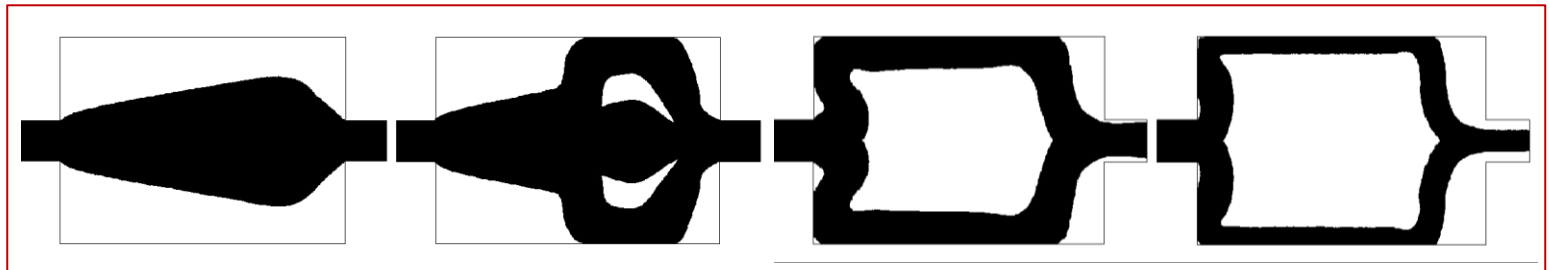


Recap (quick)

- *Pressure losses: optimisation of U – Bend. TOffee shows an improvement up to 60% higher than shape optimisation performed by VKI*

