

Ansys Turbomachinery modelling with Aeromechanics focus

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Ansys

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/Agenda

Overview, Challenges & Trends

ANSYS Turbo Systems & Workflow

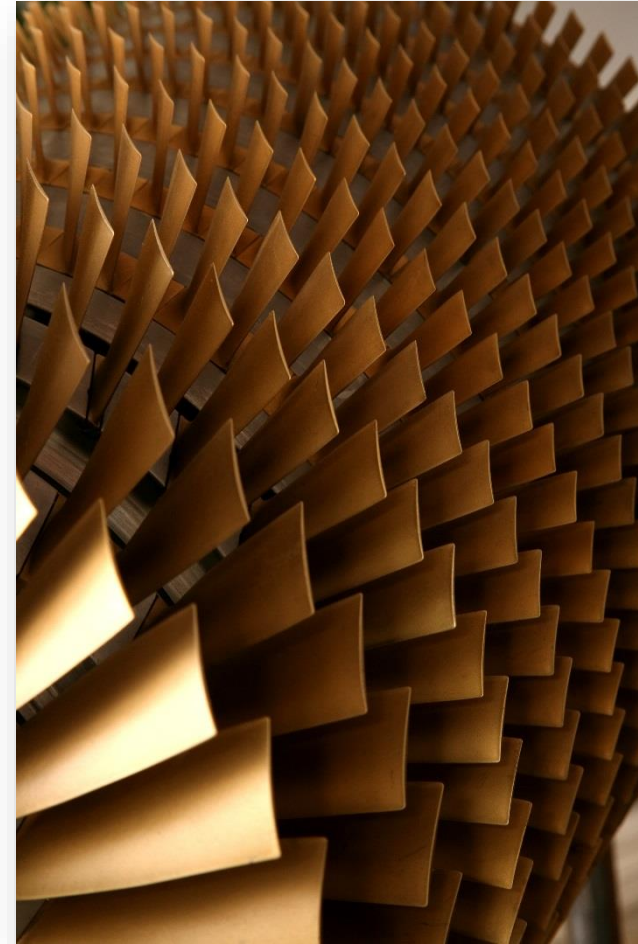
Rotor/Stator Coupling

- Steady State Models
- Transient Models

Aeromechanics

- Flutter
- Forced Response
- Thermal, Static, Modal,...

Automatization & Optimization



Overview, Challenges & Trends



/ Turbomachinery Simulations Trends

Complex geometries

Unsteady simulation

Rotor/stator interaction

Off design performance

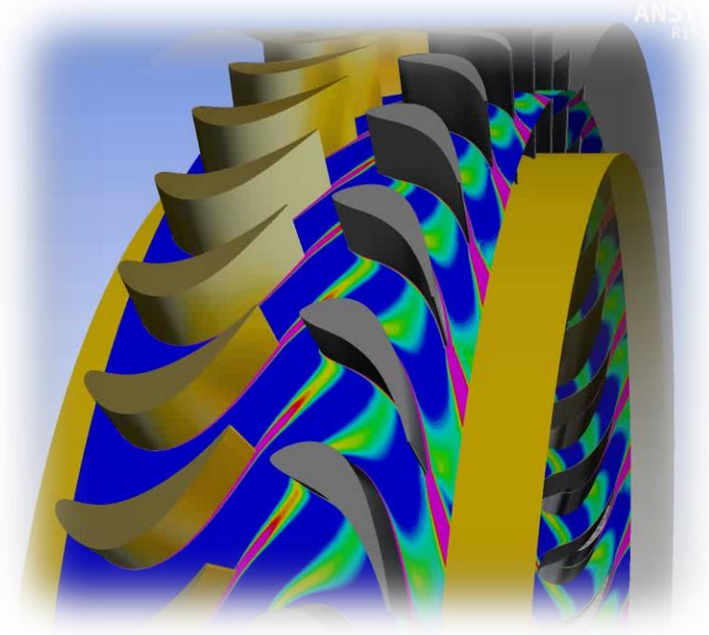
Inlet distortion

Acoustics

Multidisciplinary

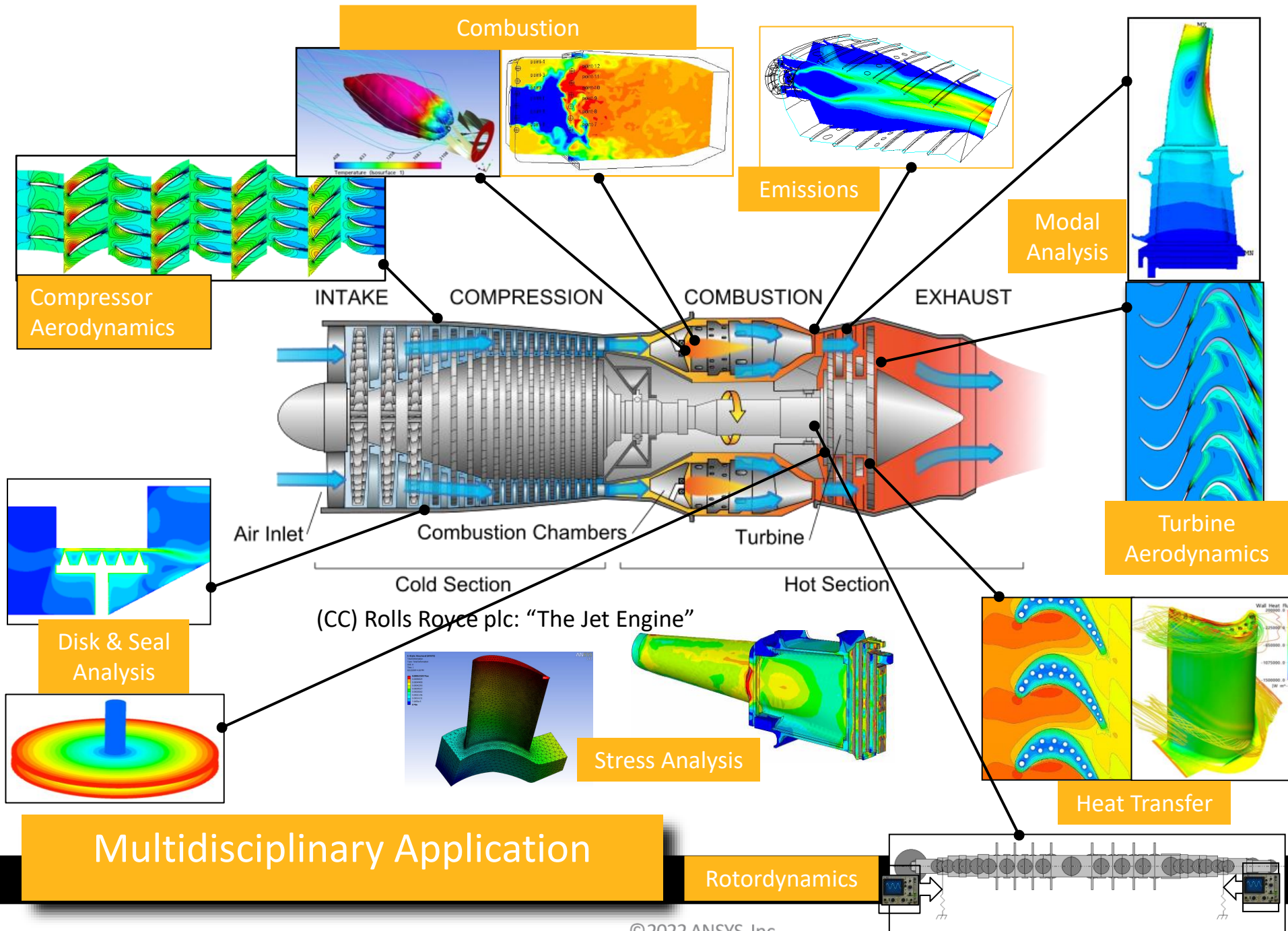
Conjugate heat transfer

Aeromechanics



Optimization & robust design

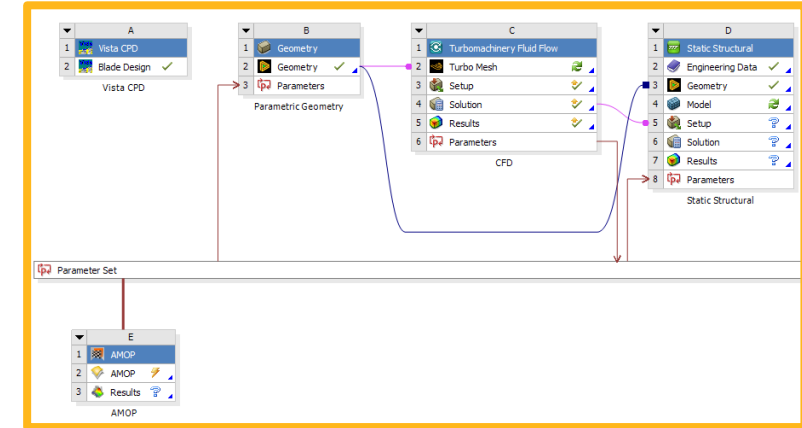
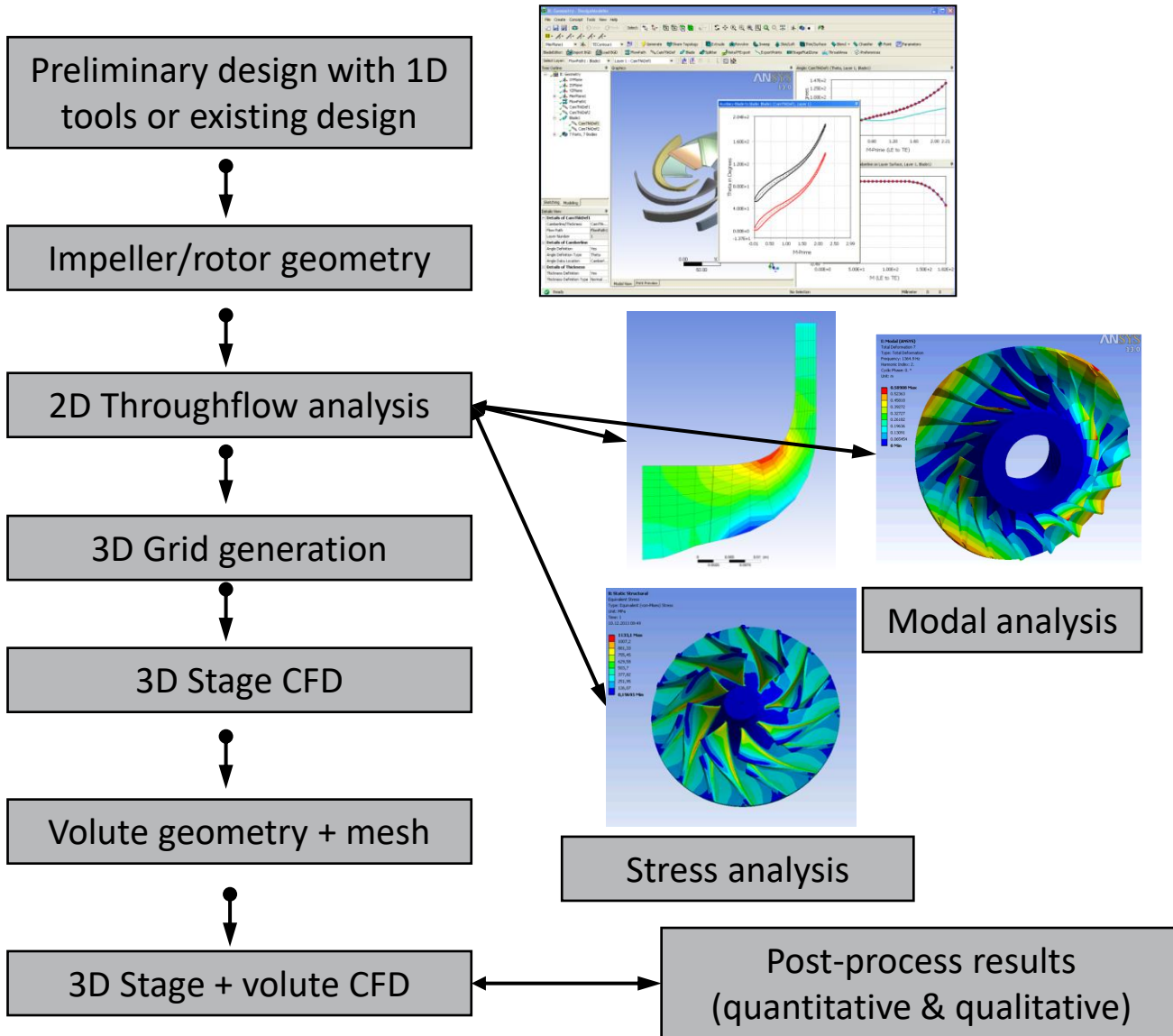
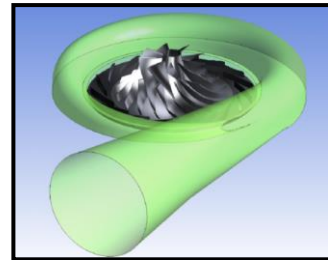
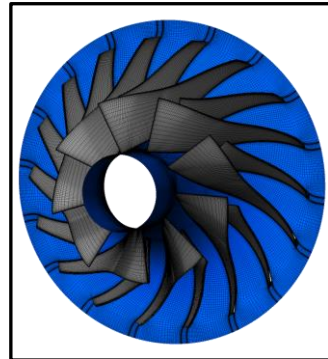
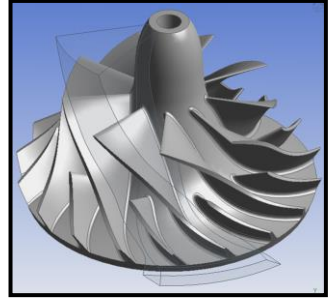
Complete system simulation



