

# Experimental and Numerical Study of Liner Film Cooling and Combustor Swirl Flow Interaction

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### Outline

- Aim and motivation
- Experimental setup
- Experimental results Flowfield
- Numerical setup
- Numerical results Flowfield
- oExperimental results Heat Transfer
- Conclusions





## Aim and motivation of this work

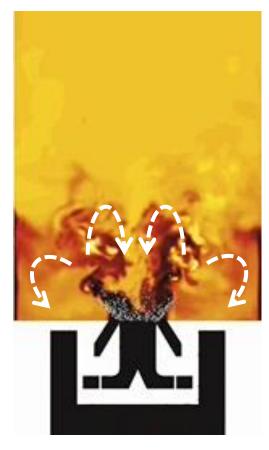
Swirl stabilized flames

Development of modern gas turbine combustors

- Reduction of NOx and soot pollutant emissions
  - Adoption of lean burn concept
  - Ground-based and aero-engine GTs
- Other challenging requirements:
  - High combustion efficiency and compact size combustors
  - Reliable ignition, combustion dynamics

 Highly swirled flows for flame stabilization and anchoring

- Large low speed regions produced by the onset of inner and outer recirculations
  - Continuous supply of high temperature gases to incoming fresh mixture
    - Strong velocity gradients and flow unsteadiness
      - » Greatly enhance freestream turbulence and then overall reaction and mixing rates



S. Puggelli, D. Bertini, L. Mazzei and A. Andreini, Assessment of Scale-Resolved Computational Fluid Dynamics Methods for the Investigation of Lean Burn Spray Flames. J. Eng. Gas Turbines Power 139(2), 2016



### Aim and motivation (continued)

Investigate the role of increasing slot film blowing ratio on the liner heat transfer coefficient under realistic swirling flow conditions

Generic axial swirlers installed in a linear three-sector rig

 Dedicated measurements of flow field (by PIV system) and Heat Transfer Coefficient (by a TLC thermography)

Description and understanding of basic mechanisms involved

Scale-resolving simulations to support flow field description

Validation with experimental data



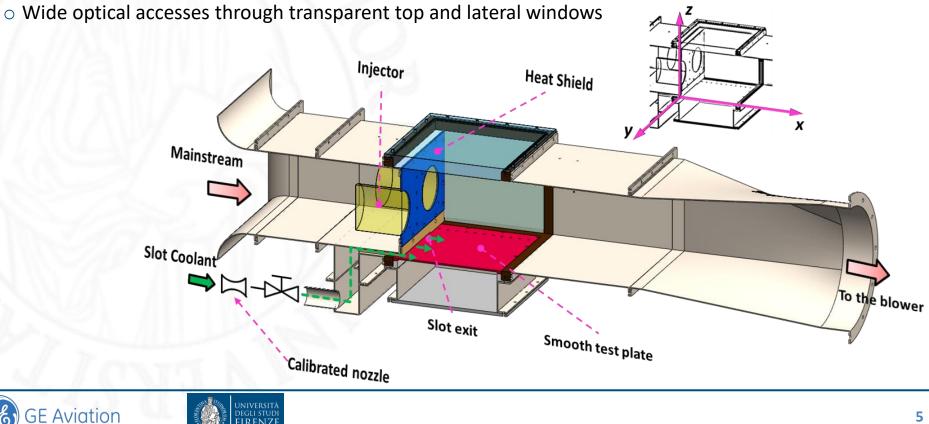


### **Experimental Setup**

#### Planar three-sector combustion chamber

Open Loop suction type wind tunnel (scaled ambient conditions)

- Slot cooling system fed by dedicated plenum chamber (slot lateral extension: 2 swirler pitchs)
- Smooth test plate representing the inner liner



### **Measurement Techniques**

#### Flow field investigation: Standard 2D Particle Image Velocimetry (PIV)

Two different planes investigated: Center plane and Wall plane
Several camera-laser positions to cover the investigation areas

Dantec Dynamic 2D PIV system

- 120mJ New Wave Nd:YAG pulsed laser 532nm
- Phantom Miro 320S camera
- Olive oil particles as seeding

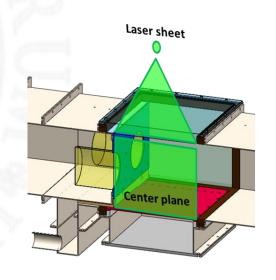
#### Settings

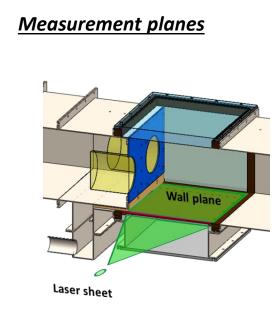
- Time delay: 10 μs
- Laser sheet thickness: 1mm