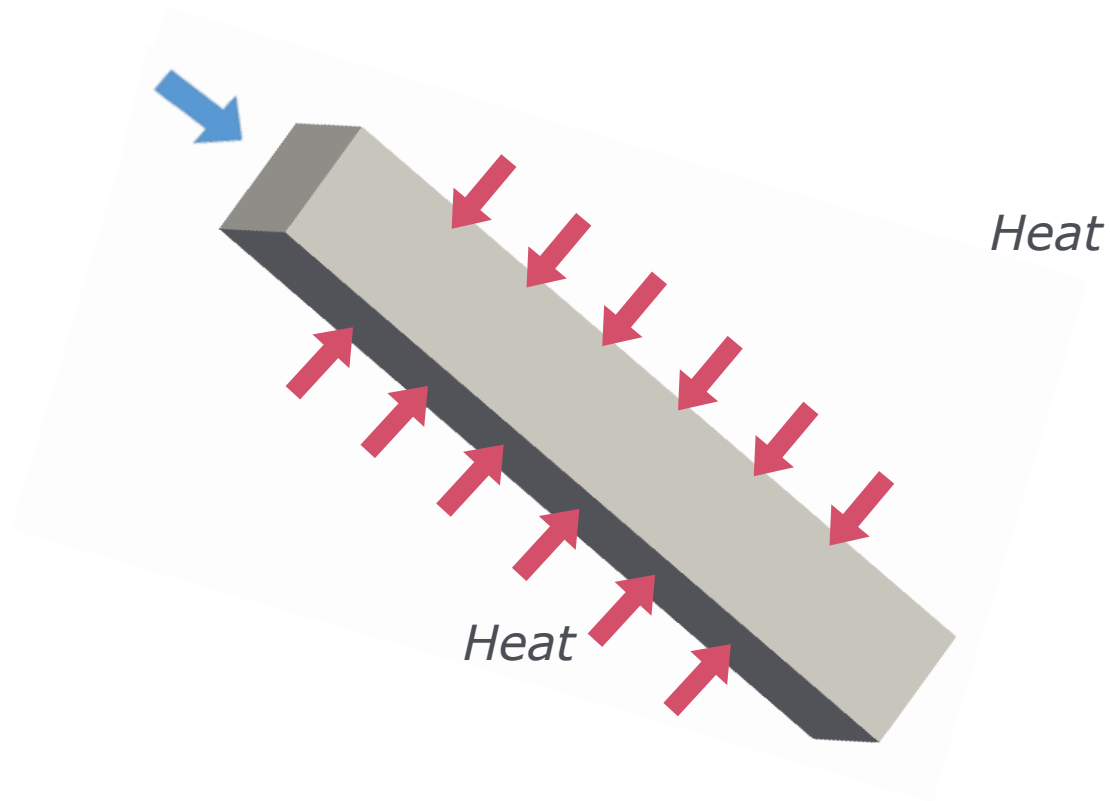


- *Heat transfer: main instability issues have been fixed. 2D results*



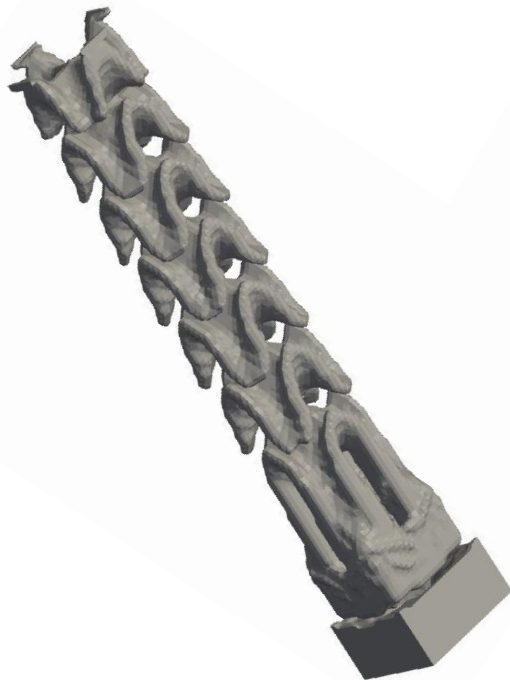
3D?

- *Squared duct test case*



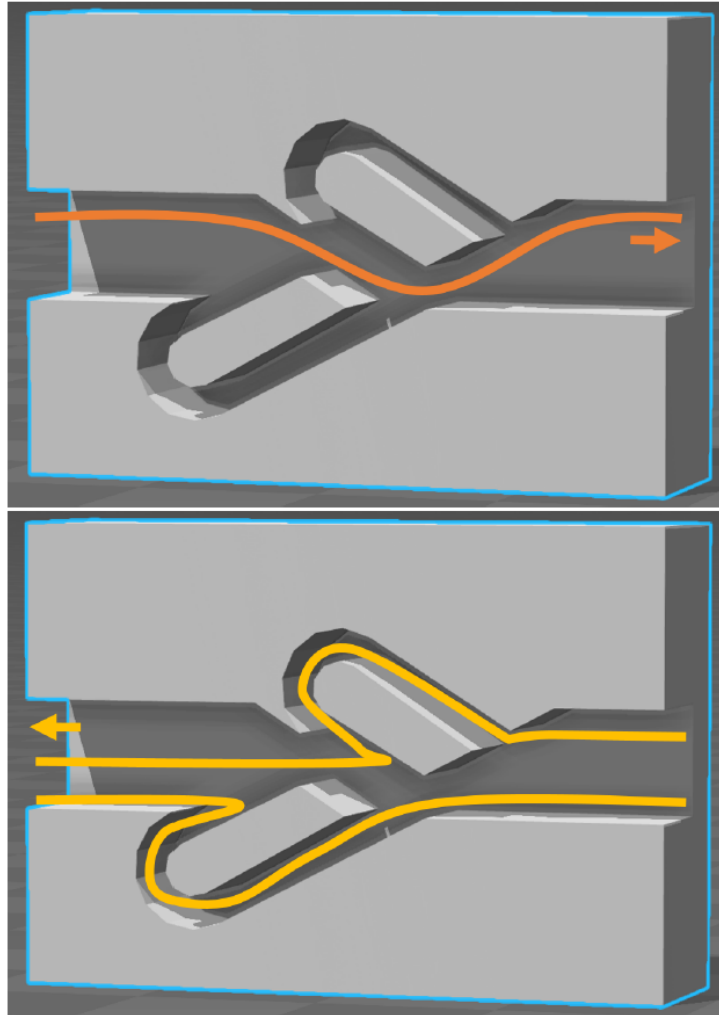
solution

- *Velocity streamlines generated from the inlet*



Can we build valves without moving parts?