ANSYS Turbo Systems & Workflow

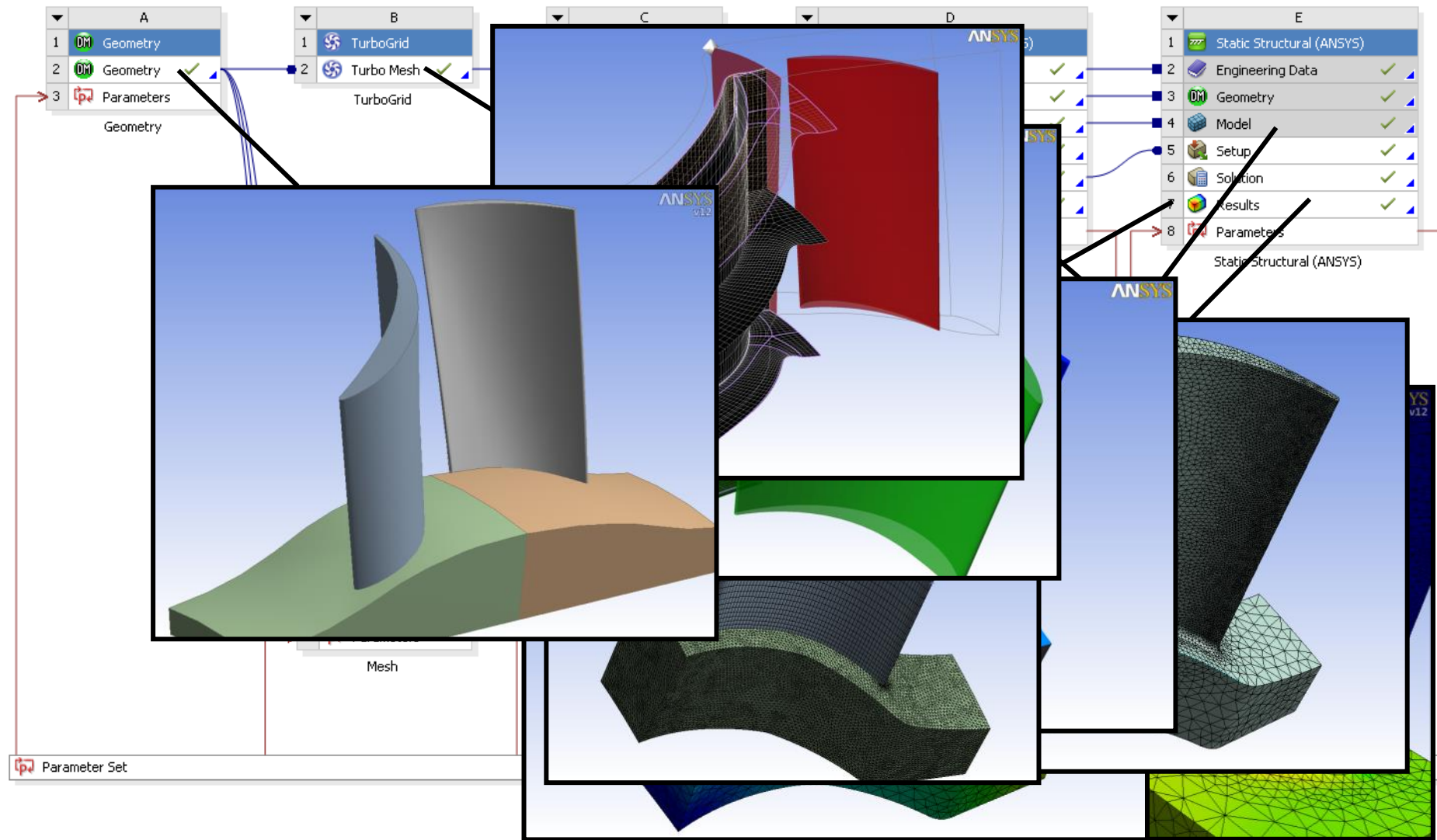


Design Workflow for a Centrifugal Compressor

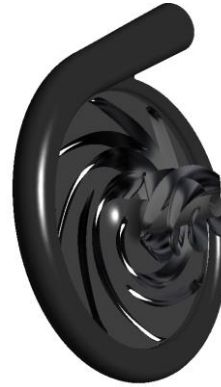


Ansys Workbench workflow for Compressor/Pump/Turbine Analysis

ANSYS Workbench – Parametric Workflow



Automated Machine Performance prediction

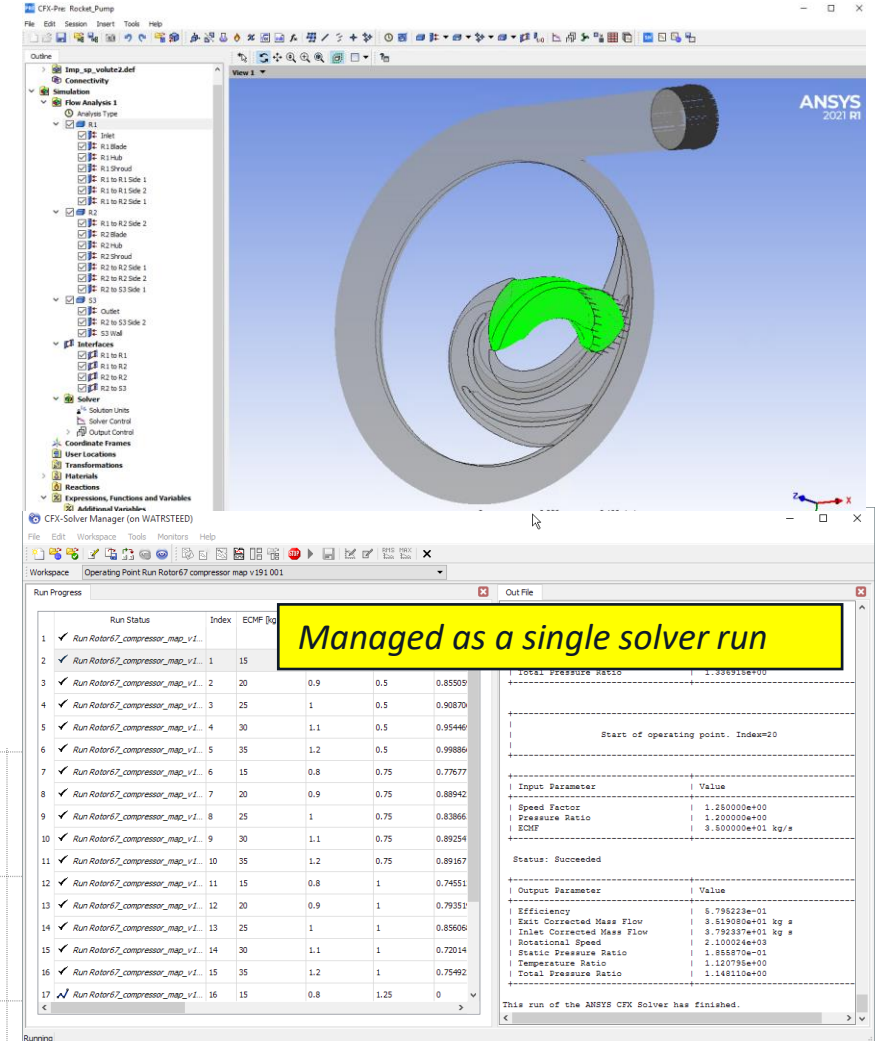
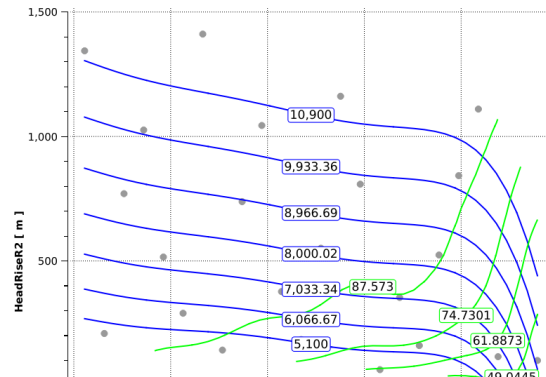


Input data
(Geometry, Operating conditions)

Generate Meshes, Build CFD Setup

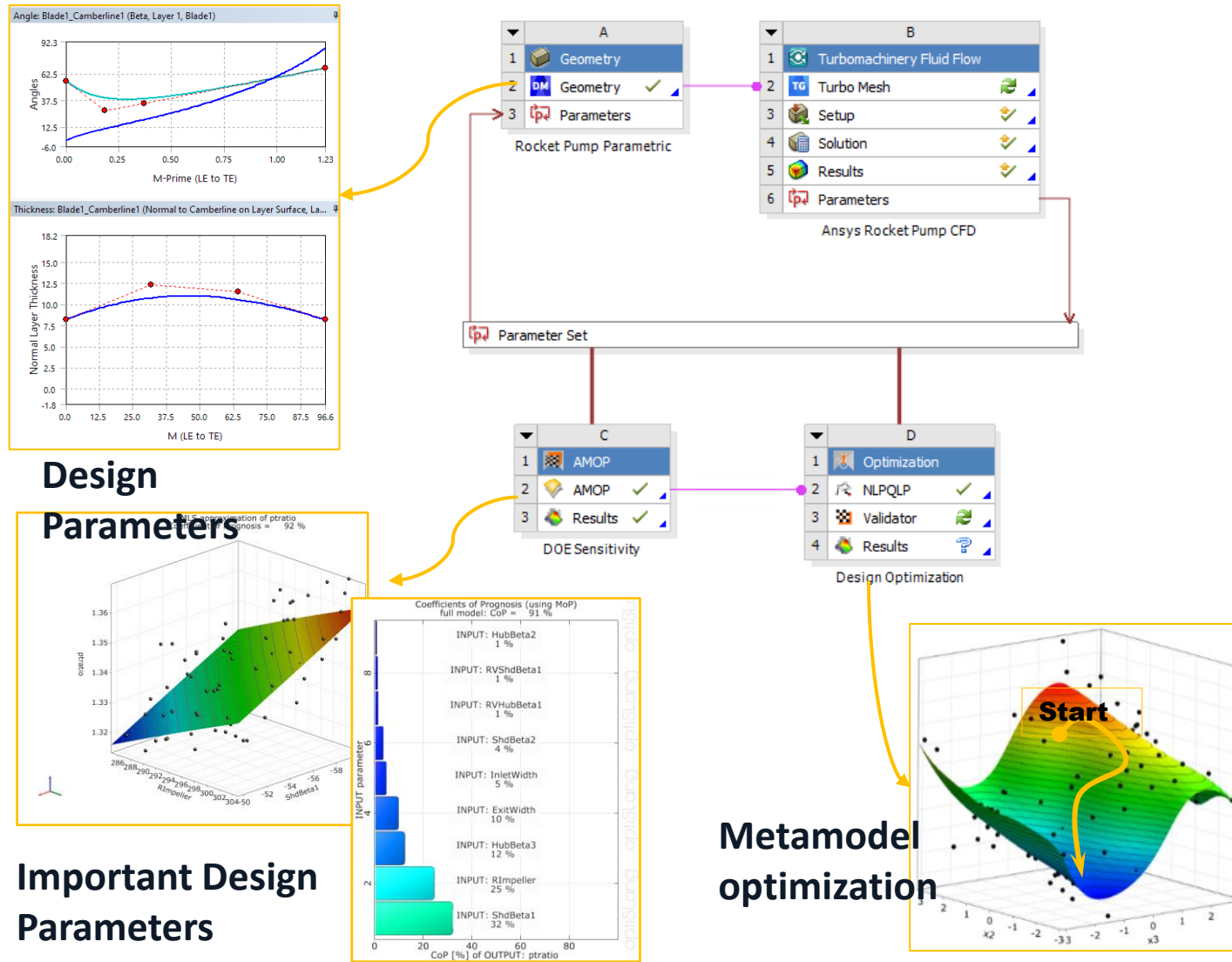
Run Parallel CFD Simulations

Performance map plot + Additional post-processing



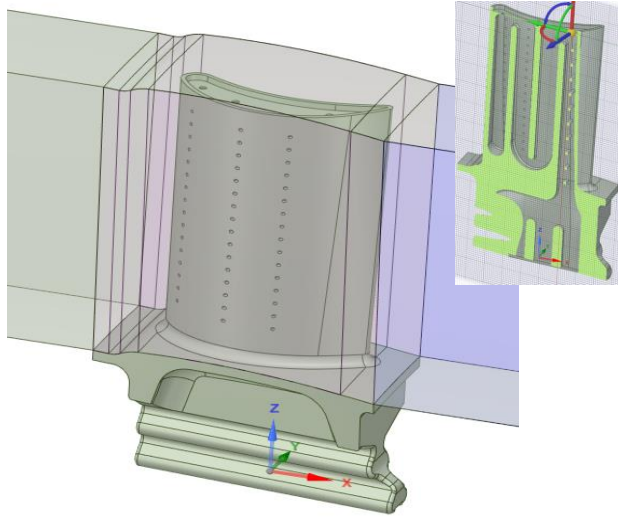
Accurate performance map generation in less than 30 minutes