#### Data post processing

- Adaptive grid iterative method
- Peak-height validation and moving-average validation







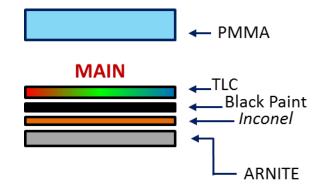
### **Measurement Techniques**

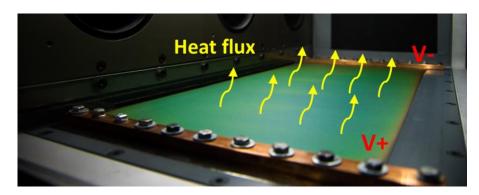
Heat transfer coefficient measurements

- Steady state technique with isothermal flows conditions
- Surface heat flux
  - Inconel heating foil 25.4  $\mu m$ 
    - Two copper bus bars on lateral sides
- Wall temperature measurement
  - TLC wide band 30-50°C

$$HTC = \frac{q_{conv}}{T_w - T_{aw}}$$

 Data post-processing exploiting a 3D thermoelectric FEM analysis









#### **Test conditions**

Rig inlet air at ambient pressure and temperature

Mainstream: Pressure drop across the injectors and Reynolds of the main-flow

- Reynolds number estimated using the cross sectional area of hydraulic diameter of test section

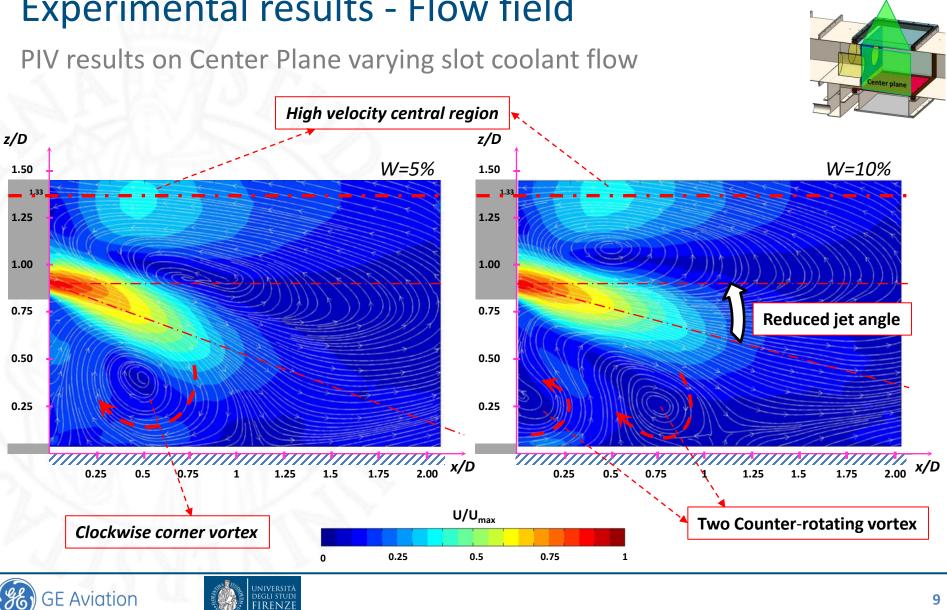
$$\frac{\Delta P}{P} = \frac{P_{in} - P_{out}}{P_{in}} \qquad \qquad Re_{main} = \frac{m_{main} \cdot d_h}{A_{main} \cdot \mu_{air}}$$

FIRENZI

Slot cooling system: Coolant flow parameter and Reynolds number

- Reynolds number estimated using the height of the slot exit

$W = \frac{m_{slot}}{m} \cdot \frac{3}{2} \qquad Re_{slot} = \frac{\dot{m}_{slot} \cdot h}{\Lambda}$		PIV	НТС
$m = \frac{m_{main}}{2}$ $m_{eslot} = \frac{1}{A_{slot} \cdot \mu_{air}}$	T <sub>main</sub> [K]	295	295
	ΔP/P [%]	5	5
P <sub>in,sx</sub> P <sub>in,cnt</sub> P <sub>out,cnt</sub>	W [%]	5;10	0; 5; 6; 7.5; 8.5; 10
P <sub>in,dx</sub> P <sub>out,dx</sub>	Re <sub>main</sub>	124000	124000
	Re <sub>slot</sub>	5200 - 8200	0 - 8200
			0



Laser sheet

#### **Experimental results - Flow field**

#### **Experimental results - Flow field** Wall plane PIV results on Wall Plane varying slot coolant flow Laser sheet x/D x/D W=10% W=5% 1.25 1.25 1.00 1.00 Jet stagnation region 0.75 0.75 0.50 0.50 0.25 0.25 $\frac{1}{1}$ 0 n 0.75 -0.75 -0.50 0.25 0.50 0.75 -0.75 -0.50 -0.25 0.25 0.50 -0.25 Ó y/D y/D U/U<sub>max</sub> **Coherent slot coolant structure** 0 0.1 0.2 0.3 0.4