Valves without moving parts?





Valves without moving parts?

Designed by TOffee....

low losses

High losses

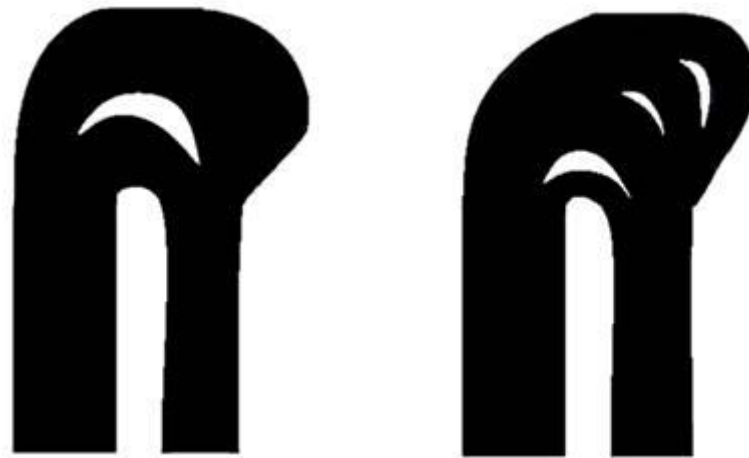


Robust Solutions

Problem 1: Solution dependent on BCs

Changing BCs gives different results/designs

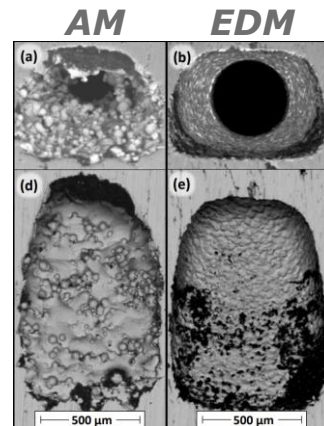
Example:



Problem 2: AM geometries affected by errors

AM surface roughness impact experimental results

We are not explaining here how to do it



[Effectiveness Measurements of Additively Manufactured Film Cooling Holes](#)

Paper: GT2017-64903; Author(s): Curtis K. Stimpson, Jacob C. Snyder, Karen A. Thole, Dominic Mongillo

Problem Statement



*Is it possible to tackle
uncertainties in the BCs
during TO?*

Boundary Conditions

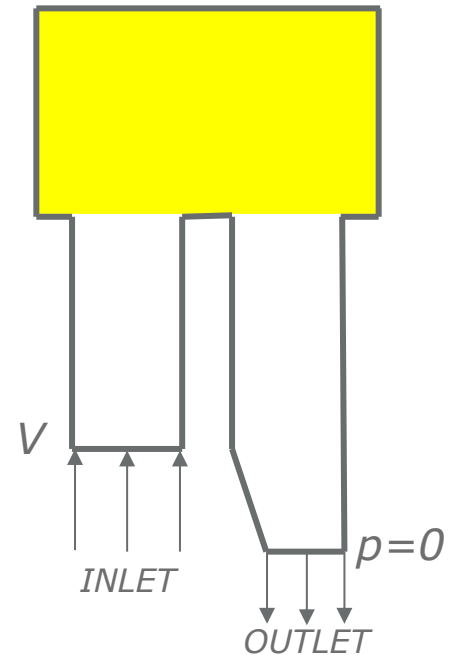
Inlet: uniform distribution velocity:

$$V = [V_{min}, V_{max}]$$

Outlet : atmospheric pressure

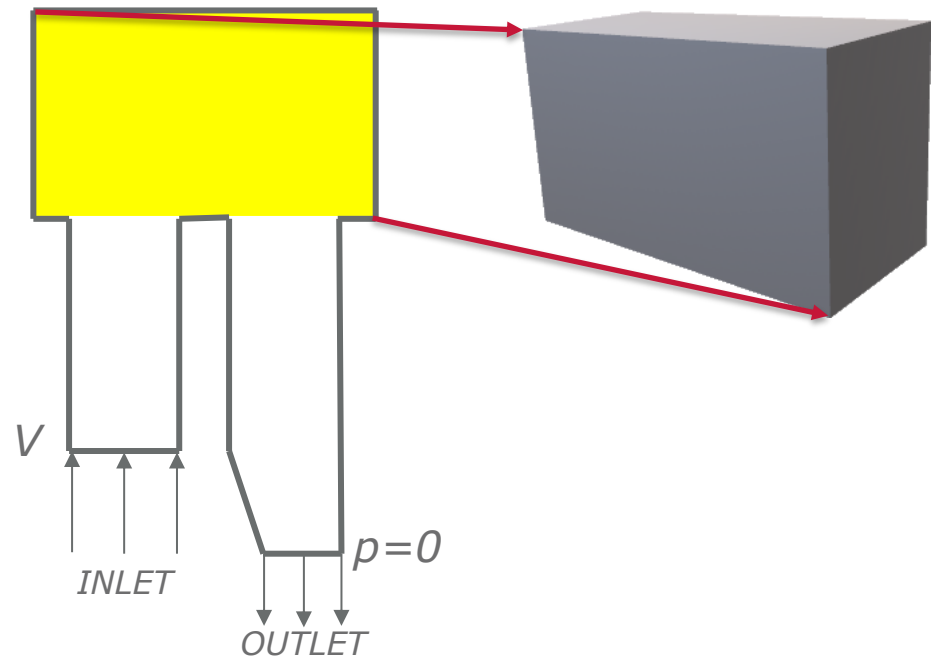
Wall boundaries everywhere else

Yellow volume given to the optimizer

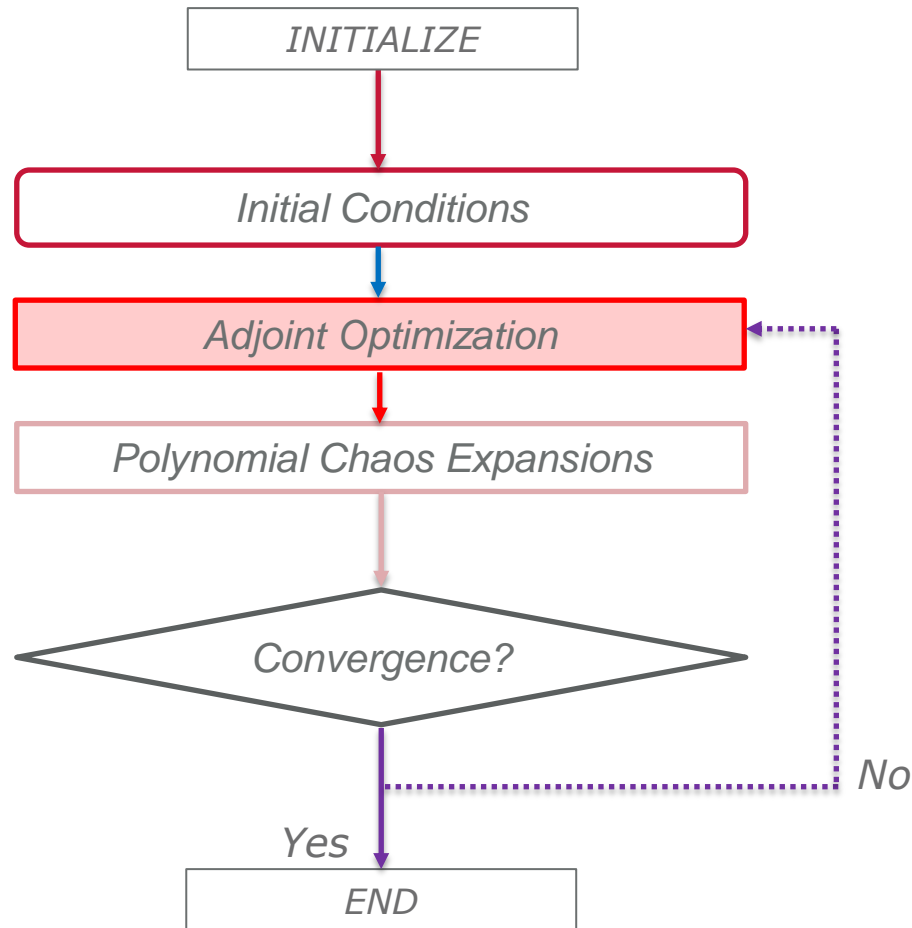