Aero-Mechanical Design Optimization

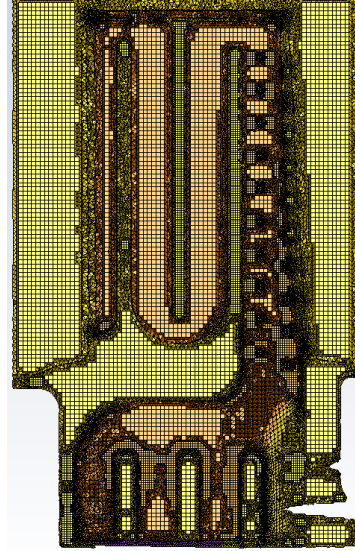


Optimized Inducer & Impeller of a Rocket Pump
(200 Design Points, run concurrently for 3 hour
total wall clock time to get optimized design)

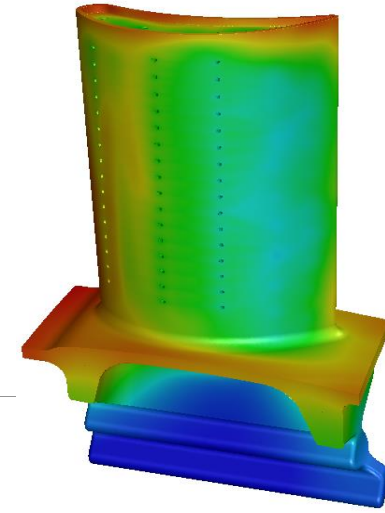
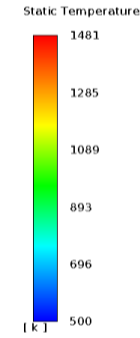
Conjugate Heat Transfer



*Automated Geometry Clean-up
and Volume Extraction*

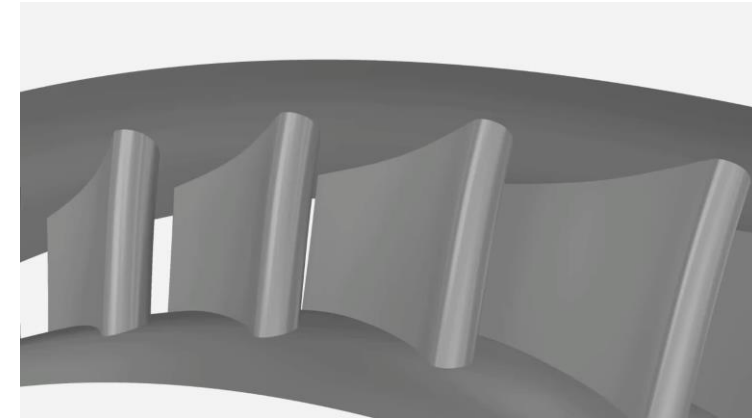


High Quality mesh in 5 minutes



CHT Results

- CHT with fast transient statistics
- Cooled blades and combustor liner durability
- Blade film cooling

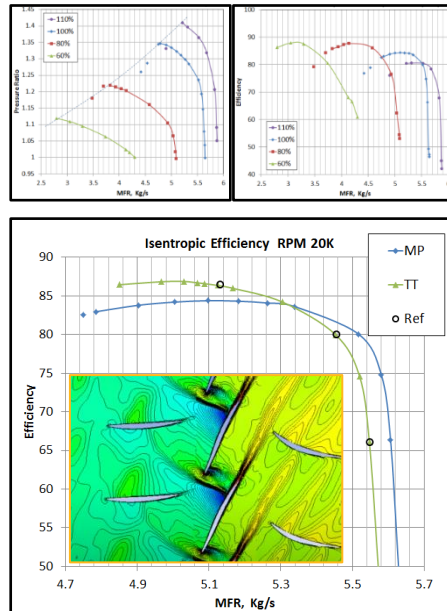


*Blade Film
Cooling*

Rotor/Stator Coupling

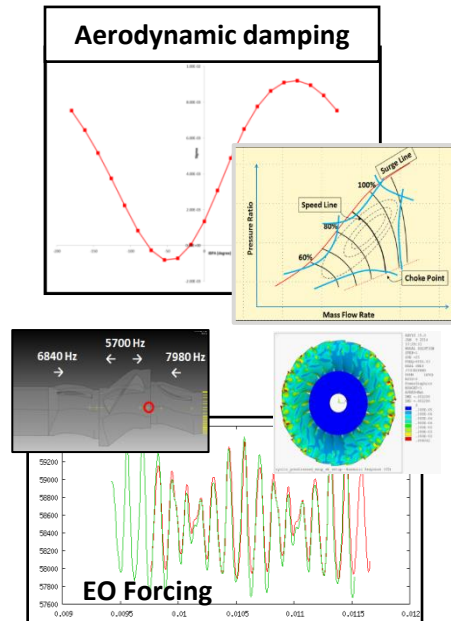
Turbomachinery Aero-Mechanical-Thermal Analysis

Aerodynamic Analysis



Steady & Transient

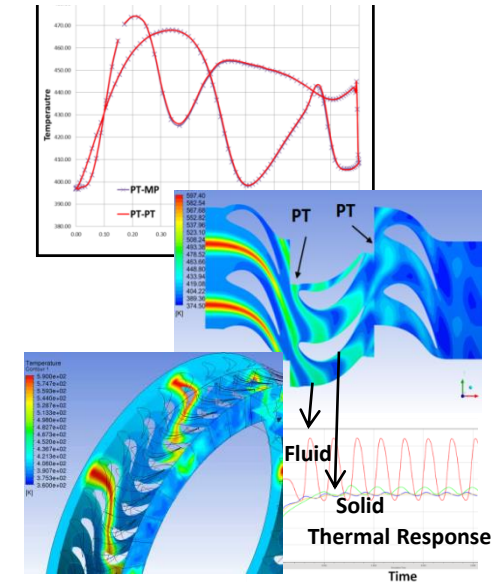
Aeromechanical Analysis



Transient

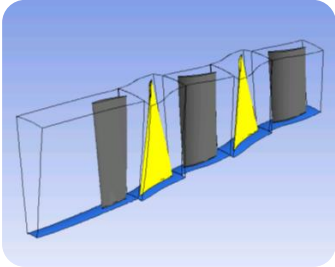
Aerothermodynamic Analysis

Surface Temperature Distribution

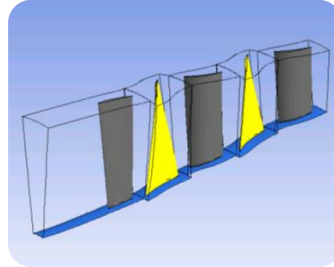


/ Steady State Rotor Stator Interfaces

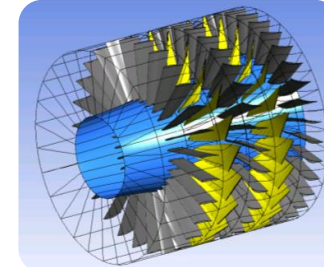
Steady
pitch change



Transient
pitch change



Transient
full-annulus

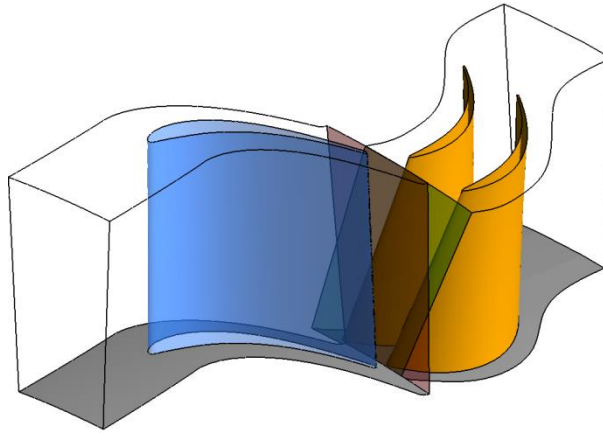


Stage interface (mixing plane)

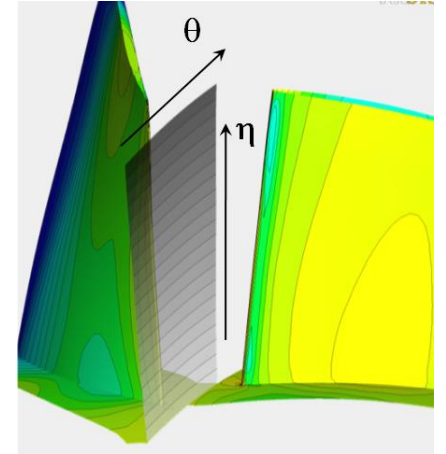
Frozen rotor interface

/ Steady State – Stage Interface (Mixing Plane)

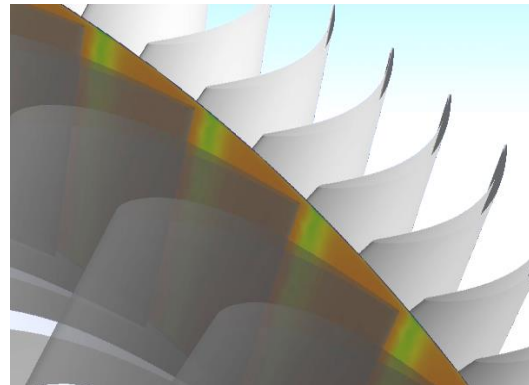
- Industry standard for steady state prediction of aerodynamic performance
- Blade pitch change via conservative “mixing” process in circumferential direction



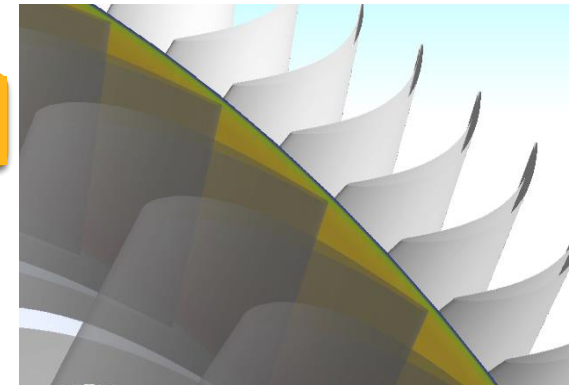
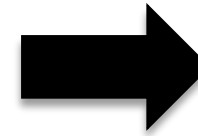
Upstream interface side