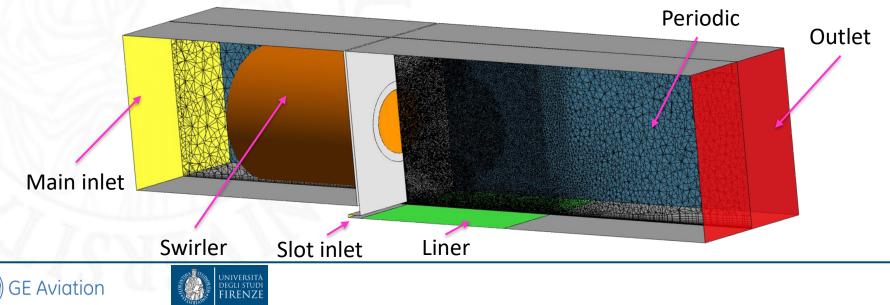


#### **Numerical Details**

Setup

- Code: ANSYS Fluent v16.2
  - SAS Scale-Adaptive Simulation
  - Pressure: 2nd order
  - Momentum: Bounded Central Difference
  - Other equations: 2nd order upwind
  - Time: bounded 2nd implicit
  - Time step: 7.5e-6 s (CFL < 1)</p>

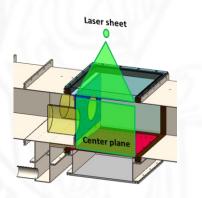
- Boundary conditions
  - Prescribed inlet mass-flow
    - Main and Slot entrances
  - Outlet: static-pressure
  - Lateral surfaces: periodicity
  - Walls: smooth, no slip, adiabatic
- Mesh: ANSYS Meshing
  - Tetrahedrons + 15 prisms ( $y^+ \approx 1$ )
  - 45 elements at the swirler exit
  - Roughly 14.25Me, 3.43Mn

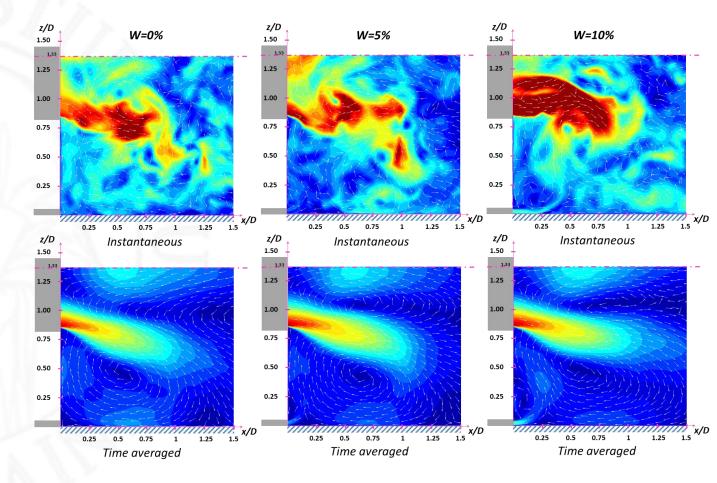


### **Numerical Results**

Flow field

Increase in W is affecting high speed swirling jet region
Highly unsteady and turbulent flow field in the shear layers