Boundary Conditions

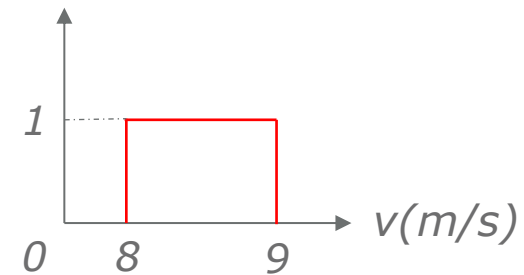
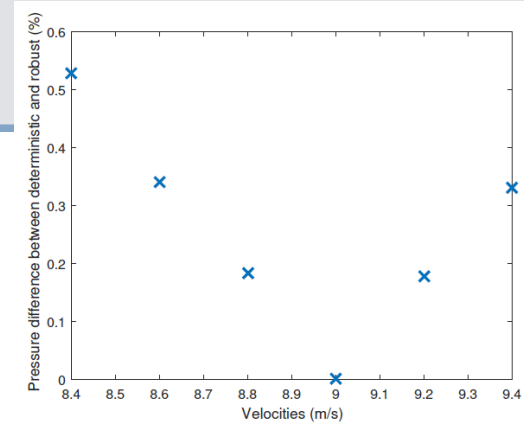
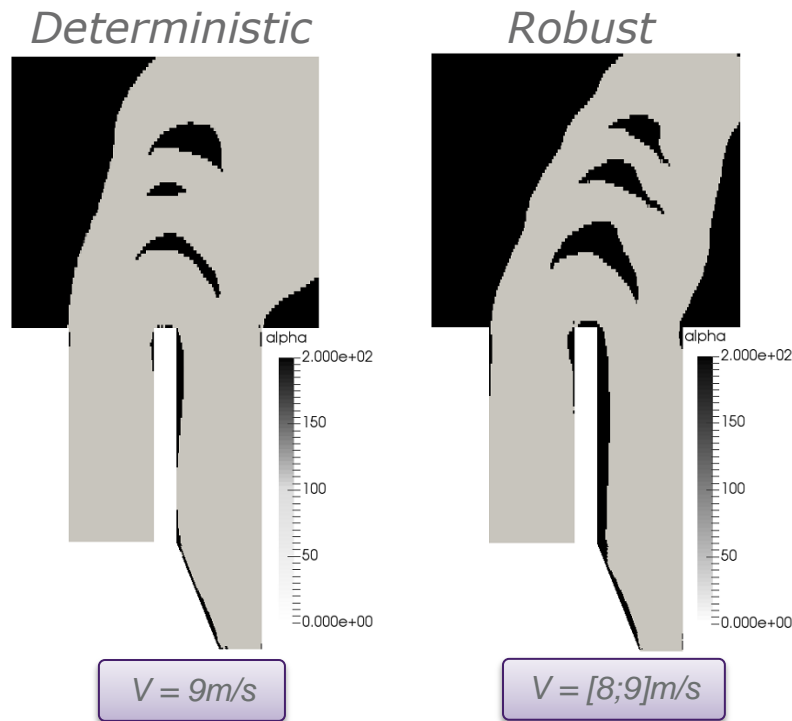
Optimizer is inherently 3D
2D obtained with one layer in the
third spatial dimension