Downstream interface side

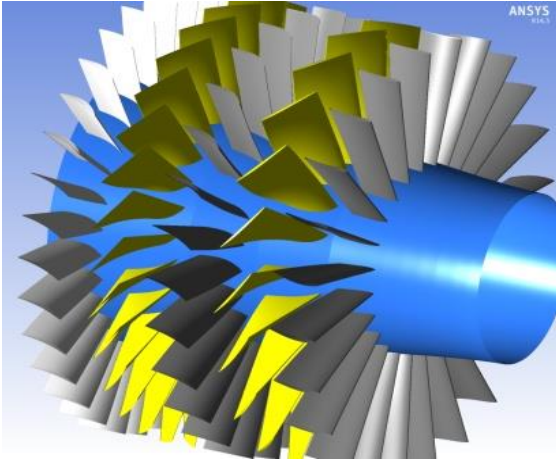


Mixing



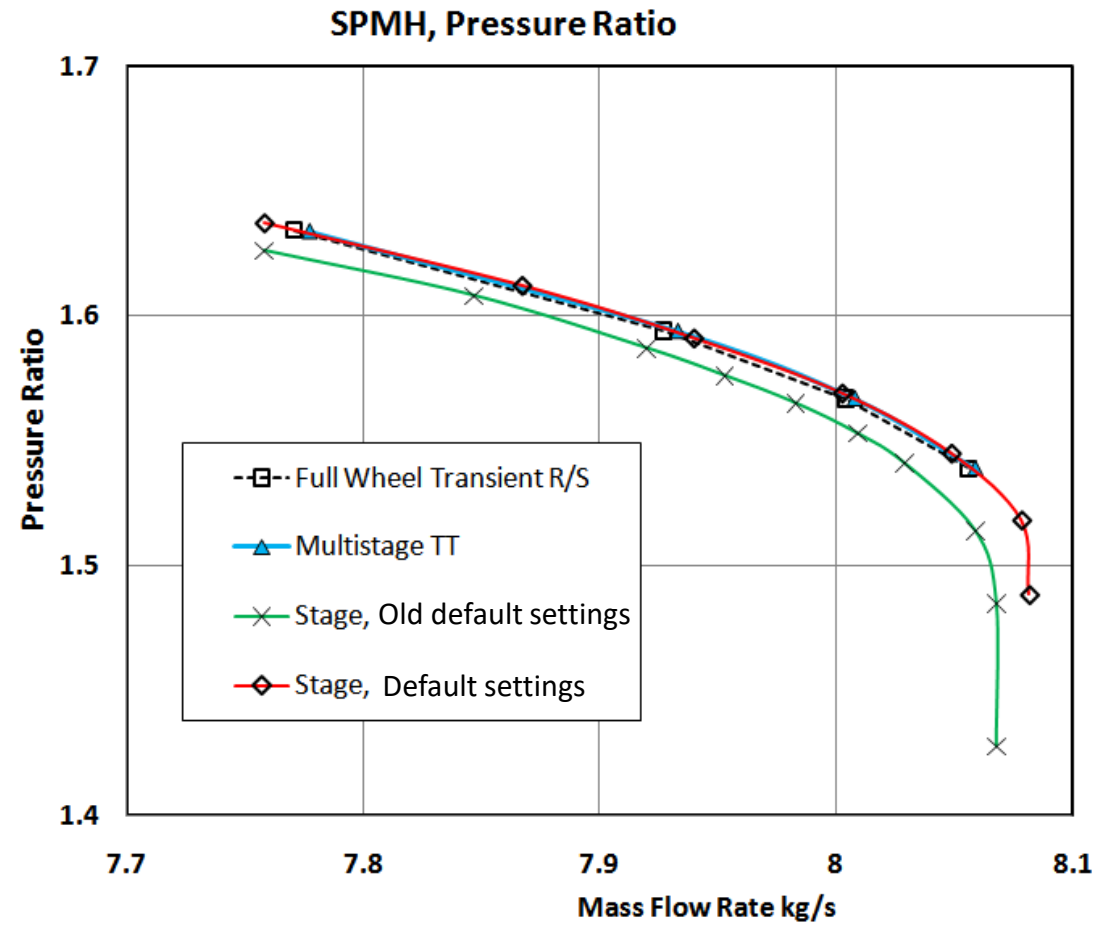
Example – 2.5 Stage Axial Compressor

Hannover compressor with modified blade counts



Design rotational speed	17100 rpm
Mass flow rate	7.82 kg/s
Total pressure ratio	2.7
Isentropic efficiency	89.8%
Inlet total pressure	60 kPa
Inlet Mach number	0.5

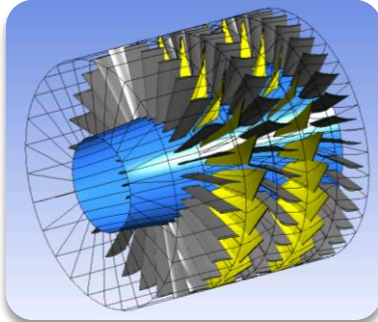
Ref. 1/3 wheel	11.6 mil nodes
TT, PT, mixing-plane	1.3 mil nodes



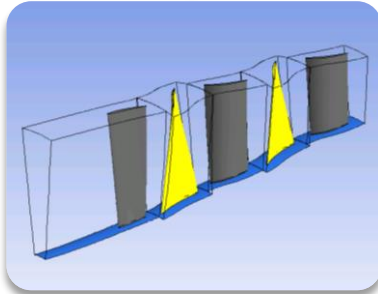
Transient Blade Row Simulations

Solve on Reduced Geometry

Full-wheel Model



Reduced Model



ANSYS CFX transient pitch-change models

Profile Transformation (PT)

Small/Moderate Pitch

- Single Stage
- Multistage

Time Transformation (TT)

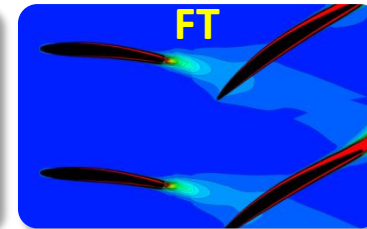
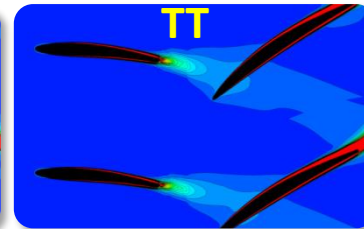
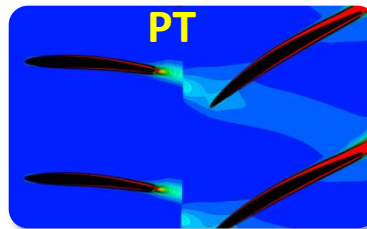
Small/Moderate Pitch

- Frozen gust
- Single Stage
- Multistage

Fourier Transformation (FT)

Large Pitch

- Frozen gust
- Single Stage
- Multistage
- Blade Flutter



Transient interaction

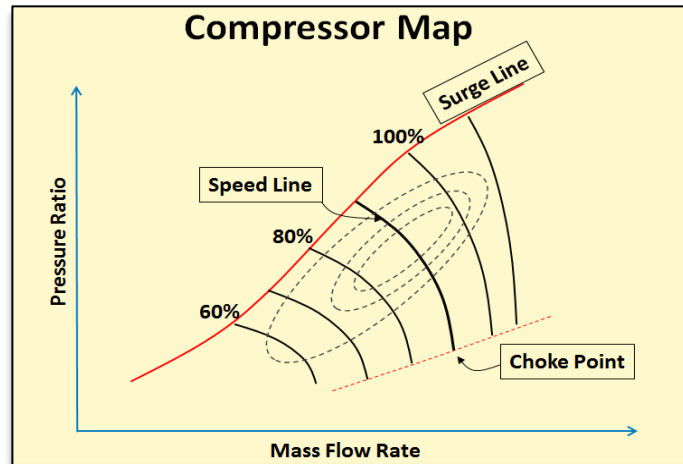
Transient interaction

+

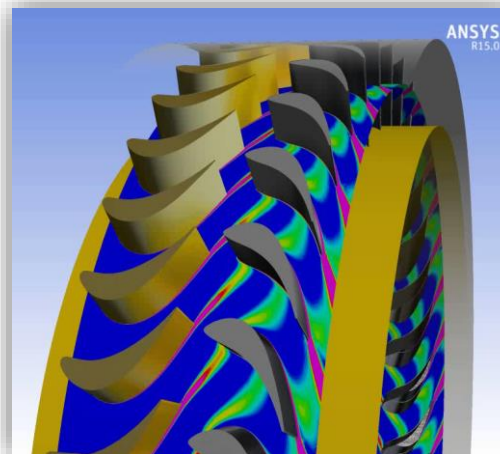
Correct blade passing frequencies

Transient Applications

Better performance prediction



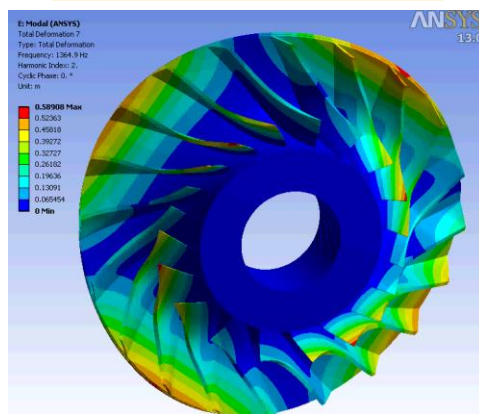
Unsteady interactions



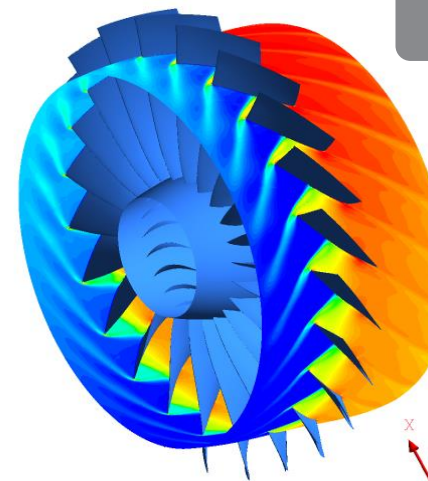
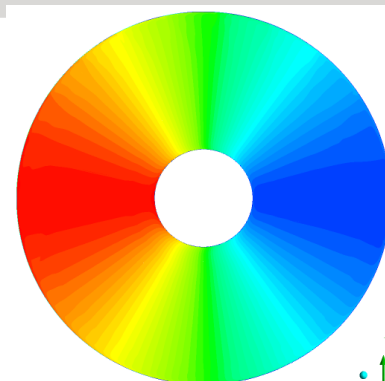
Acoustics