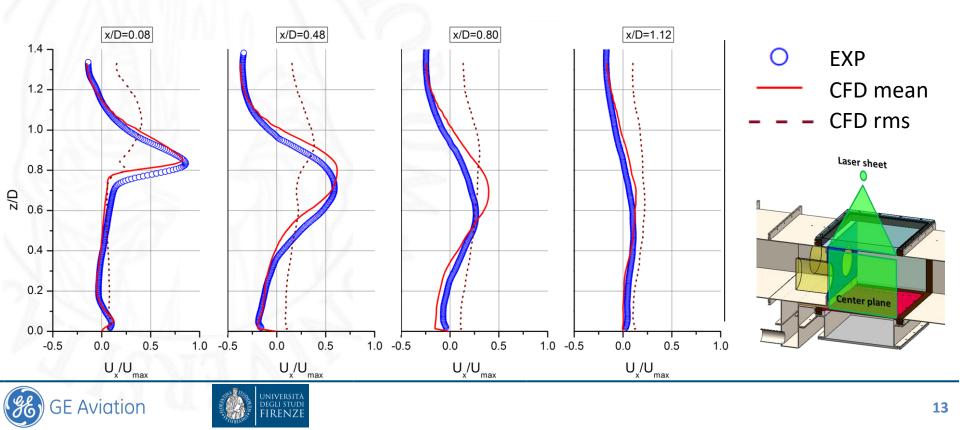


### **Numerical Results**

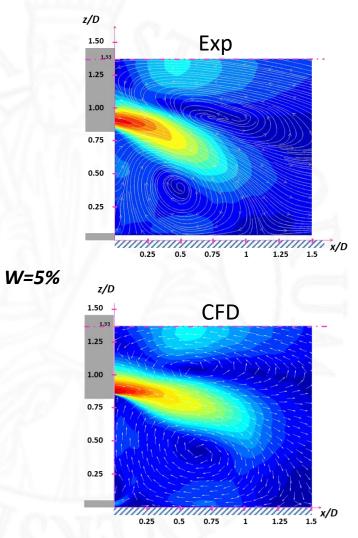
Comparison with experimental data

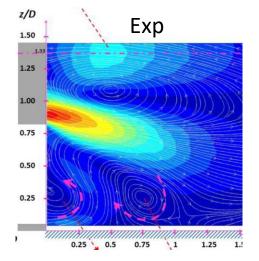
Reference condition: W=5%

- Reasonable agreement between Exp and CFD
- Differences due to to turbulence modelling & geometrical inconsistency due to manufacturing

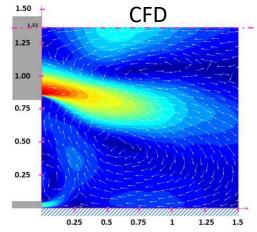


#### Numerical vs. Experimental





W=10%





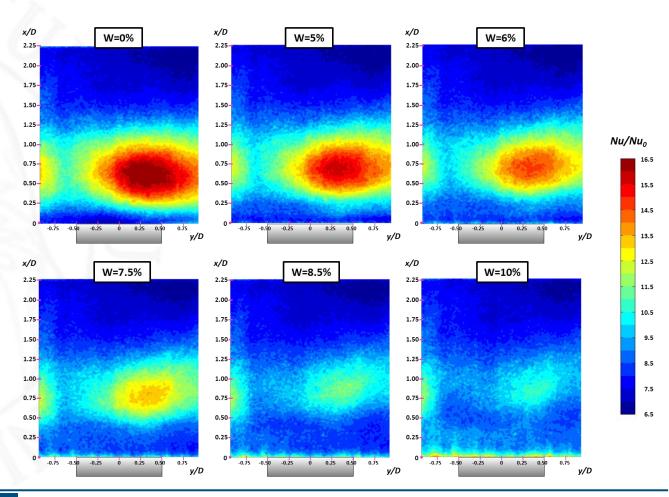


### Experimental results - Heat transfer

2D maps - Heat transfer enhancement factor

- High heat transfer region due to swirling jet impingement
- Slot coolant increase leads to reduced Nu/Nu<sub>0</sub> peak values
  - Mainstream jet lift
  - Corner vortex structures counteracting
- Increasing convection at the slot exit
- Cooling system effects negligible downstream of x/D≈1.6

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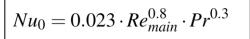


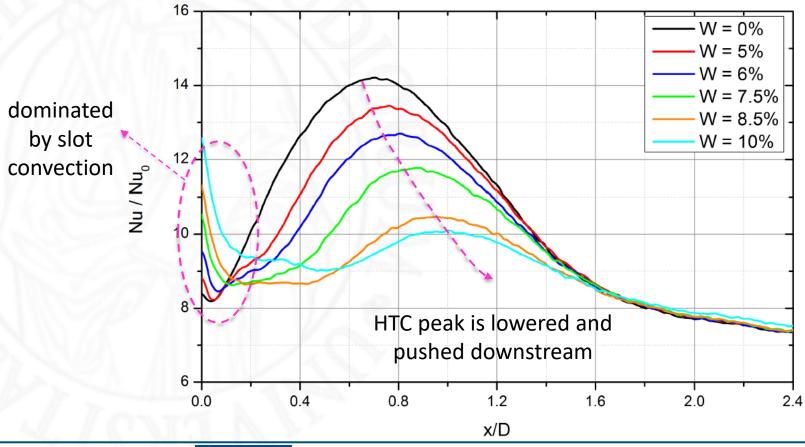


#### Experimental results - Heat transfer

Laterally averaged axial profiles

Non monotonic behavior of HTC with slot coolant flow W







#### Conclusions

Interaction between swirling main flow and slot cooling flow is complex

• Effects of coolant injection on flow field and liner heat transfer studied

- -2D Particle Image Velocimetry
- -Steady state technique using Thermochromic Liquid Crystals
- -Scale resolving CFD simulations

Slot cooling has significant effect on combustor liners

- -Has impact on flow field
- Substantial impact on heat transfer augmentation with respect to classical smooth channel correlations