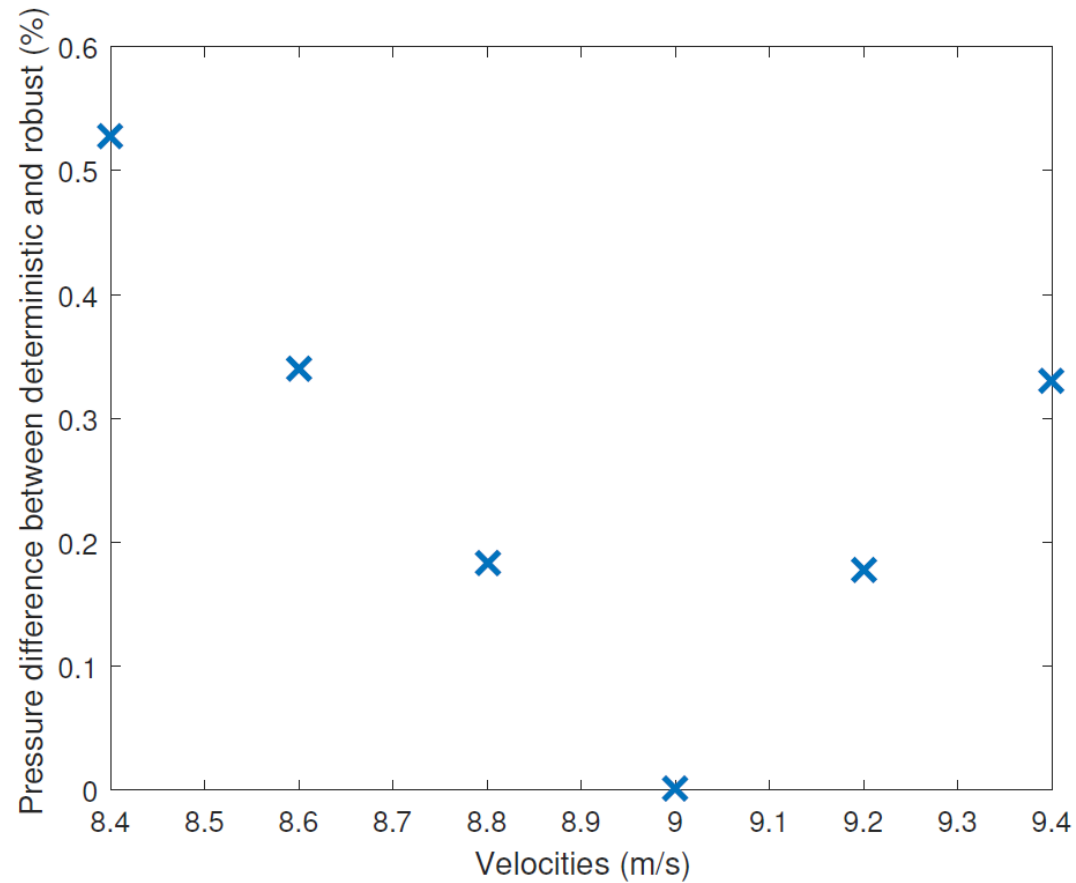
Governing equations - Polynomial Chaos Expansions



Results



Robust Results



Conclusions

- *Additive Manufacturing for the production of complex mechanical components for coolant systems*
- *Fluid Topology Optimisation is the way to exploit the flexibility of AM*
- *We have a framework to solve such problem*
- *SO vs TO: fluid TO less cost*