Aeromechanics

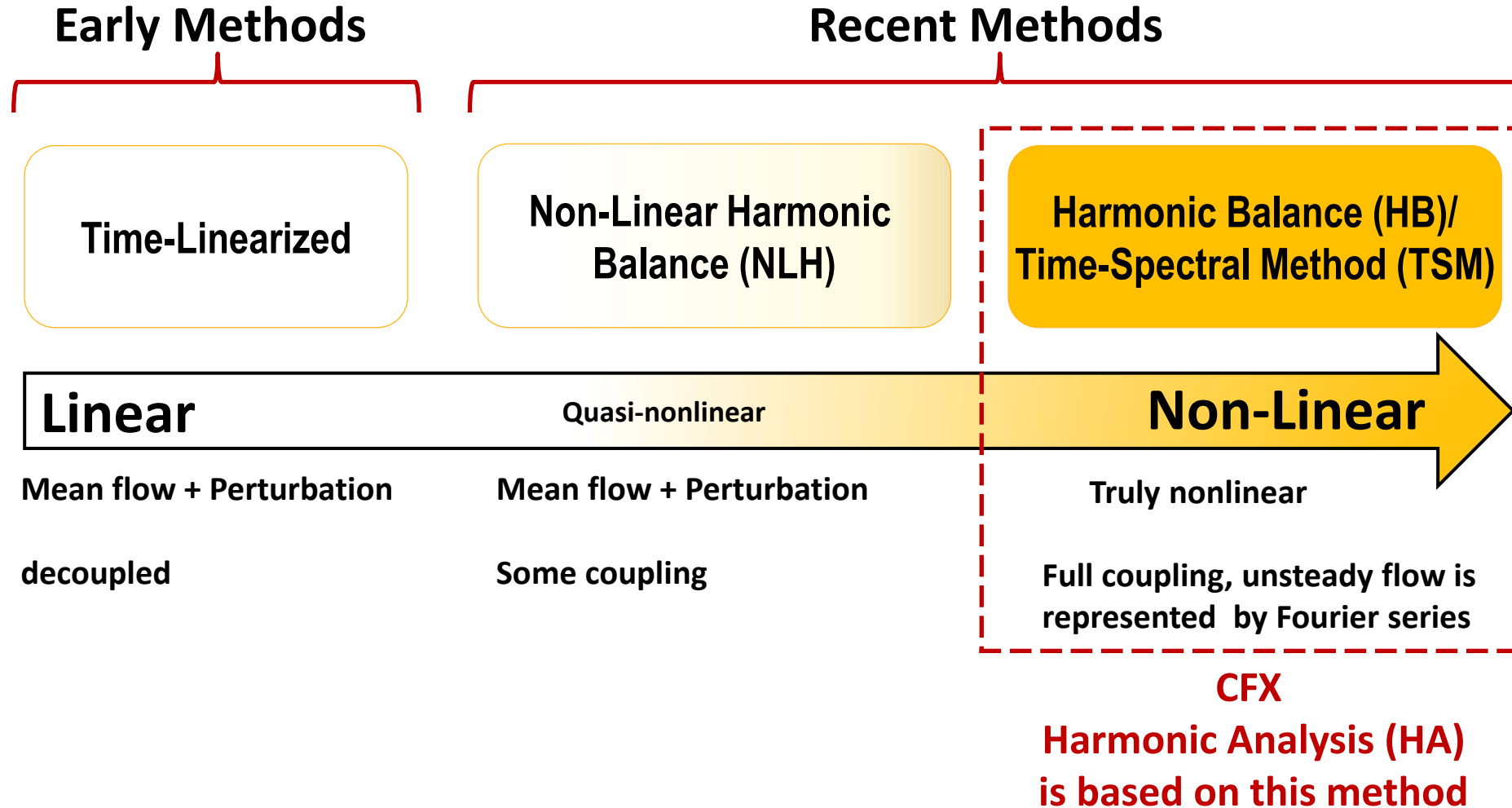


Asymmetric geometries



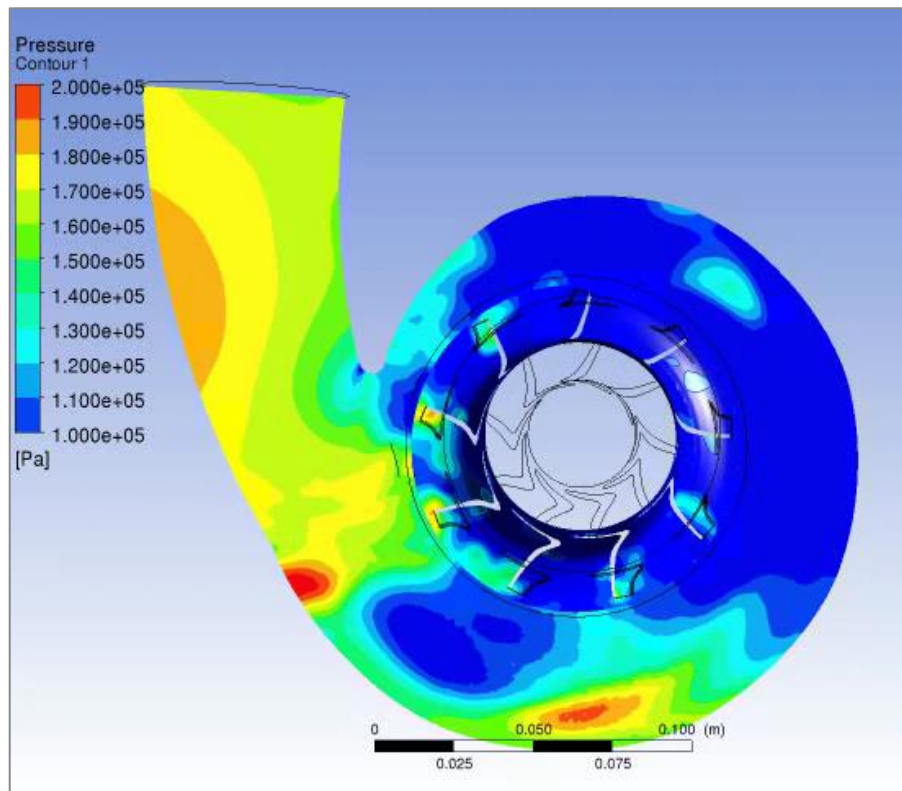
Transient Blade Row Simulations

Frequency Based Methods in CFD Codes

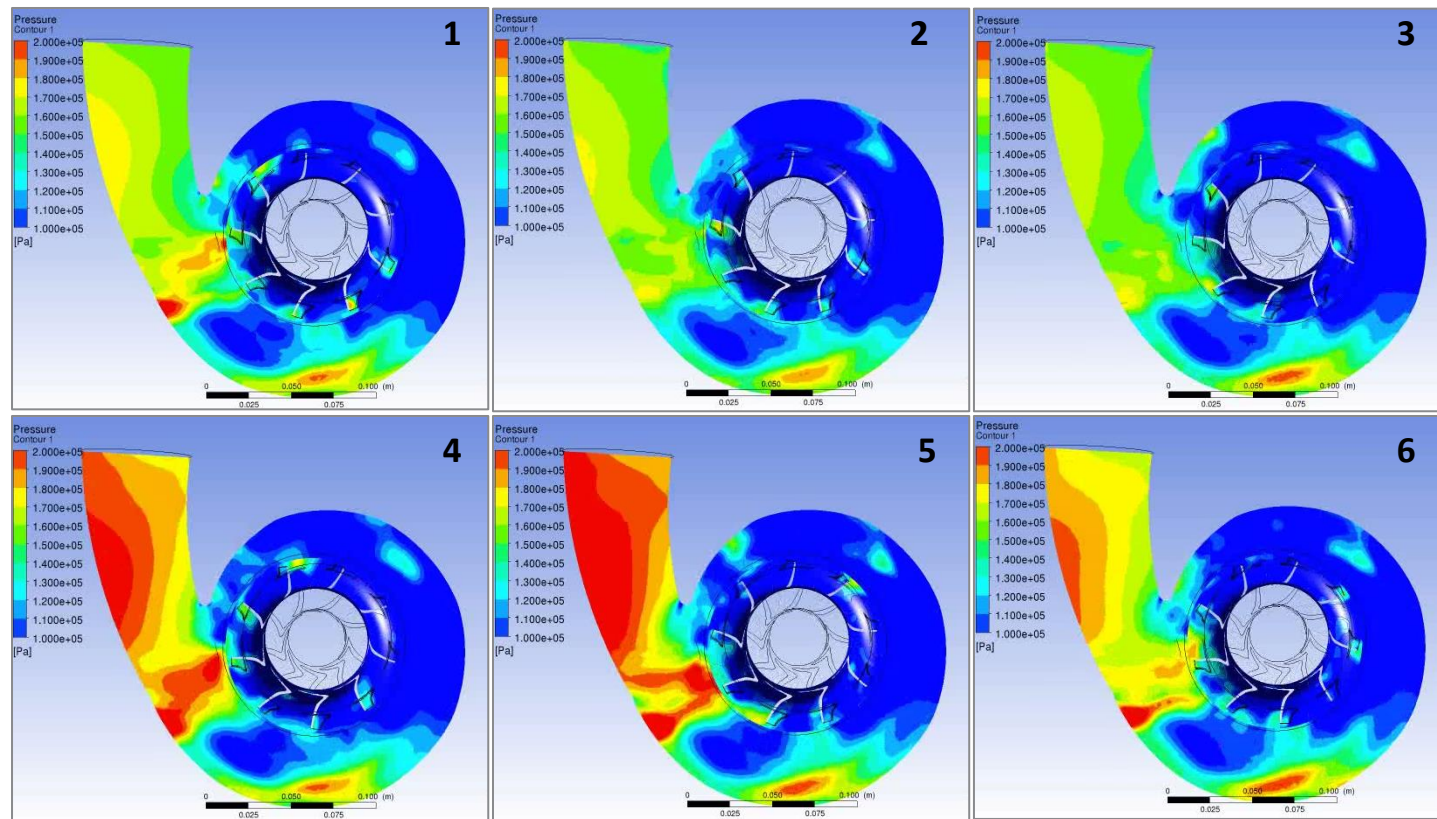


Harmonic Balance accelerates transient analysis

Harmonic Analysis captures transient behavior for turbocharger turbines and 100X faster



Animation of inlet pulse captured with Harmonic Analysis



Evolution of pressure during inlet pulse cycle

Geometry courtesy of PCA Engineering

Aeromechanics

Ansys

/ Aeromechanical Analysis

- Fatigue and fracture of rotating components is a challenge for industry
 - High temperature environment
 - Highly stressed
 - Low damping
 - Many vibration cycles
- Industry trends point towards light weighting
 - More integrally bladed rotors
 - Wear or damage can demand complete replacement
 - Maintenance/downtime costs can be very high
 - Slender blades can be more susceptible to failure
- Advanced aeromechanical simulation is becoming standard practice
 - Demands a standard commercial tool that is easy to use for design engineers



Severe damage to bladed disk
(Zielinski, 2005)

/ Aeromechanical Simulations

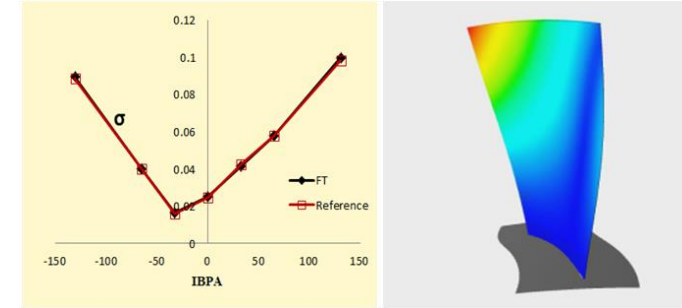
Fluid Structure Interaction (FSI)

CFD



FEM

Expensive



Alternate more practical 1-way FSI models

Blade Flutter Analysis (Aerodynamic Damping)

- Blade oscillation is specified (Modal analysis)
- Compute Aero-damping based on work done on the blade by the fluid (CFD)

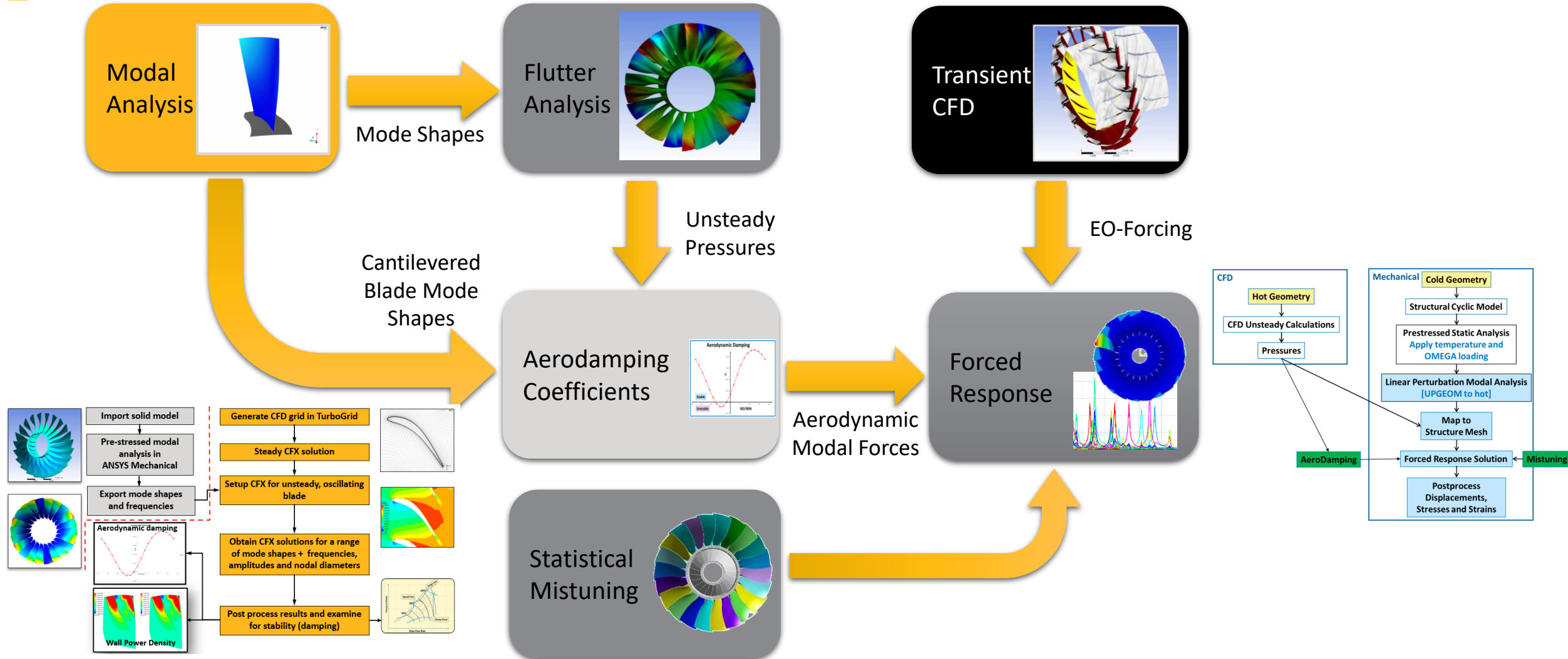
Can predict if blade will or will not flutter

Forced Response Analysis

- Excitation forces obtained from blade row (CFD)
- Compute blade response such as motion and stress level when subjected to these excitations (MSUP Harmonics).

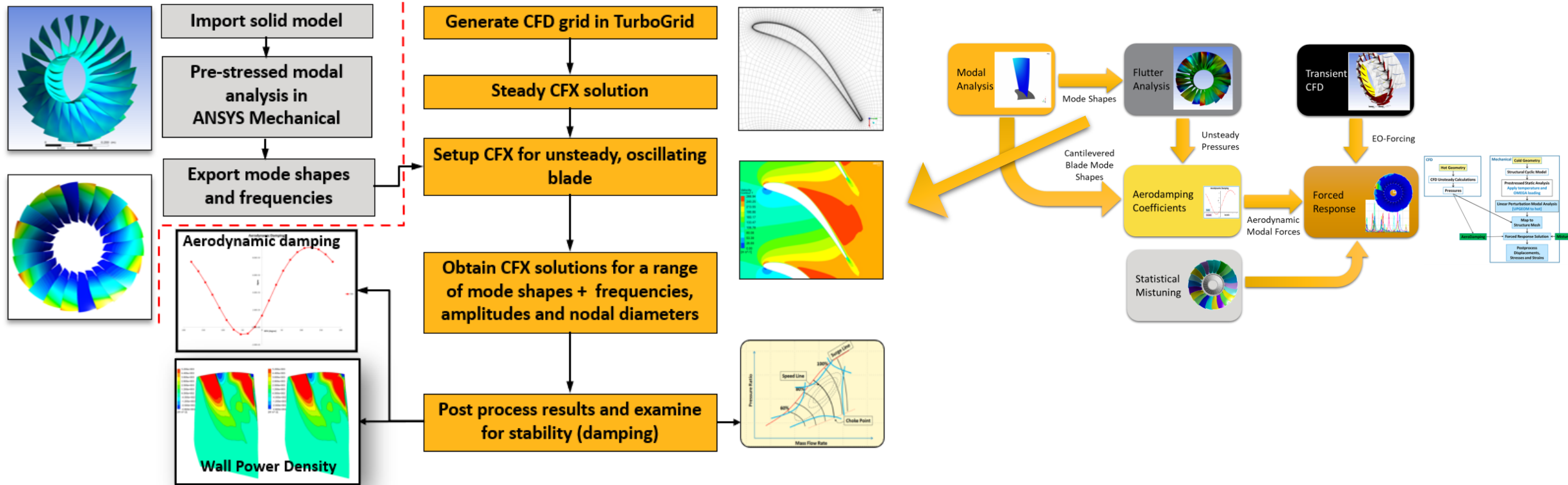
Can predict blade motion & when structure will fail

Aeromechanics Workflow



Seamless, streamlined, validated, Flutter and Forced Response Workflows

/ Aeromechanics Workflow – “Aero” side



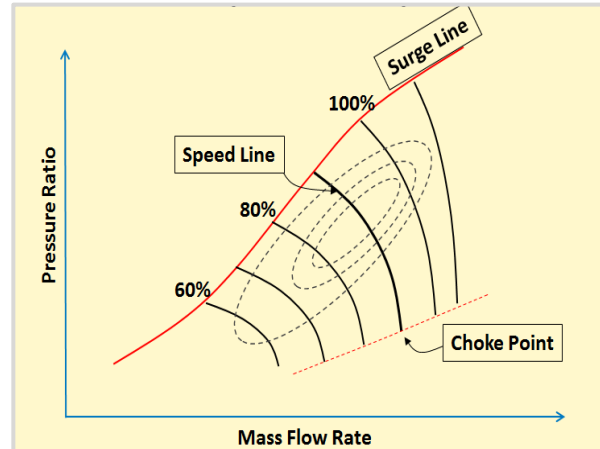
Blade Flutter & Aerodynamic Damping Calculations

Blade Flutter Analysis:

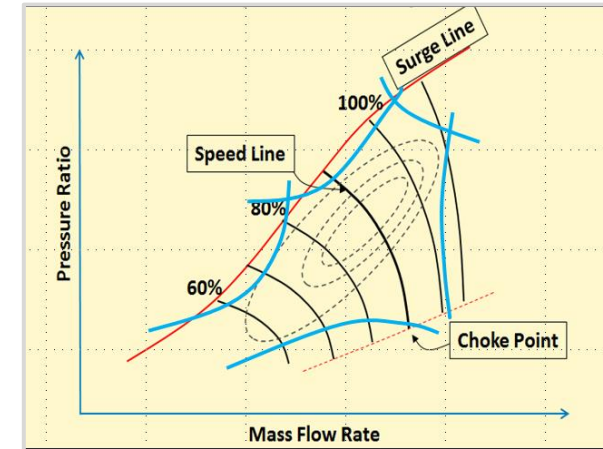
Determine if the aero loads will damp out blade vibration at natural frequencies

Flutter Margin

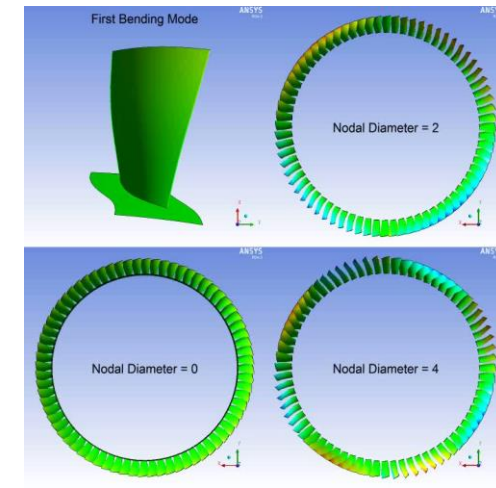
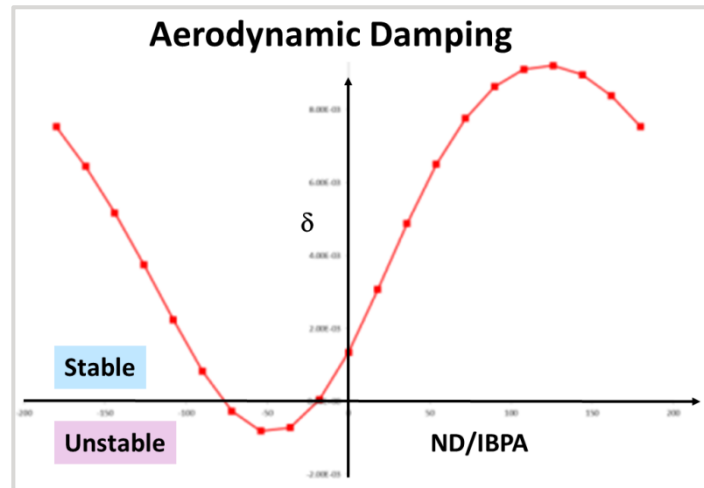
Aerodamping in Forced Response analysis



Performance map based on aerodynamic analysis



Performance map based on Aeromechanical analysis

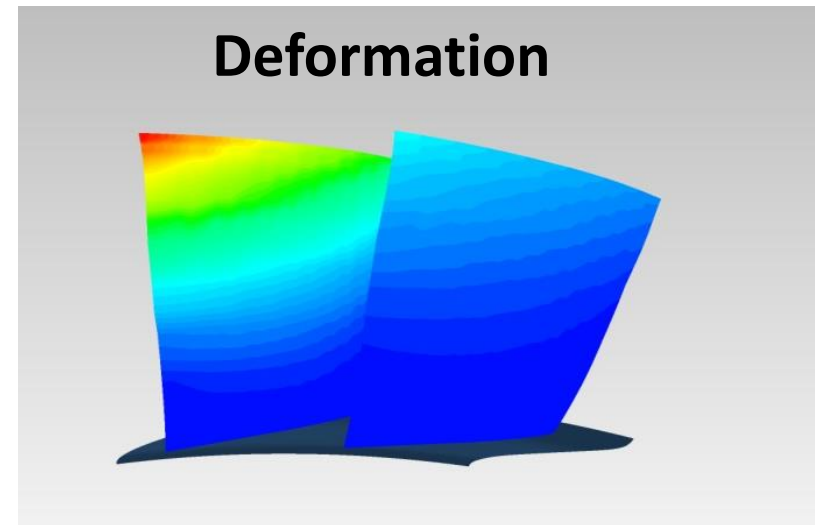
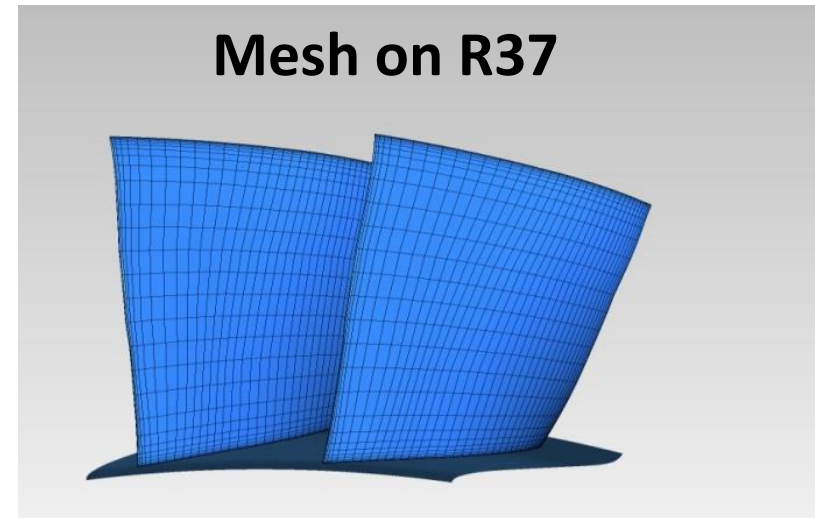


/ Harmonic Analysis (FT Pitch-Change) Blade Flutter/ Aero-damping

- Rotor 37 axial compressor
- 36 rotor blades
- Rotation= 1800 rad/s
- Blade vibration:
 - From ANSYS Mechanical Modal analysis
 - 1st Bending, frequency=1152.13 Hz
 - Max displacements = 0.00129 m

Test objective is to compare HA-FT to Transient-FT solution

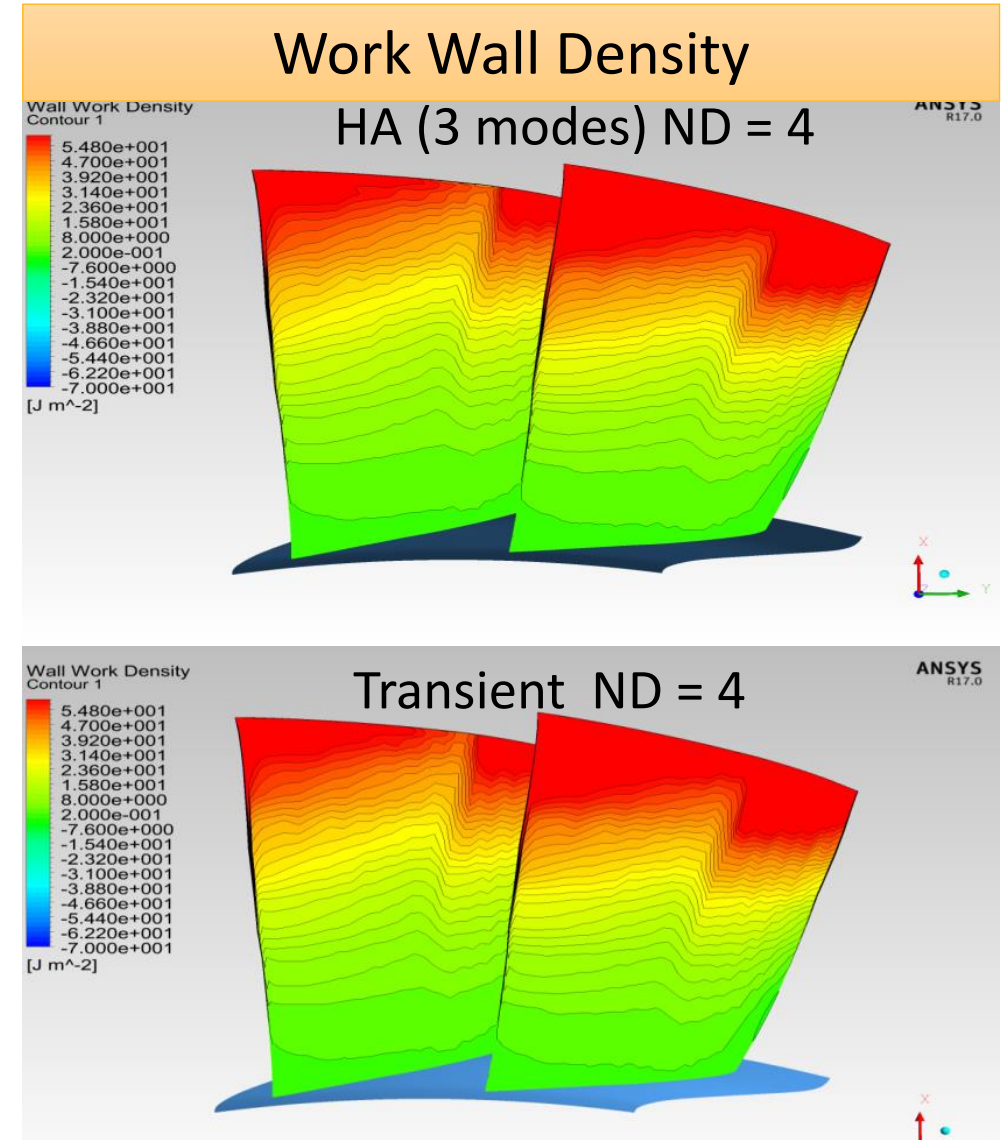
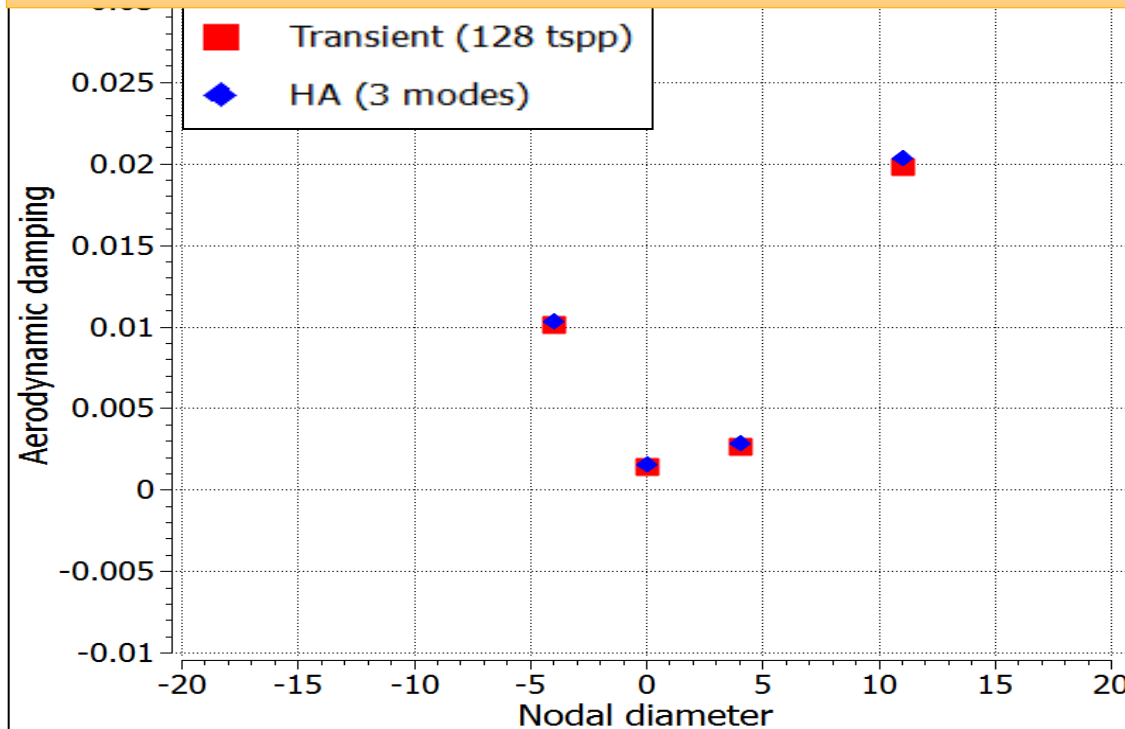
- Aero-damping values
- Computation efficiency



Harmonic Analysis (FT Pitch-Change) Blade Flutter/ Aero-damping

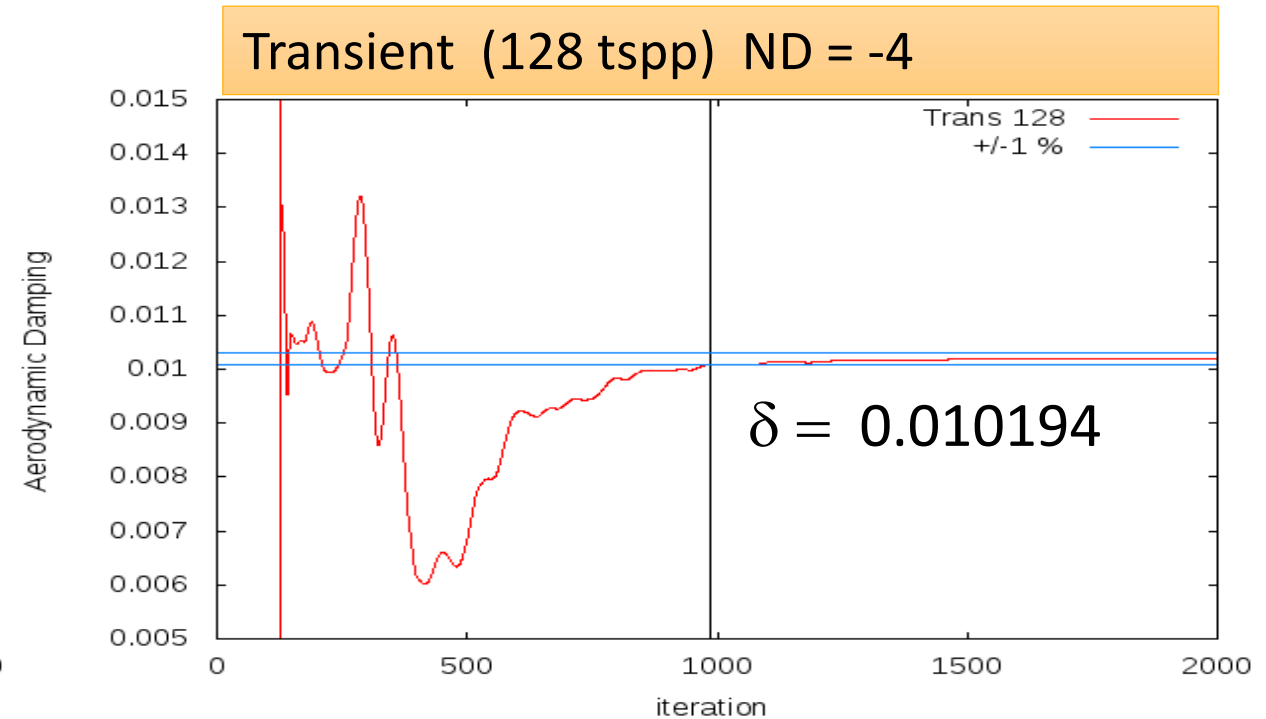
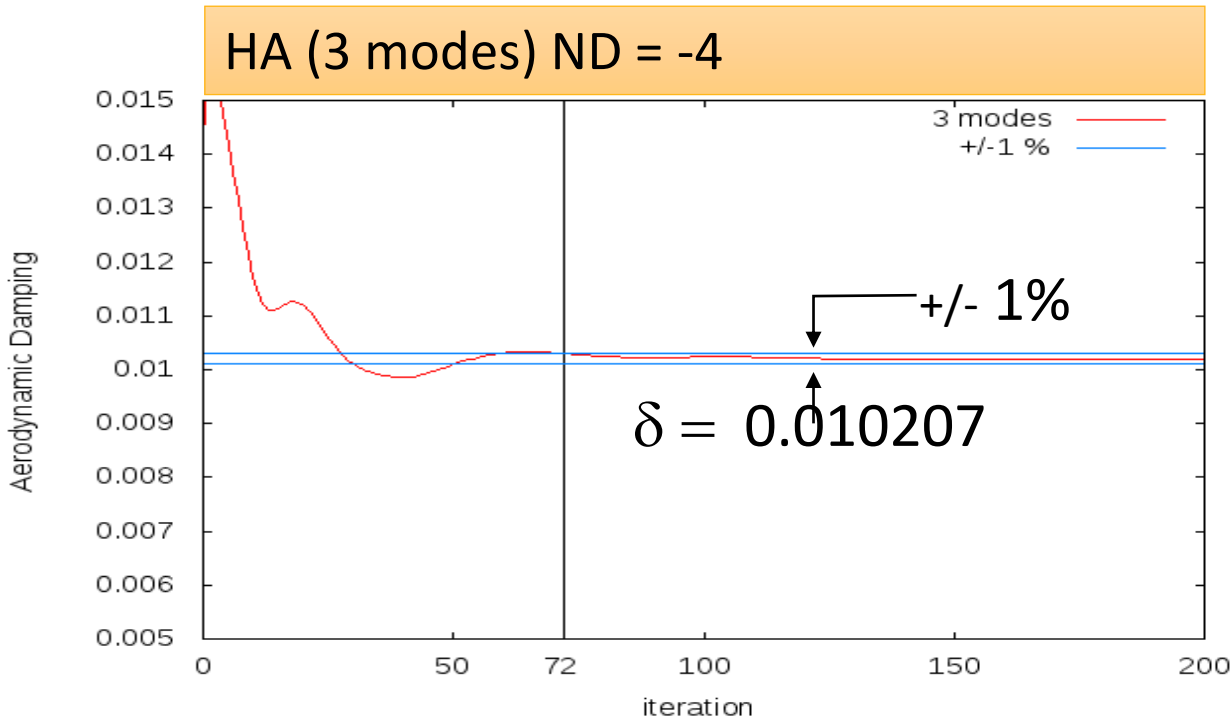
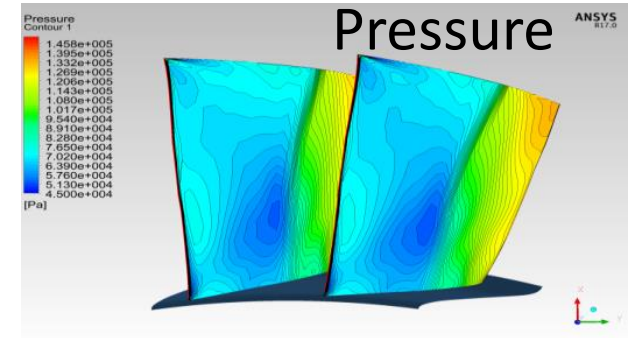
Aero-damping comparison

Aero-damping HA-FT vs Transient -FT



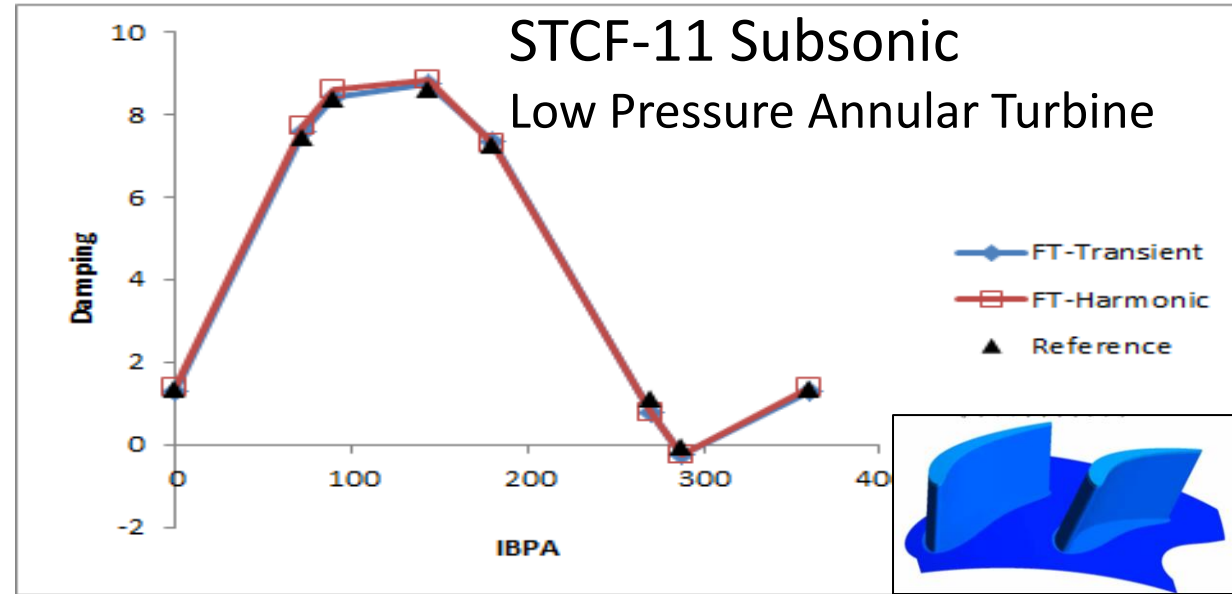
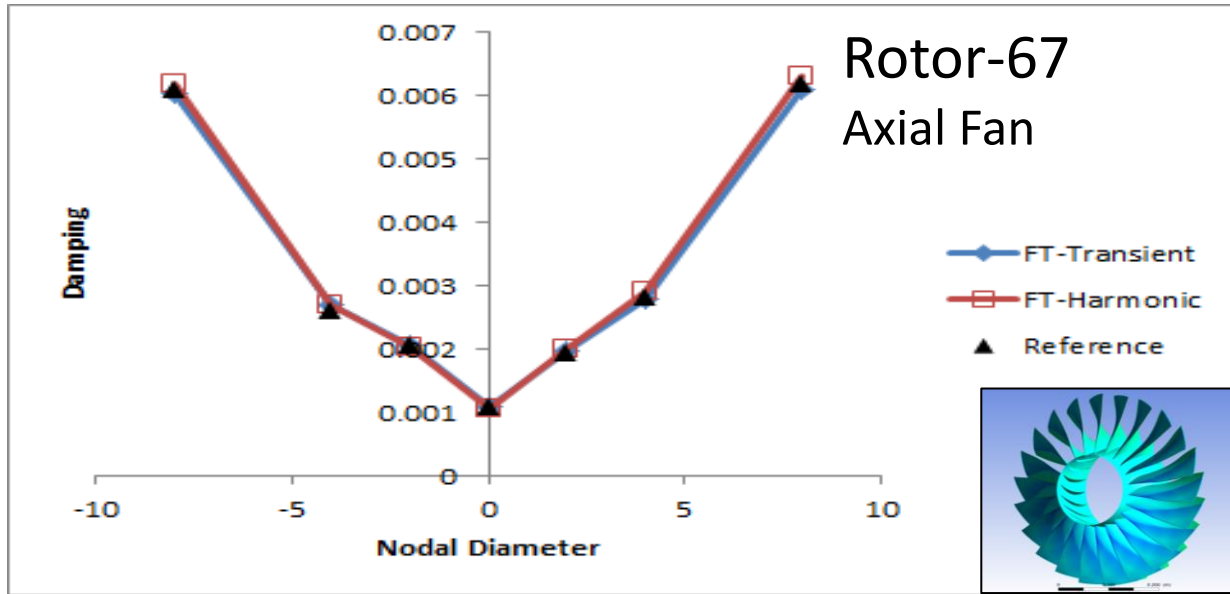
Harmonic Analysis (FT Pitch-Change) Blade Flutter/ Aero-damping

Convergence comparison



FT-HA (3 modes) **16X faster** than FT-Transient

Harmonic Analysis (FT Pitch-Change) Blade Flutter/ Aero-damping

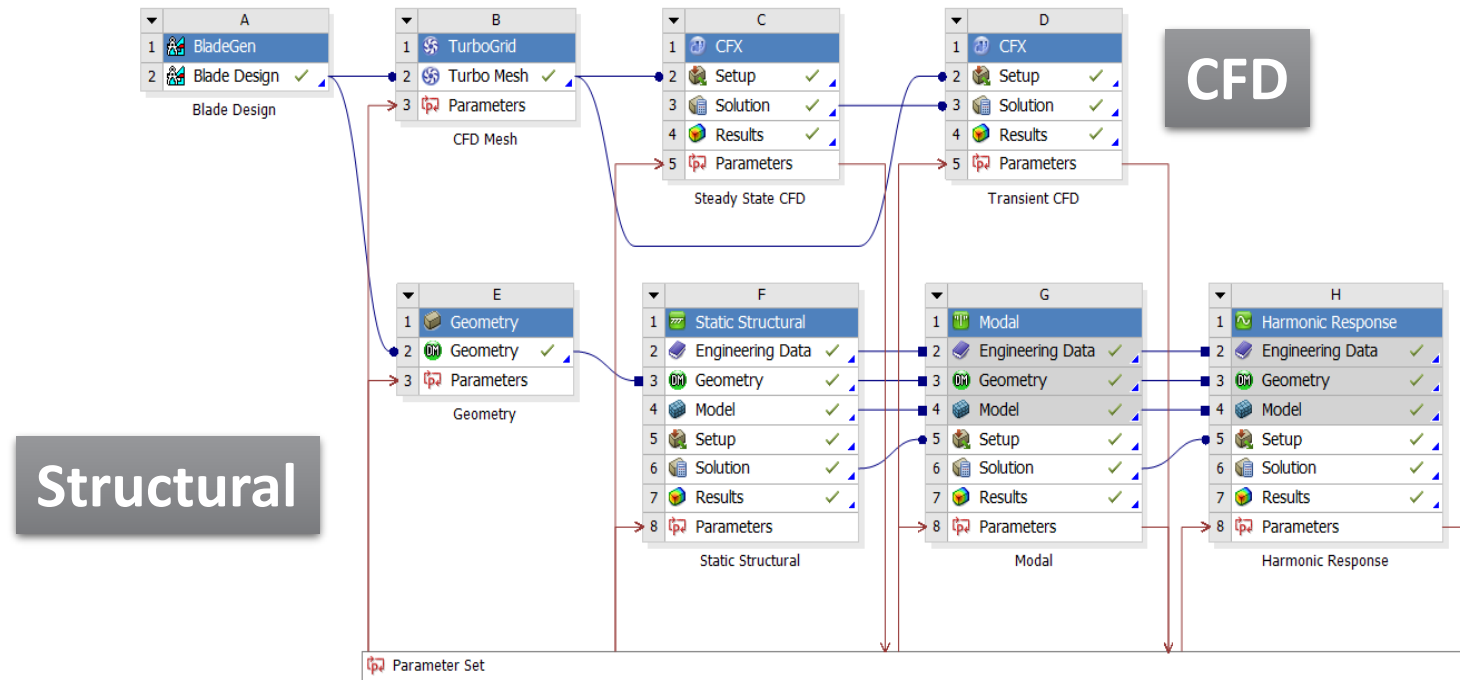
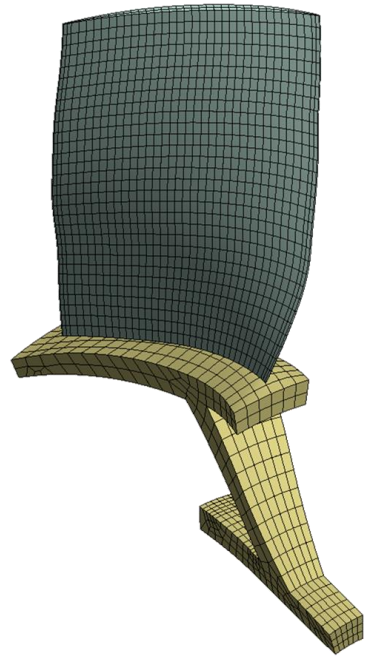


- All Calculations done with 1 mode , 15 pseudo-time-step per oscillation cycle
- starting from steady-state solution
- FT-HA (1 mode) about 20x to 30x Faster than FT-Transient (based +100 tspp)
- FT-Transient about 5x to 7x faster than Full-wheel solution

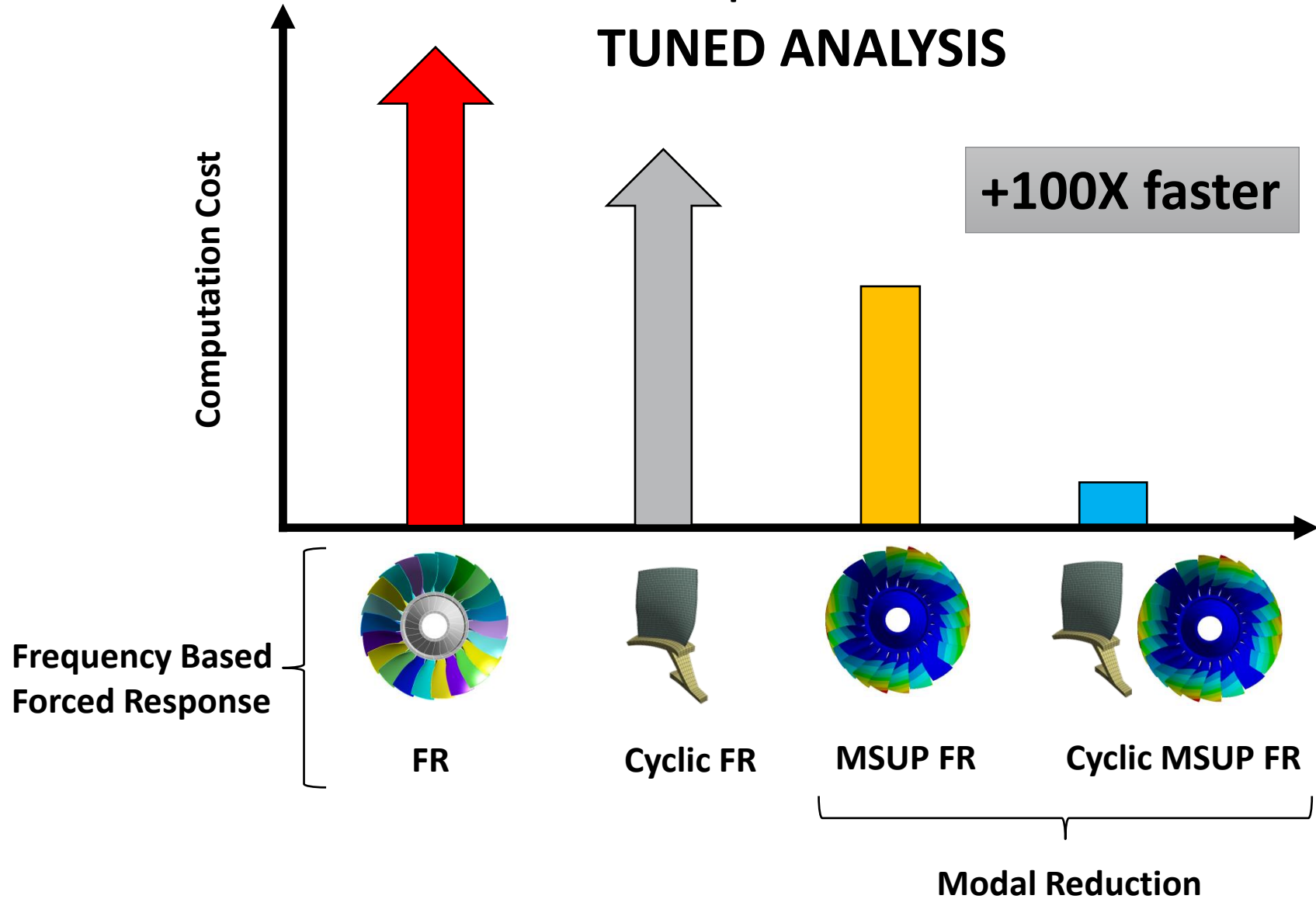
10-to-100X faster

Forced Response Analysis - Workflow

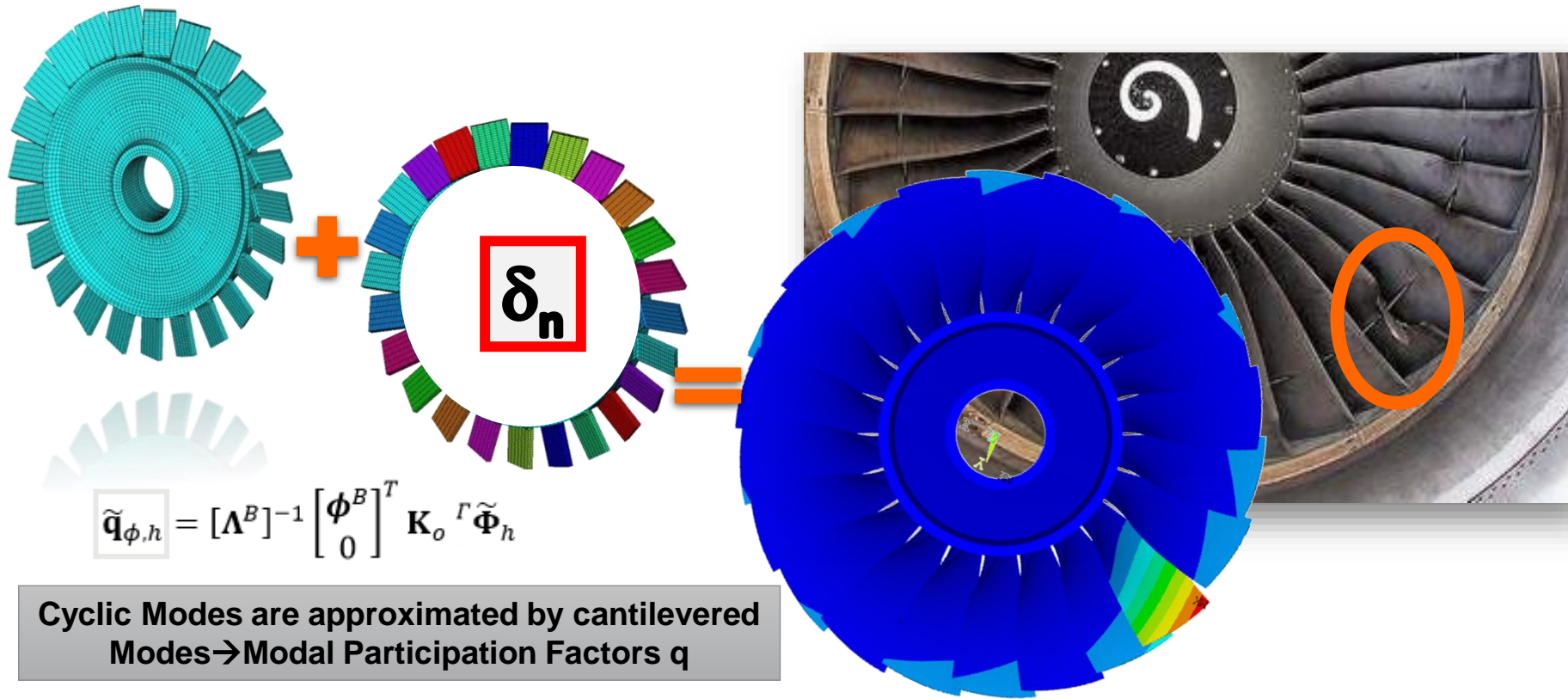
- Small industrial sized example
 - Number of sectors: 22
 - Number of elements: 3141 per sector
 - Number of nodes: 18133 per sector



Fast and Efficient Forced Response Method



/ Aeromechanics – Mistuning



$$\left\{ \underbrace{-\Omega^2 \mathbf{I}}_{\text{Excitation frequency}} + \underbrace{i\Omega \left[\left(\beta + \frac{1}{\Omega} g \right) \mathbf{\Lambda}^s + \alpha \mathbf{I} \right]}_{\text{Reduced mass}}, \underbrace{\mathbf{\Lambda}^s}_{\text{Reduced damping}}, \underbrace{\sum_n \mathbf{q}_n^T \delta_n \mathbf{K}_c \mathbf{q}_n}_{\text{Mistuning terms}}, \underbrace{\mathbf{K}_a}_{\text{Aero stiffness}} \right\} \mathbf{a} e^{i\omega t} = \left\{ \sum_n \underbrace{F_{n,EO}^T}_{\text{Projection to modal space and expansion from cyclic domain}} \underbrace{\tilde{\Phi}_h^T \mathbf{F} e^{i\phi(n-1)}}_{\text{Single sector engine order forcing}} \right\} e^{i\omega t}$$

 **Ansys**

